

# A systems approach to fluid dynamics: input-output models

UNH/IAM Workshop  
November 2015

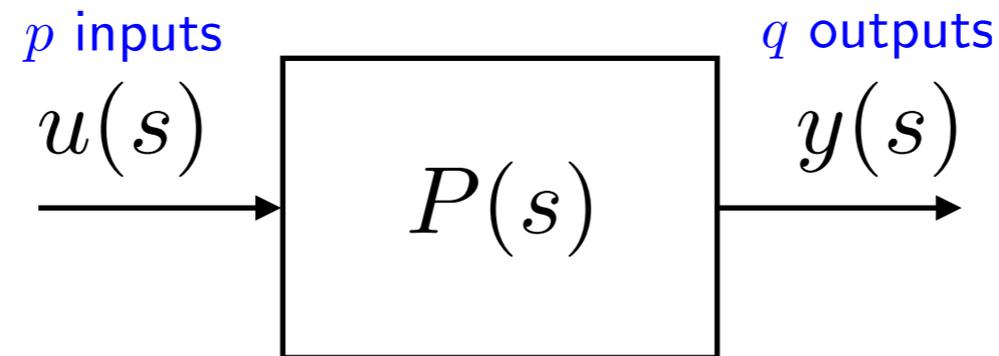
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University of Melbourne



overview

# terminology

- frequency domain



$$y(s) = P(s)u(s)$$

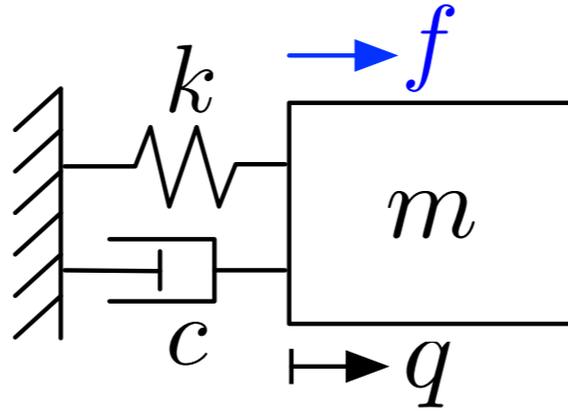
$q \times 1$        $q \times p$      $p \times 1$

- time domain: we will sometimes need the state-space description:

$$\dot{x}(t) = A x(t) + B u(t)$$

$$y(t) = C x(t) + D u(t)$$

# frequency response: SISO case



$$m\ddot{q}(t) + c\dot{q}(t) + kq(t) = f(t)$$

$$P(s) = \frac{Q(s)}{F(s)} = \frac{1}{ms^2 + cs + k}$$

$$P(j\omega) = \frac{1}{(k - m\omega^2) + jc\omega}$$

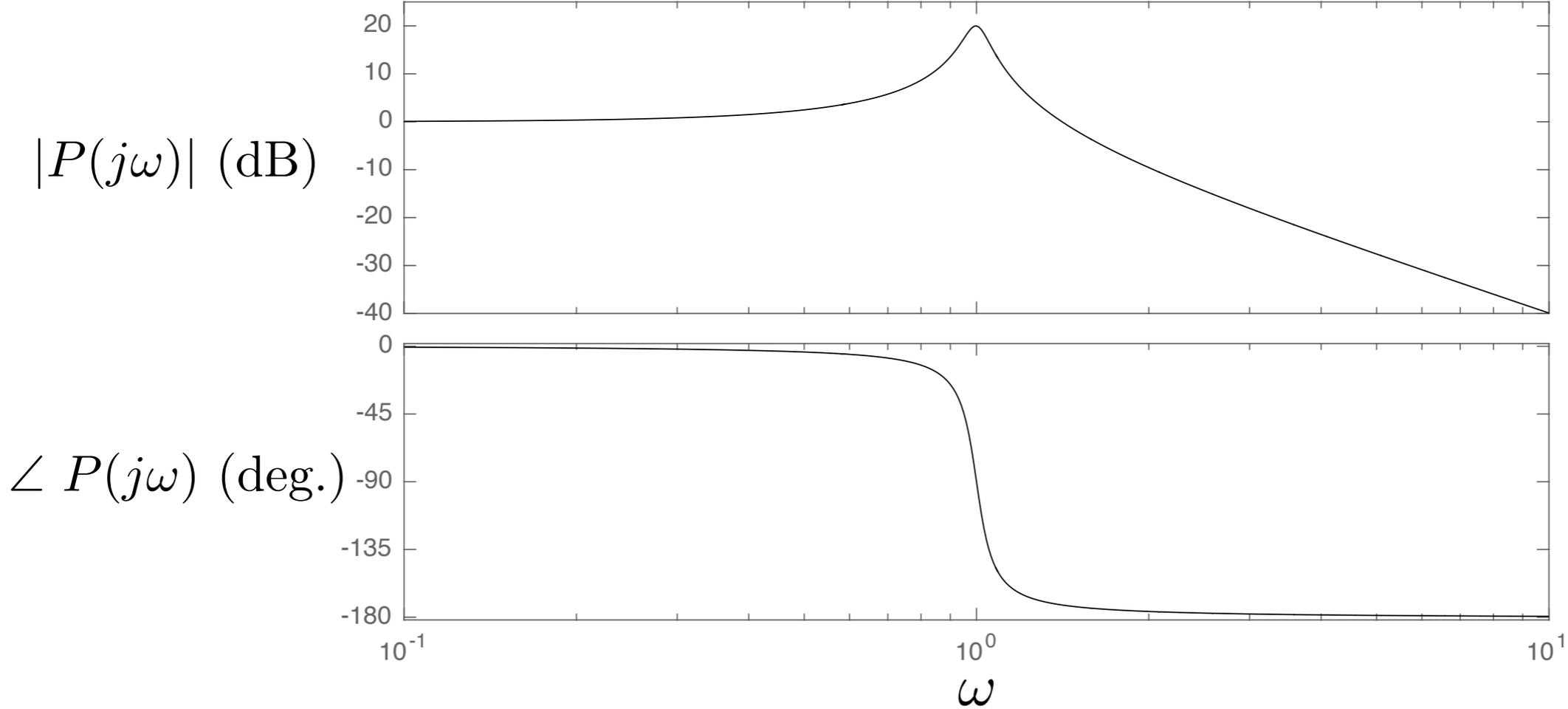
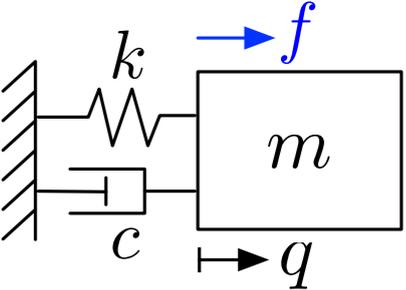


$\mathcal{L}$



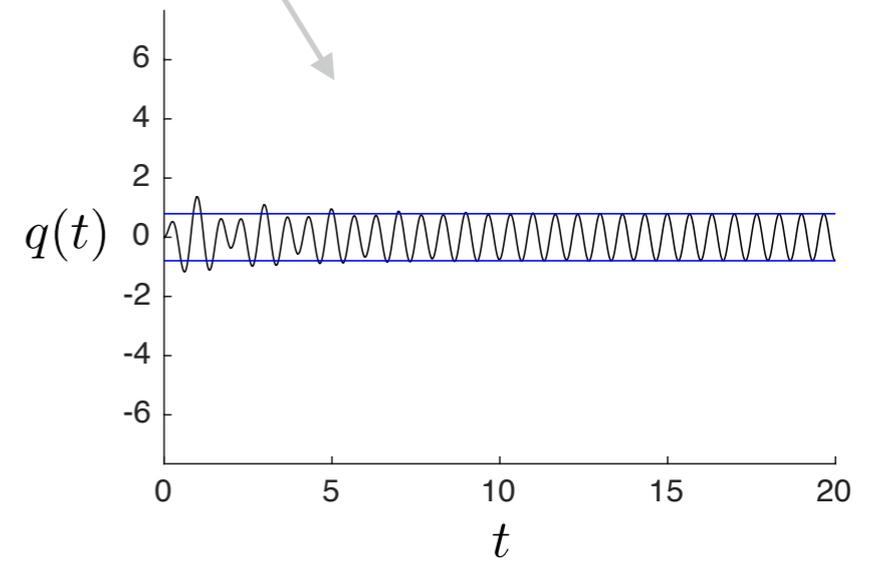
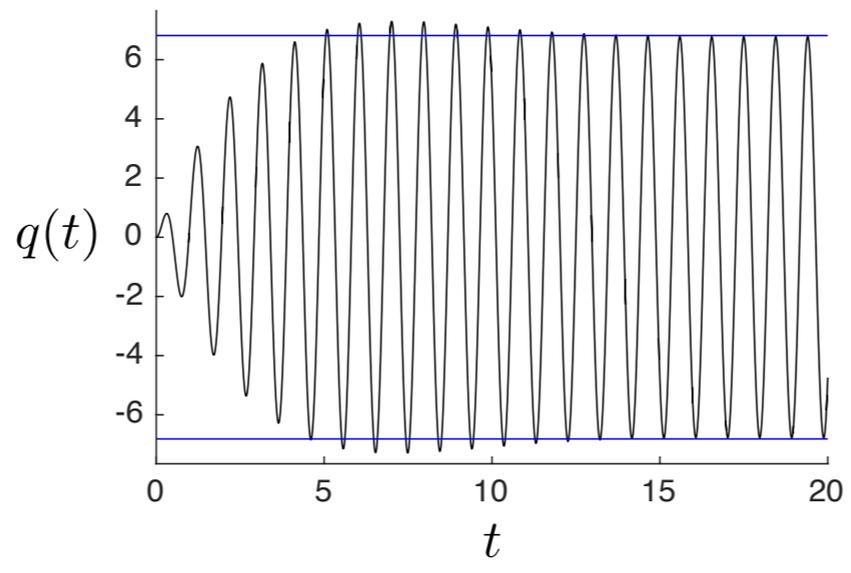
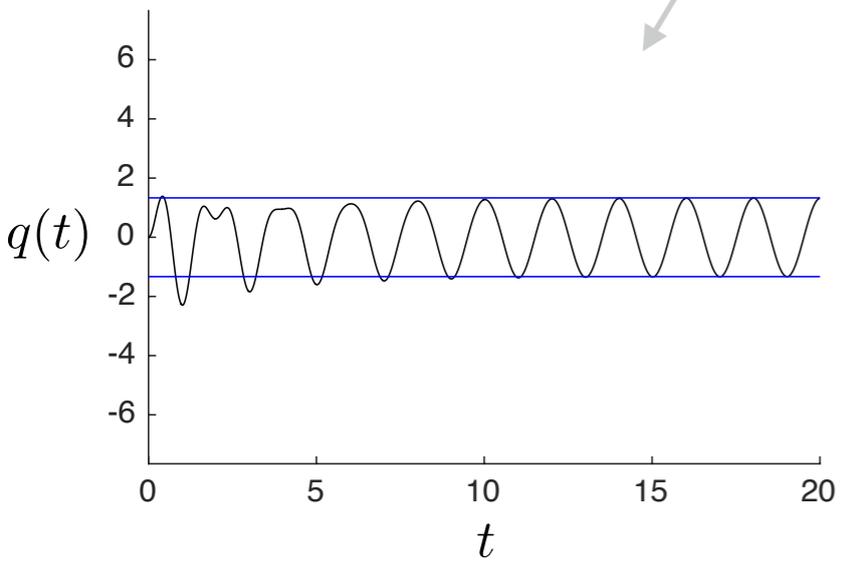
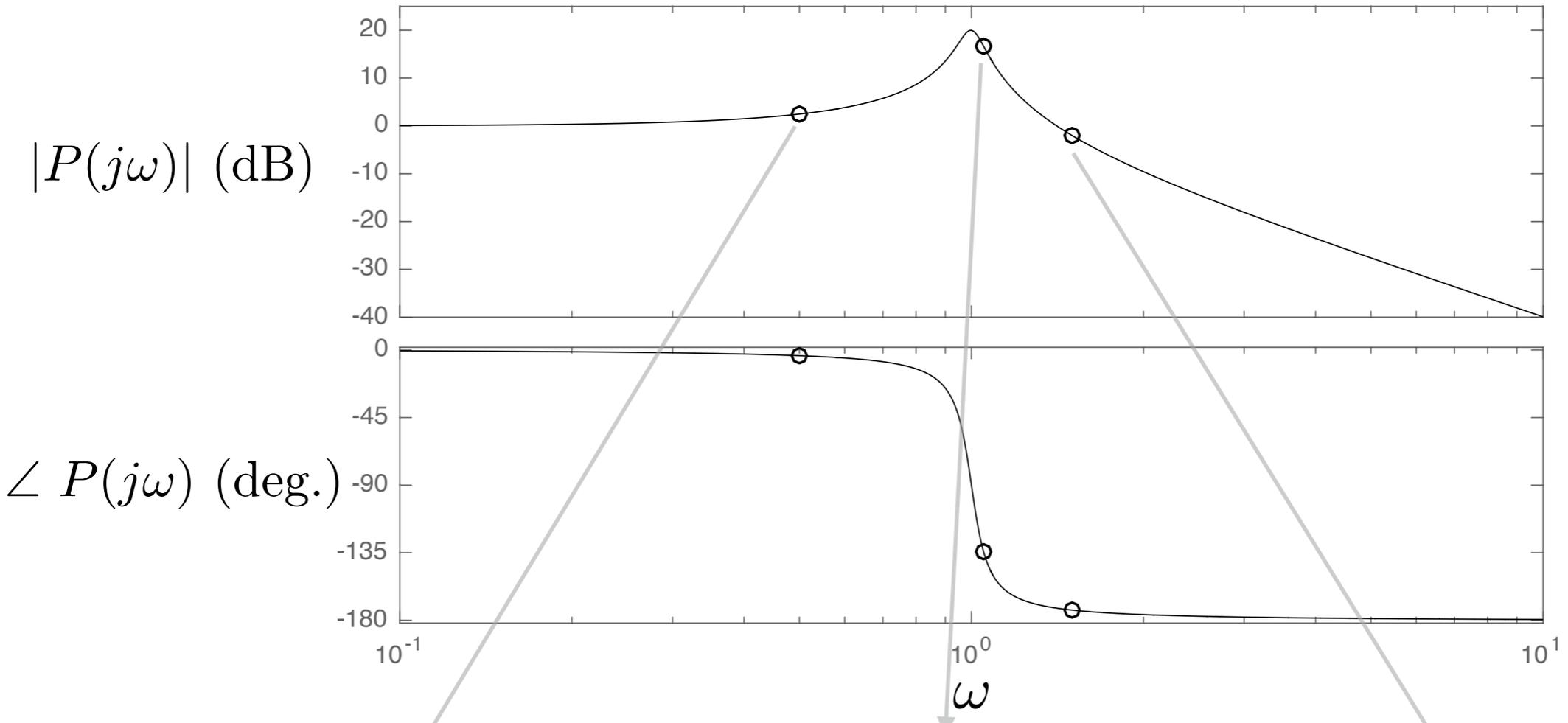
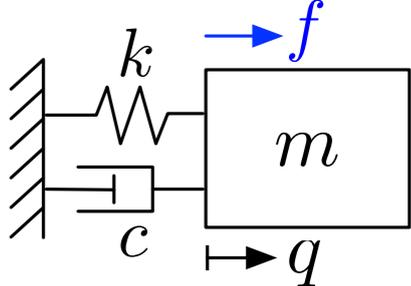
$s = j\omega$

# frequency response: SISO case

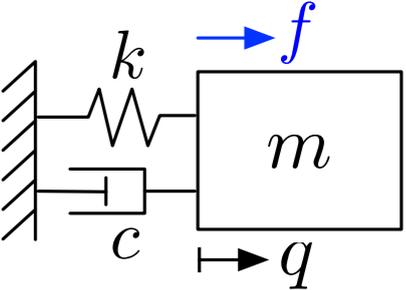


$$P(j\omega) = \frac{1}{(k - m\omega^2) + jc\omega}$$

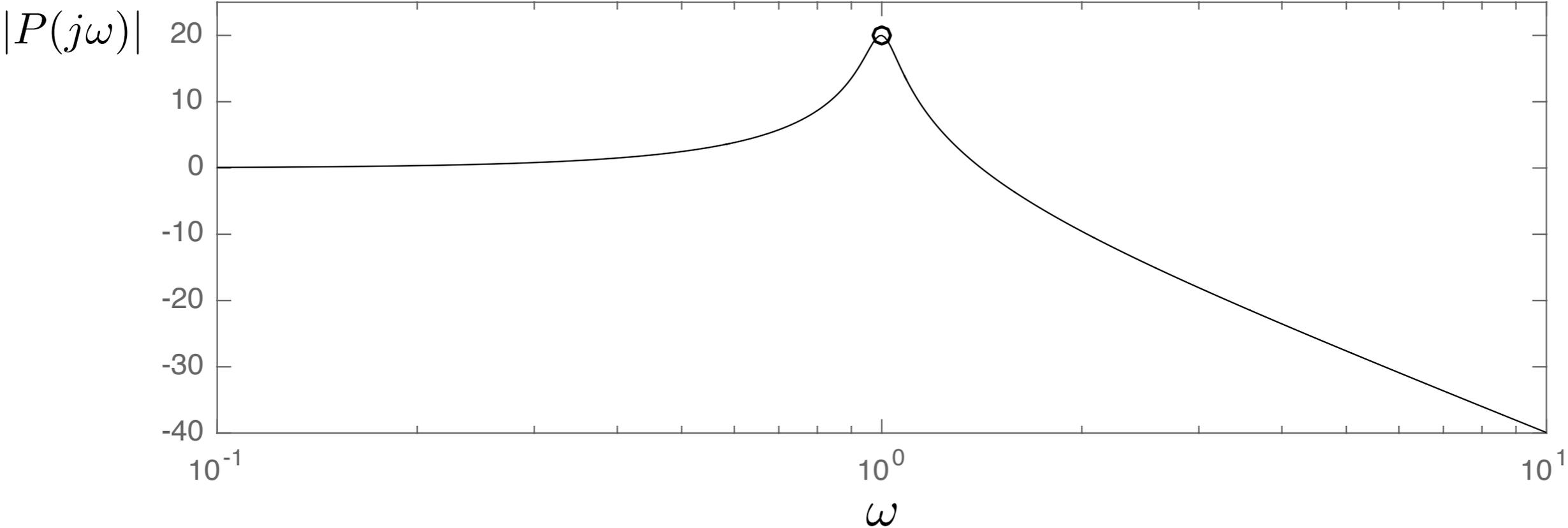
# frequency response: SISO case



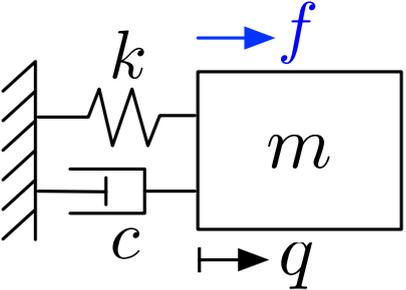
the  $\infty$ -norm: SISO case



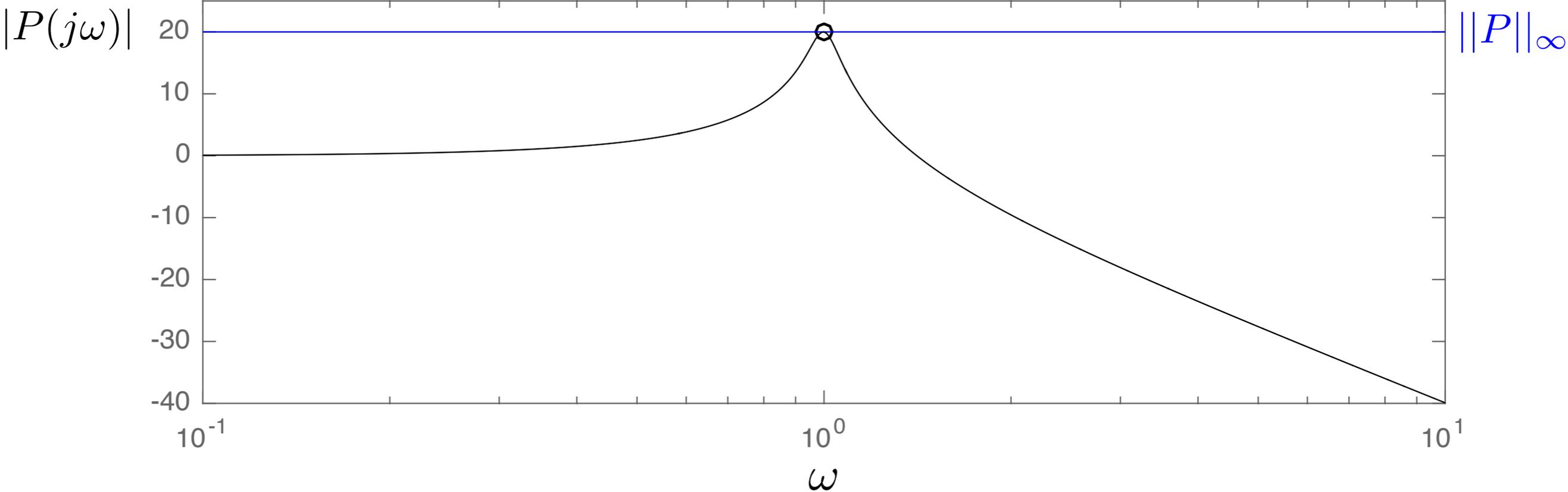
$$\|P\|_{\infty} = \max_{\omega} |P(j\omega)|$$



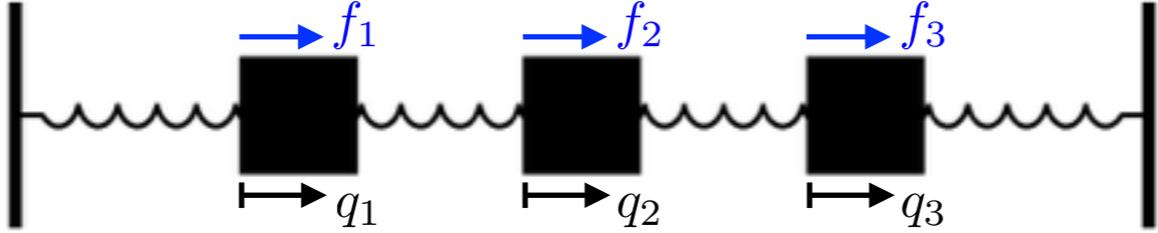
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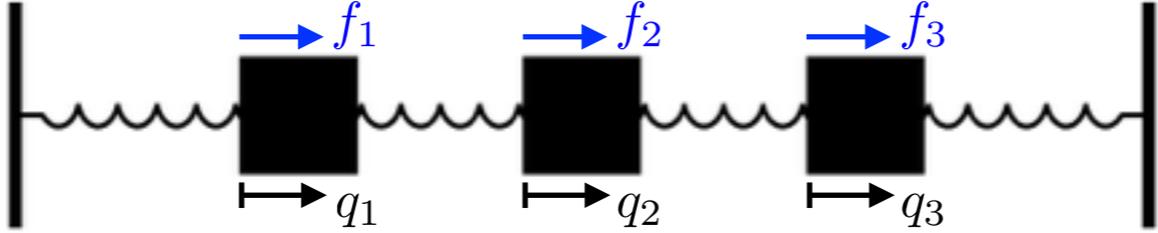
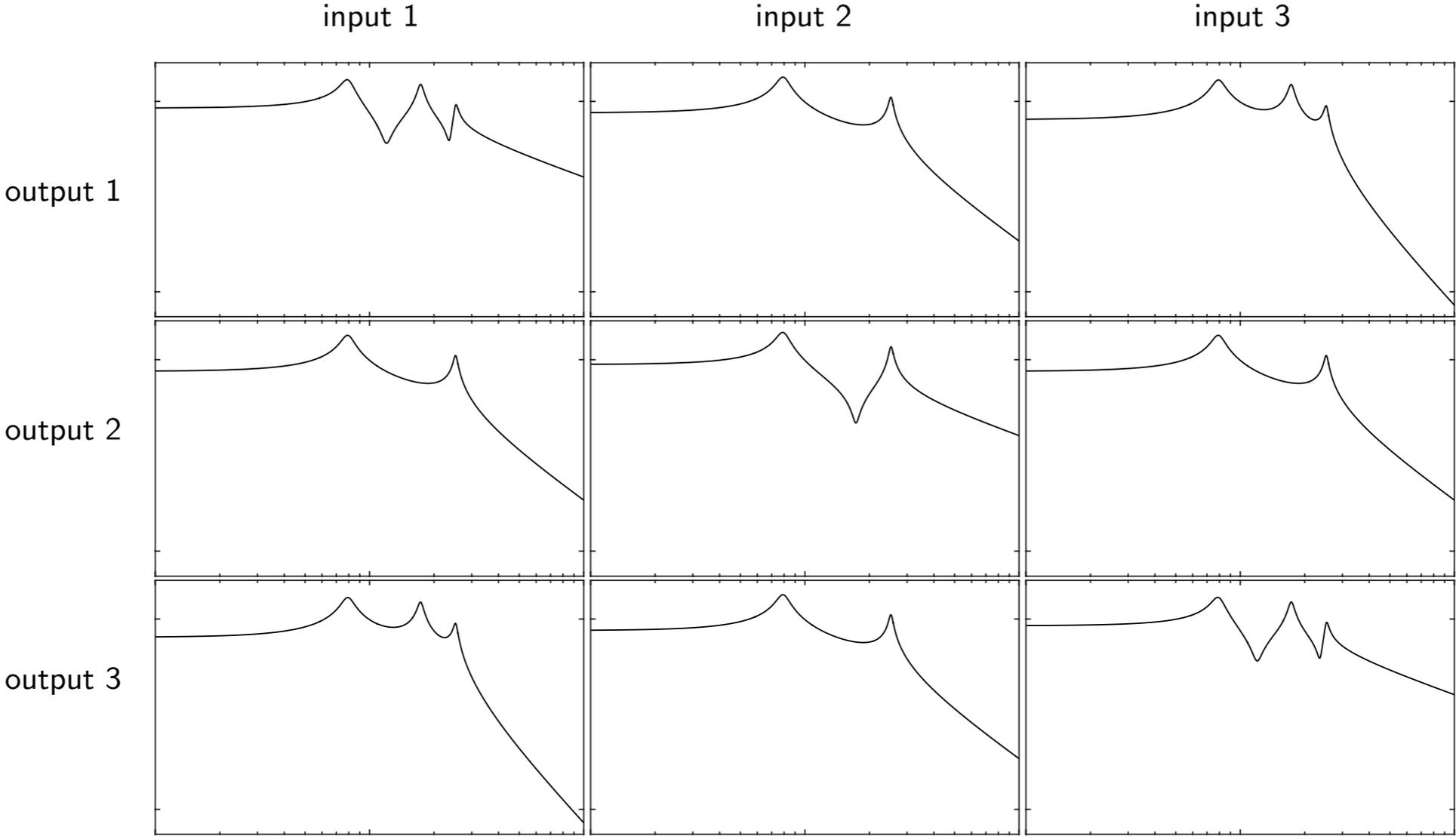
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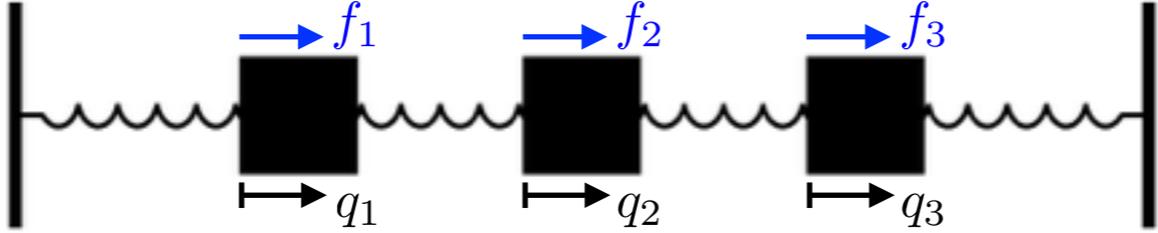
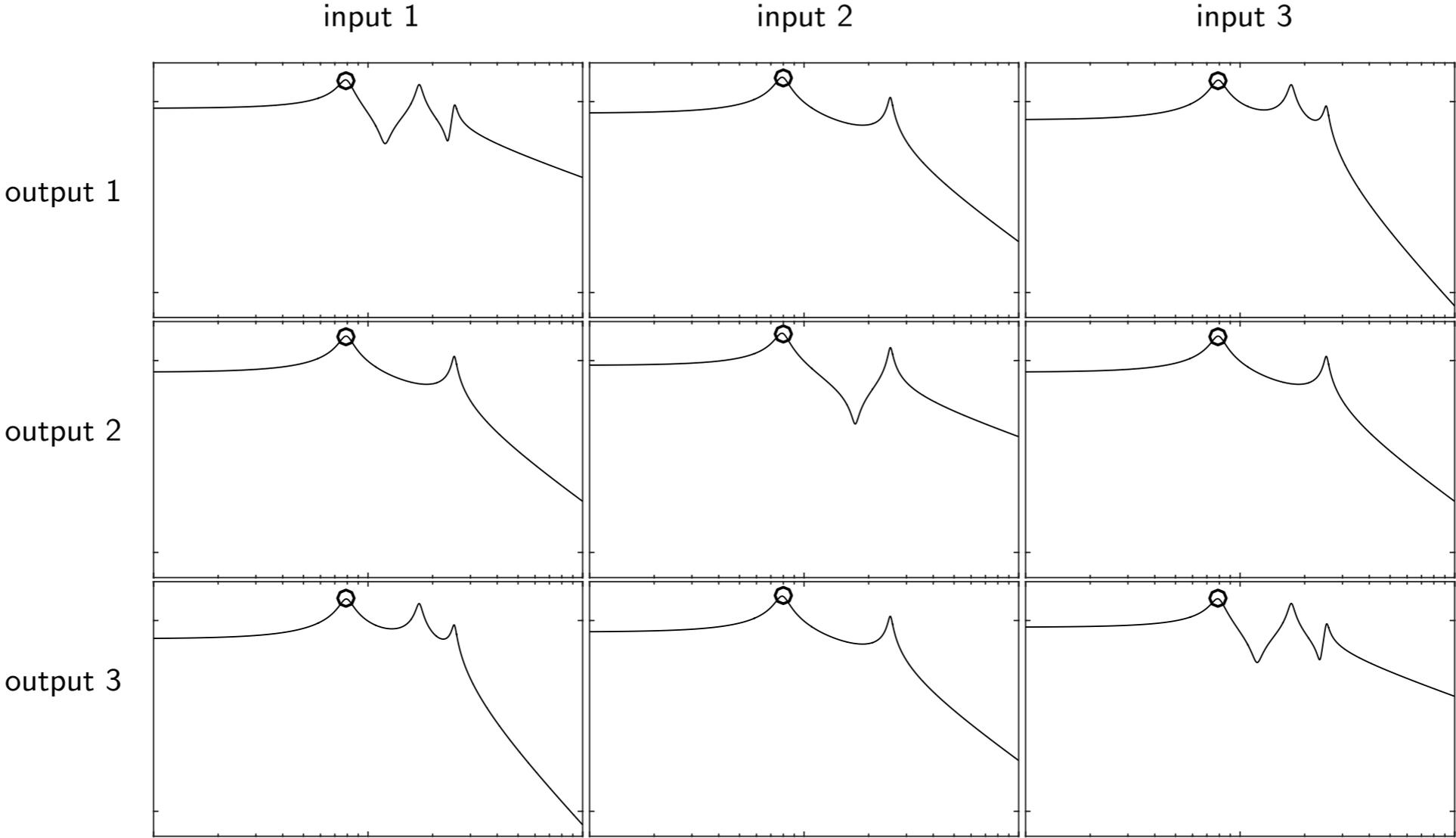
# frequency response: MIMO case



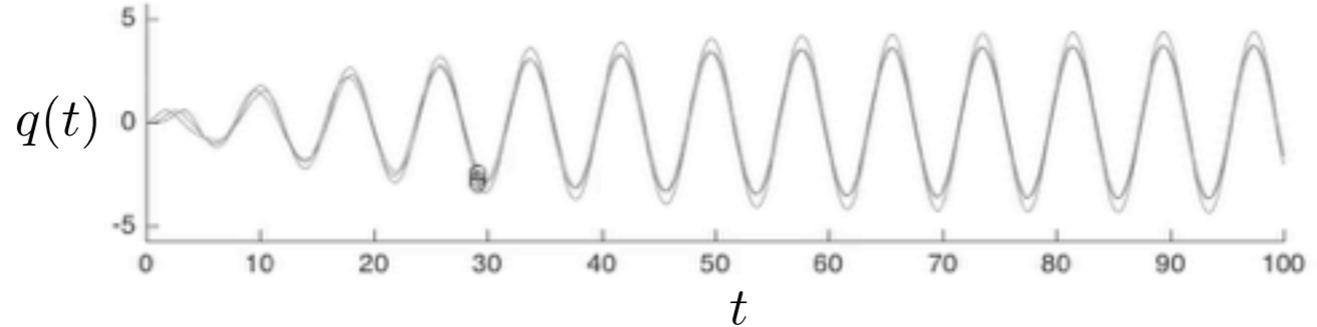
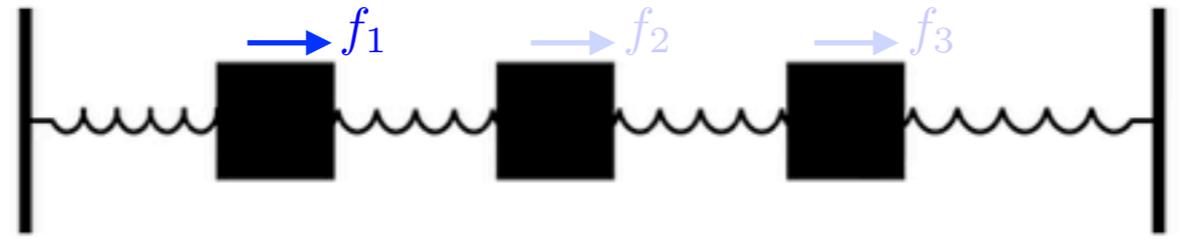
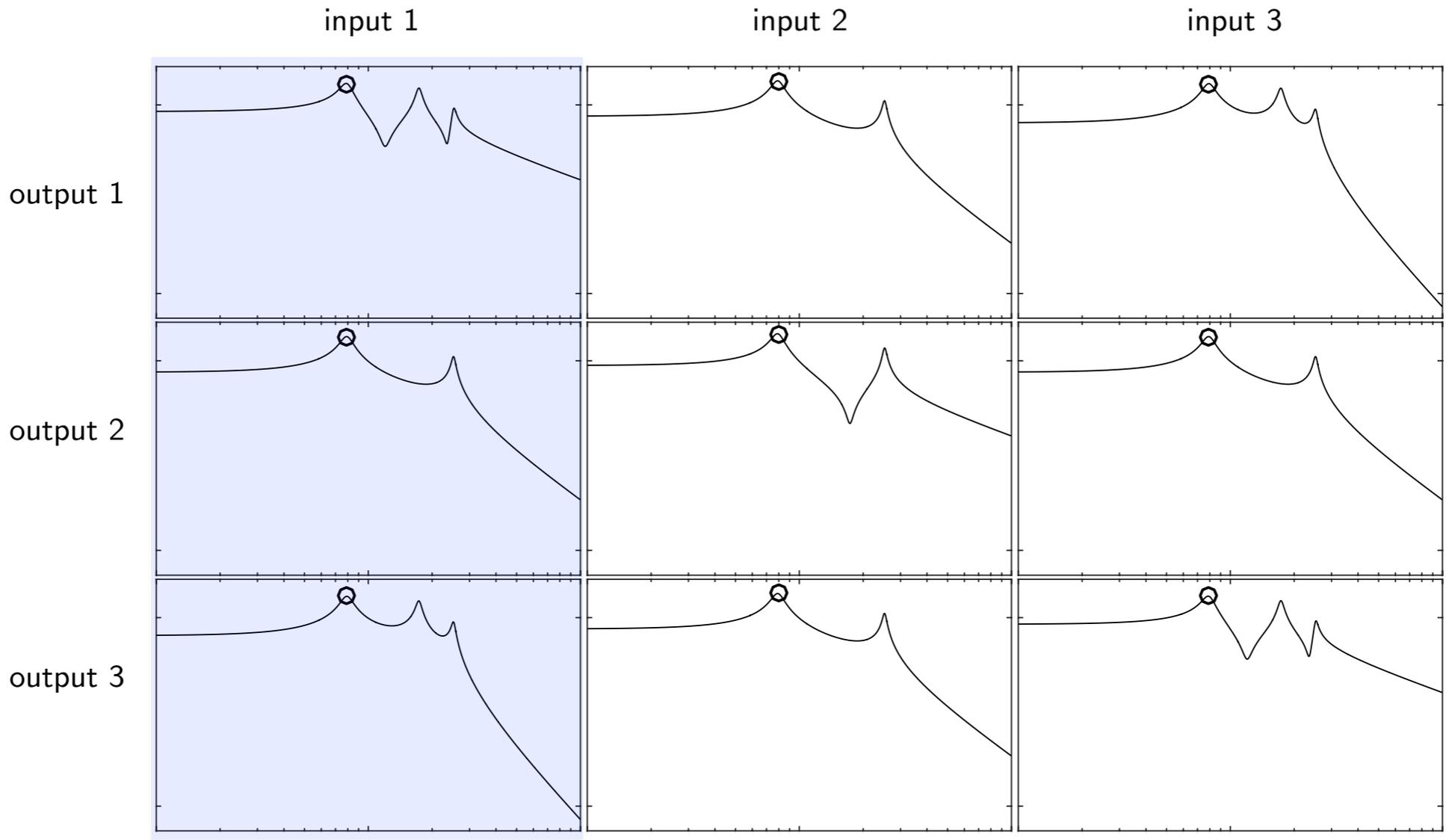
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# frequency response: MIMO case

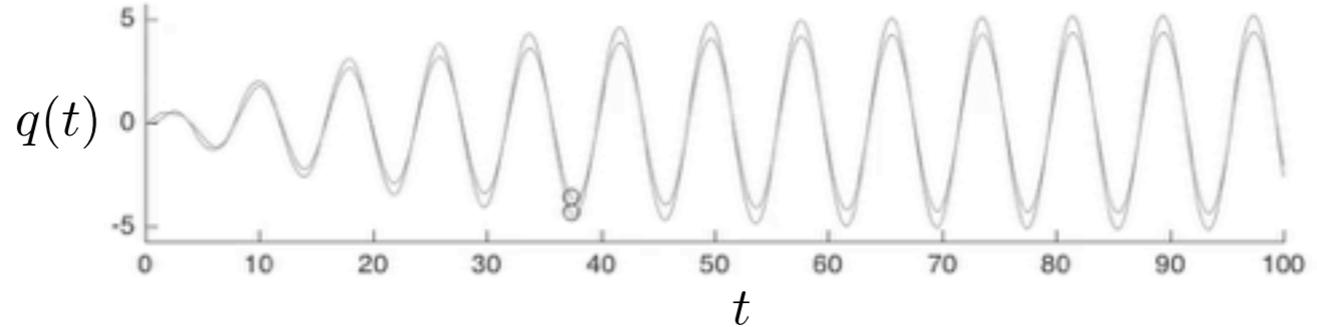
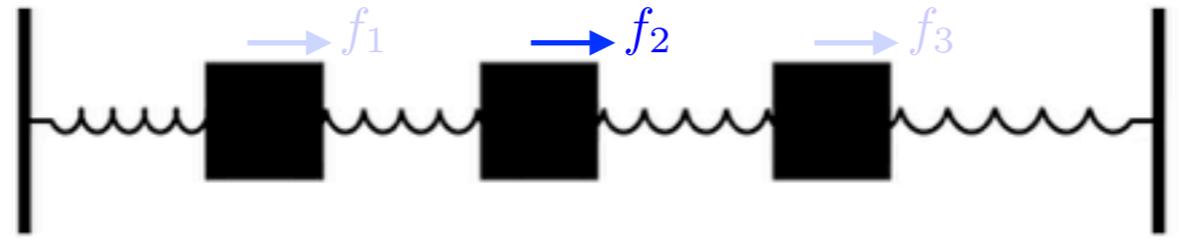
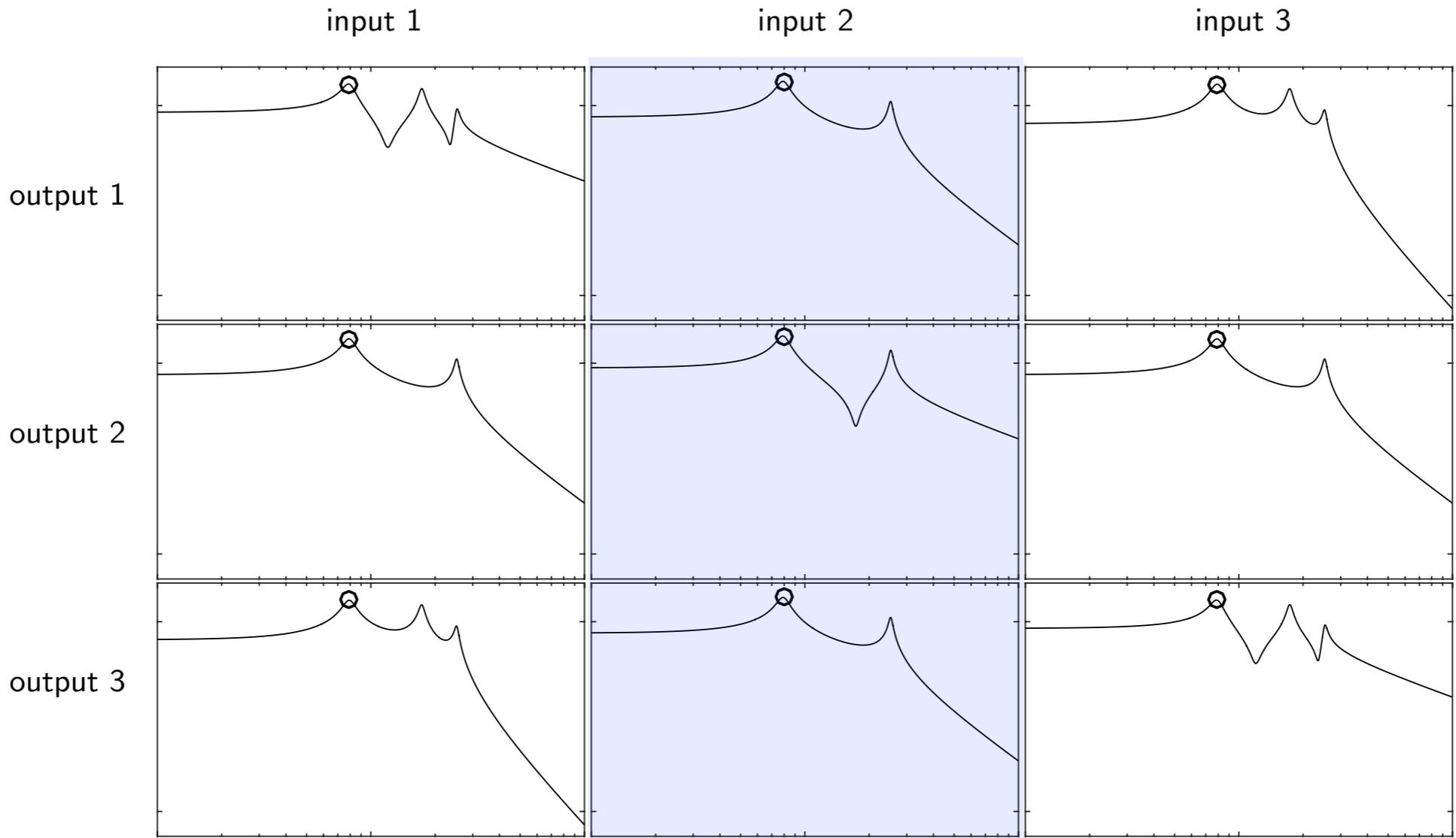


# frequency response: MIMO case



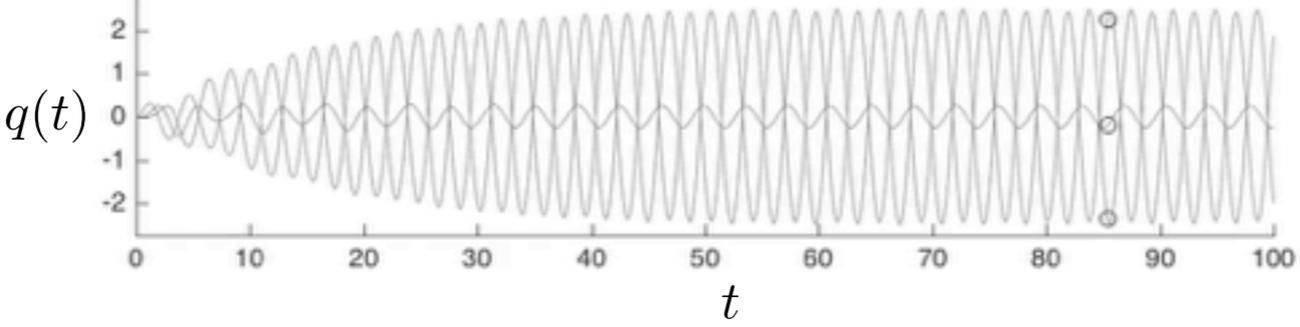
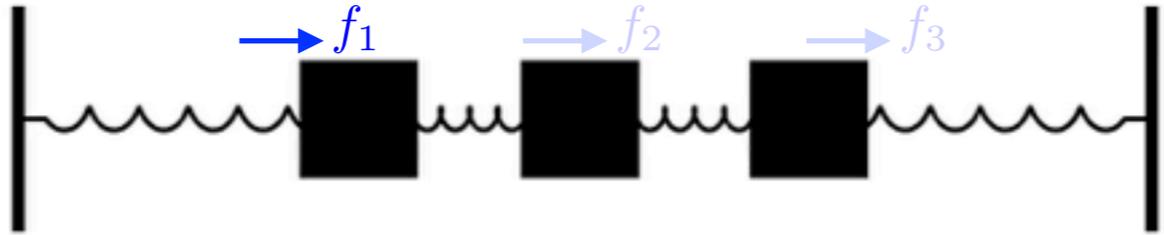
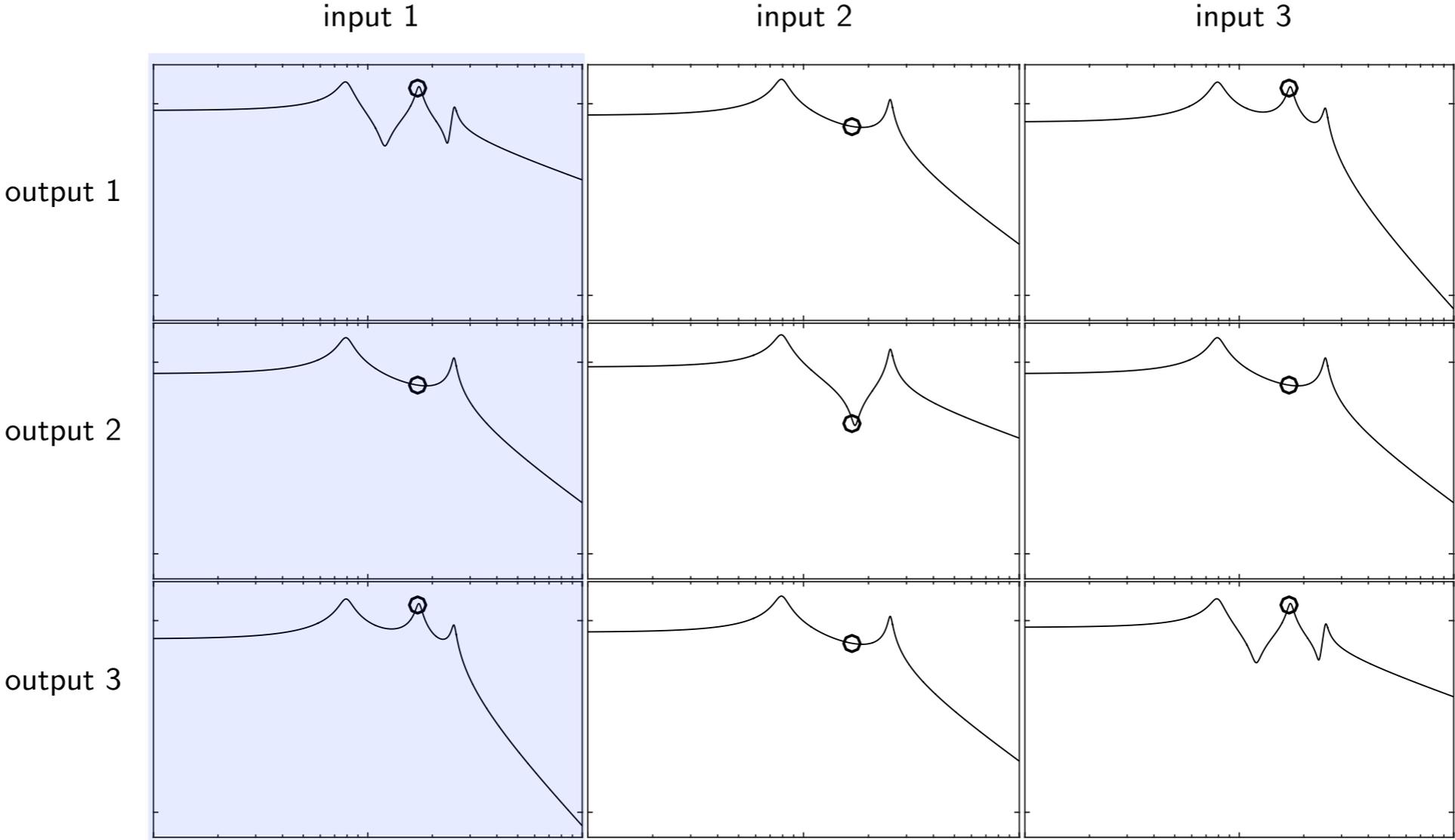
$$f(t) = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \cos(\omega_f t)$$

# frequency response: MIMO case



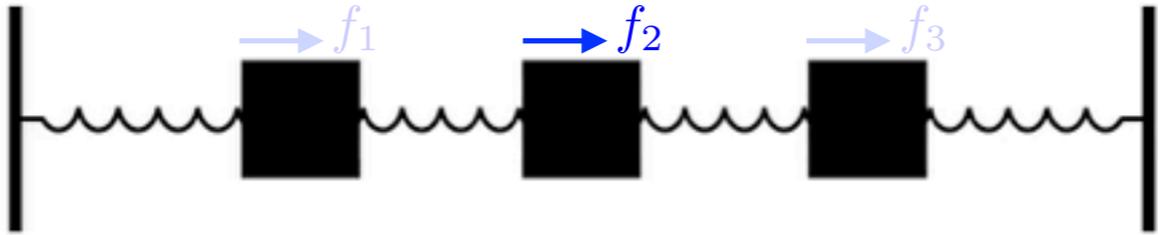
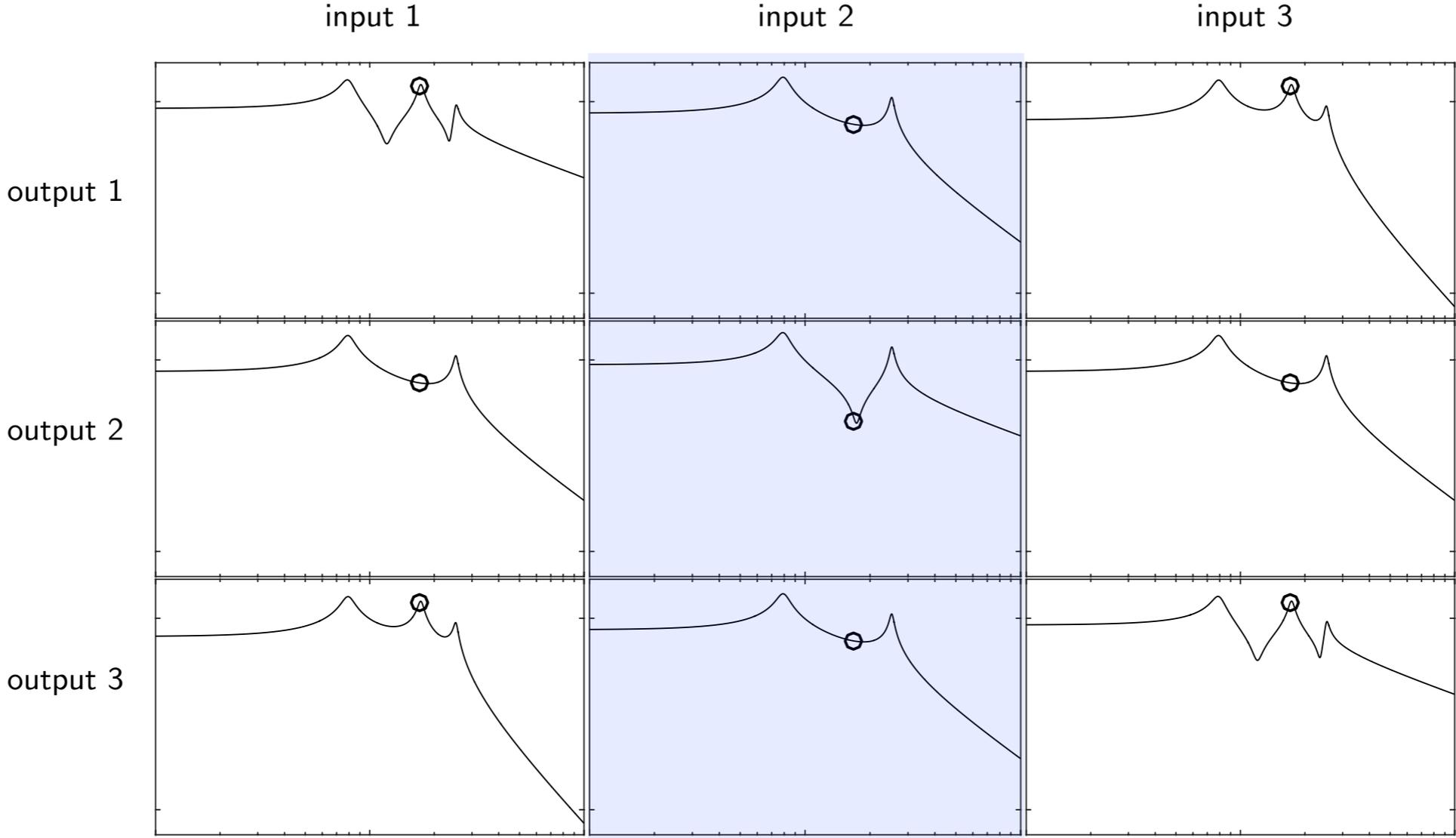
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# frequency response: MIMO case

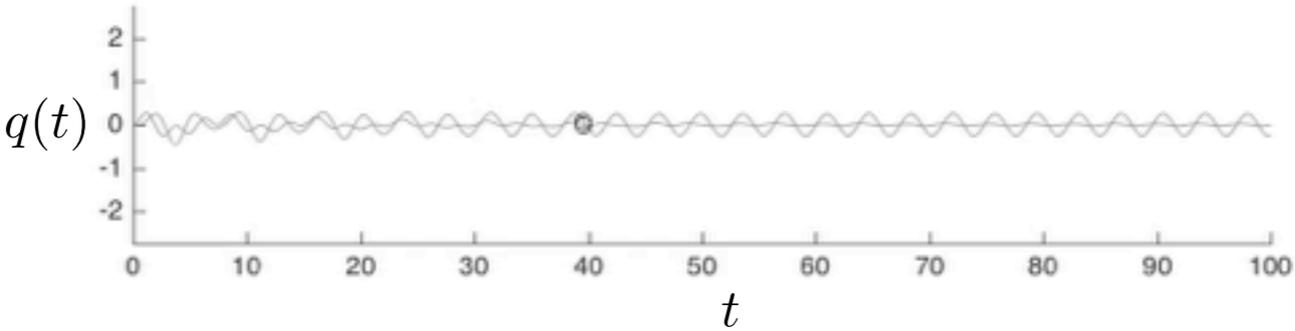


$$f(t) = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \cos(\omega_f t)$$

# frequency response: MIMO case

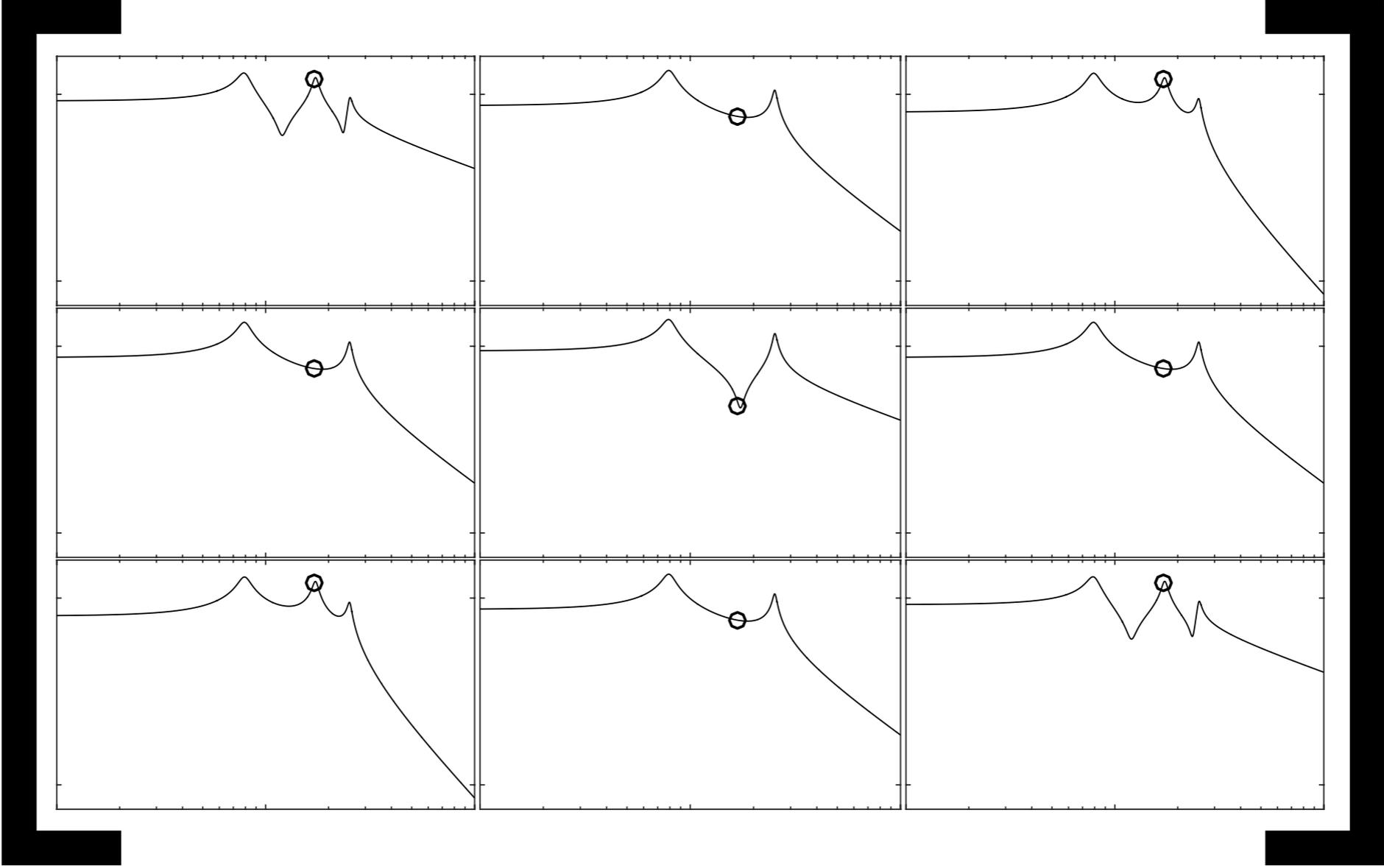


$$f(t) = \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} \cos(\omega_f t)$$



frequency response: MIMO case

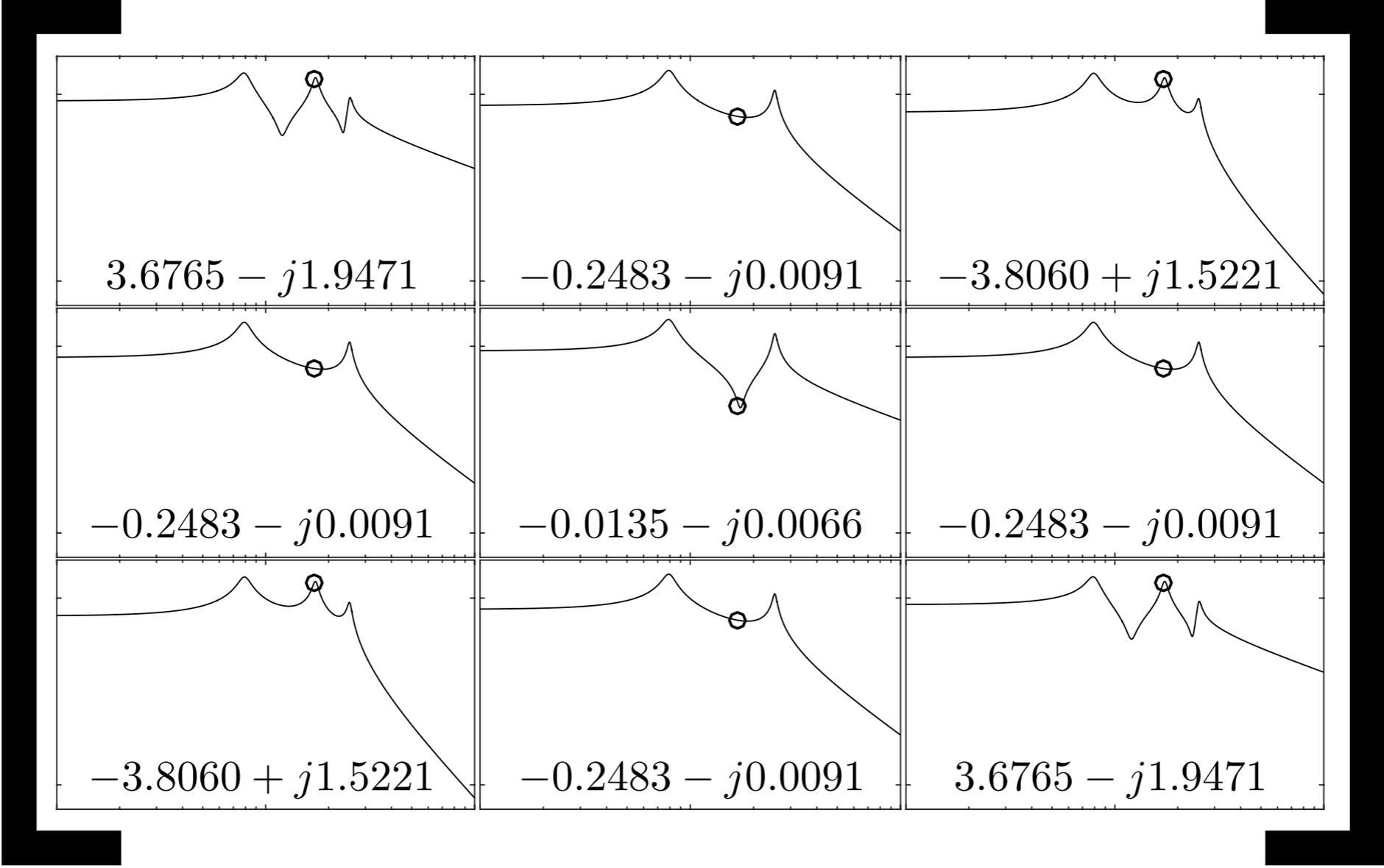
$$P(j\omega) =$$



$$Q(j\omega) = P(j\omega)F(j\omega)$$

# frequency response: MIMO case

$P(j\omega) =$



$$Q(j\omega) = P(j\omega)F(j\omega)$$

frequency response: MIMO case

$$P(j\omega) = \begin{bmatrix} P_{11}(j\omega) & P_{12}(j\omega) & P_{13}(j\omega) \\ P_{21}(j\omega) & P_{22}(j\omega) & P_{23}(j\omega) \\ P_{31}(j\omega) & P_{32}(j\omega) & P_{33}(j\omega) \end{bmatrix}$$

$$P(j\omega) = U\Sigma V^*$$

$$P(j\omega) = \begin{bmatrix} | & | & | \\ u_1 & u_2 & u_3 \\ | & | & | \end{bmatrix} \begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix} \begin{bmatrix} \text{---} v_1 \text{---} \\ \text{---} v_2 \text{---} \\ \text{---} v_3 \text{---} \end{bmatrix}$$

$$P(j\omega) = \sigma_1 u_1 v_1^* + \sigma_2 u_2 v_2^* + \sigma_3 u_3 v_3^*$$

- interpretation:

$\sigma_i$ : *gains* of the plant

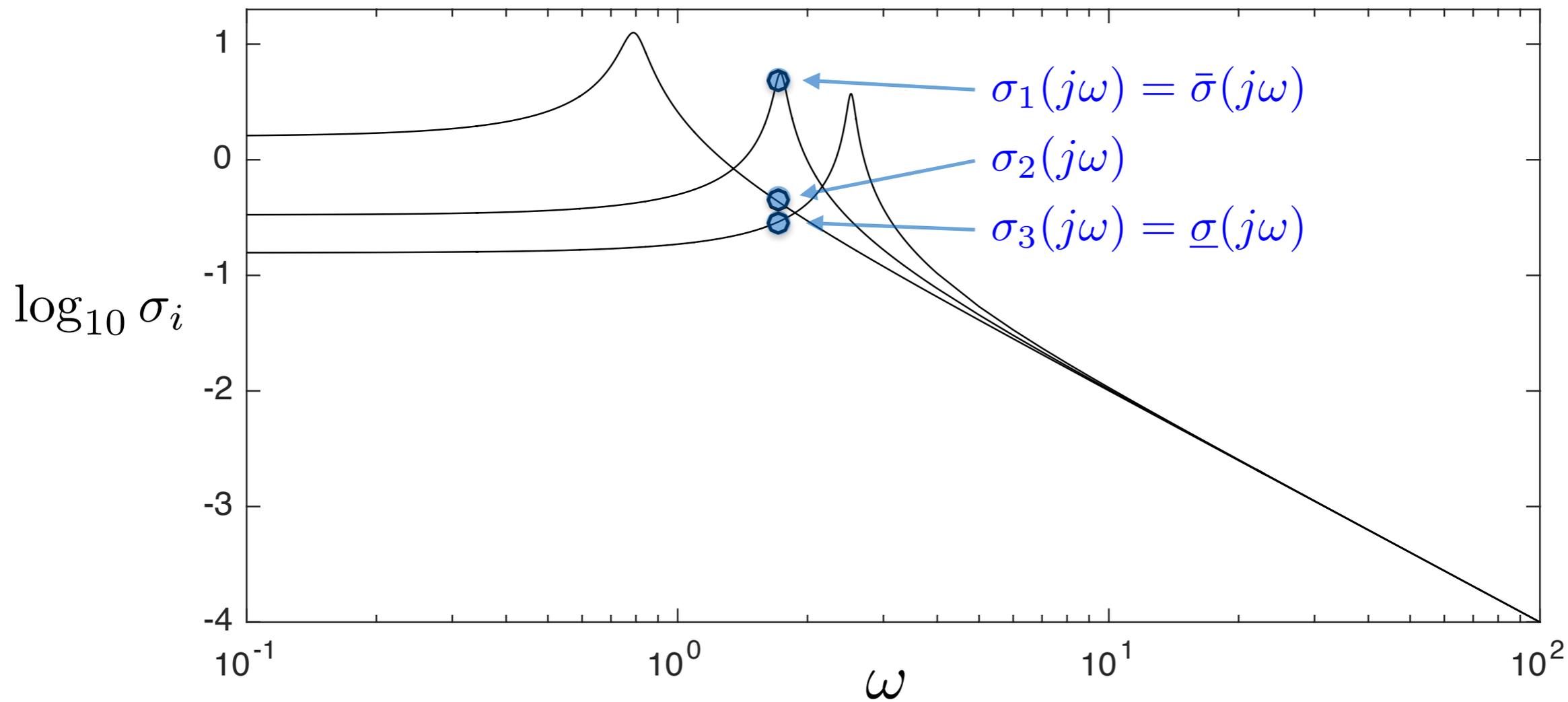
columns of  $V$ : *input directions* of the plant

columns of  $U$ : *output directions* of the plant

(columns of  $U$  and  $V$  are orthogonal and of unit length)

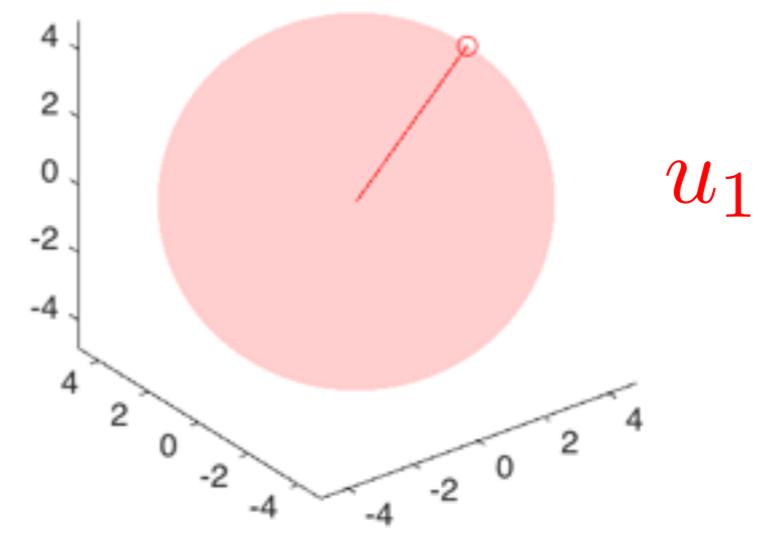
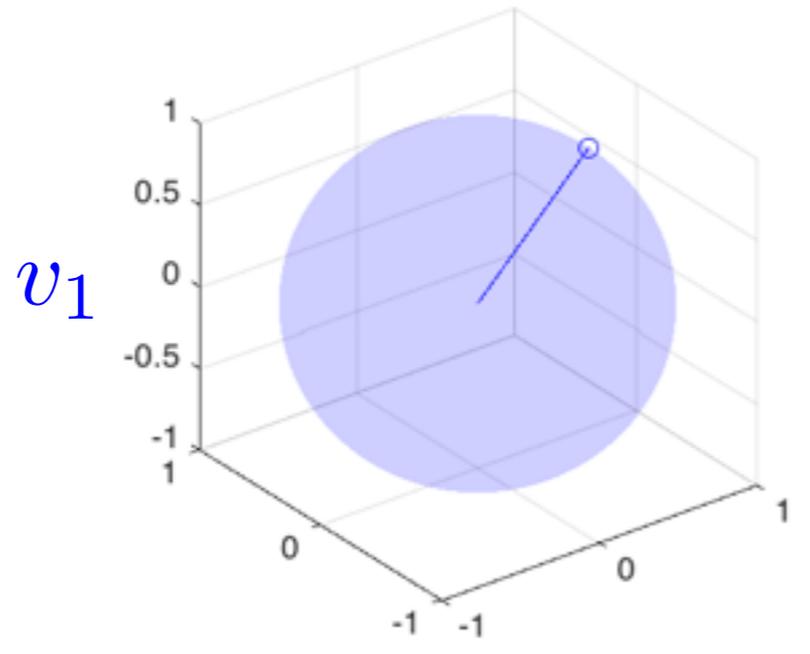
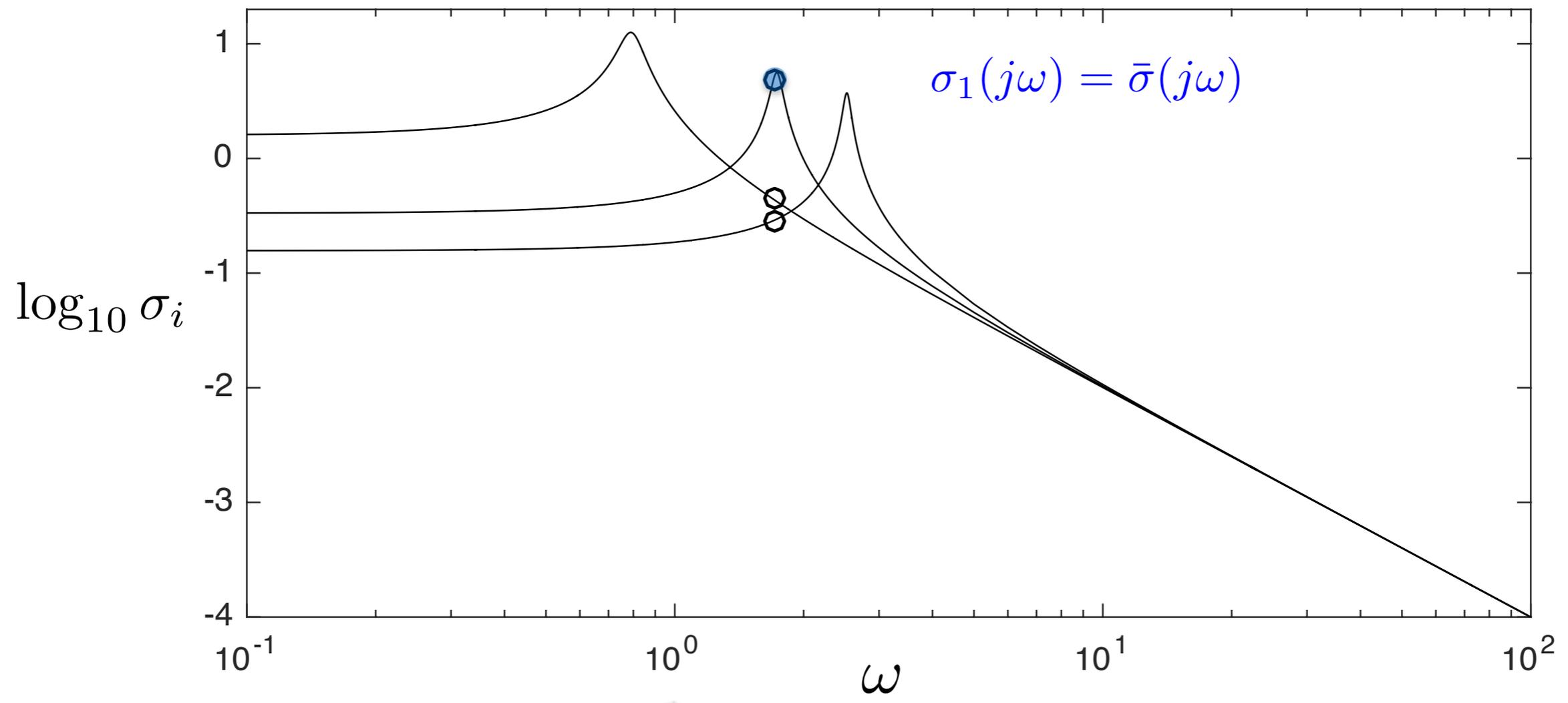
# frequency response: MIMO case

$$P(j\omega) = \begin{bmatrix} | & | & | \\ u_1 & u_2 & u_3 \\ | & | & | \end{bmatrix} \begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix} \begin{bmatrix} - & v_1 & - \\ - & v_2 & - \\ - & v_3 & - \end{bmatrix}$$

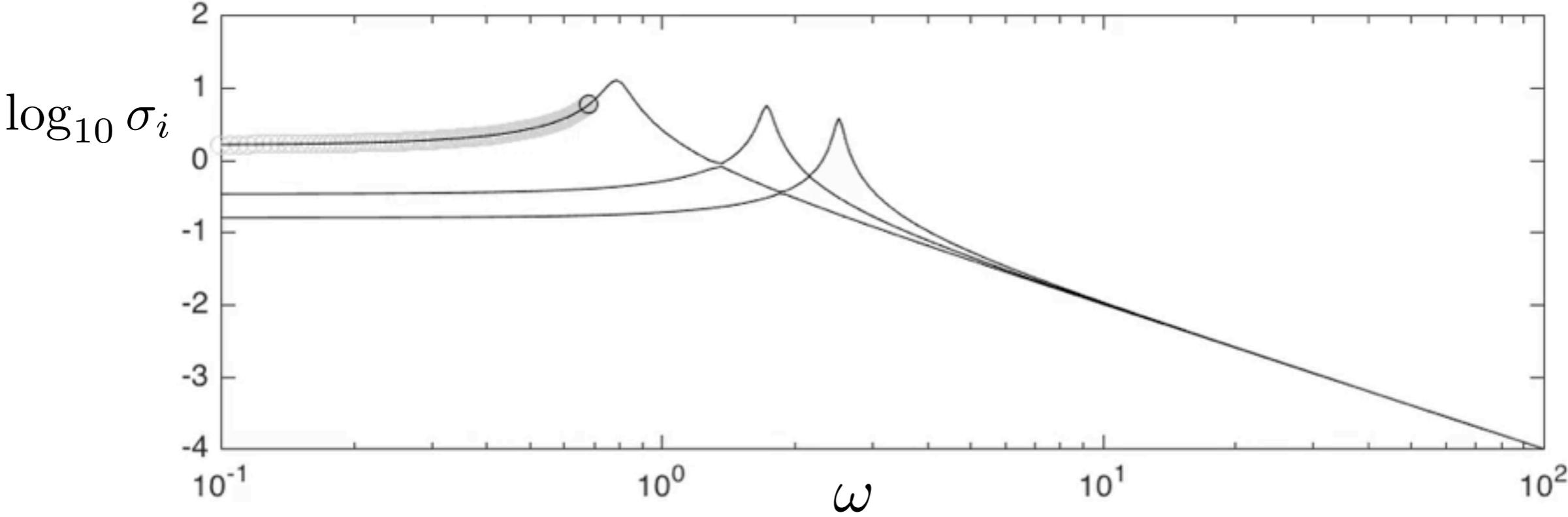


# frequency response: MIMO case

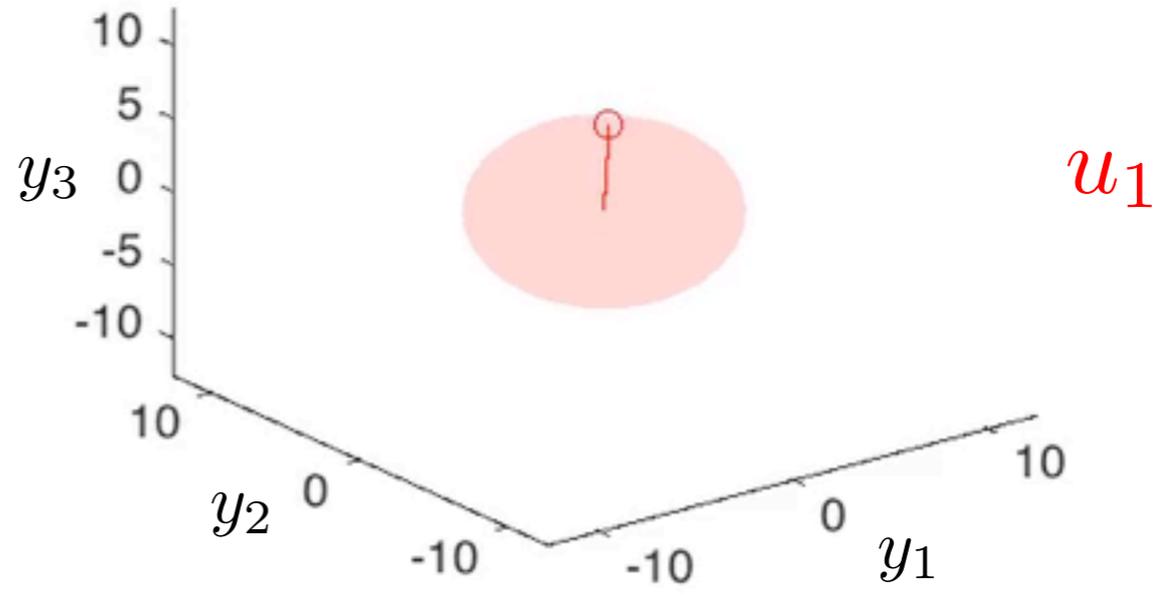
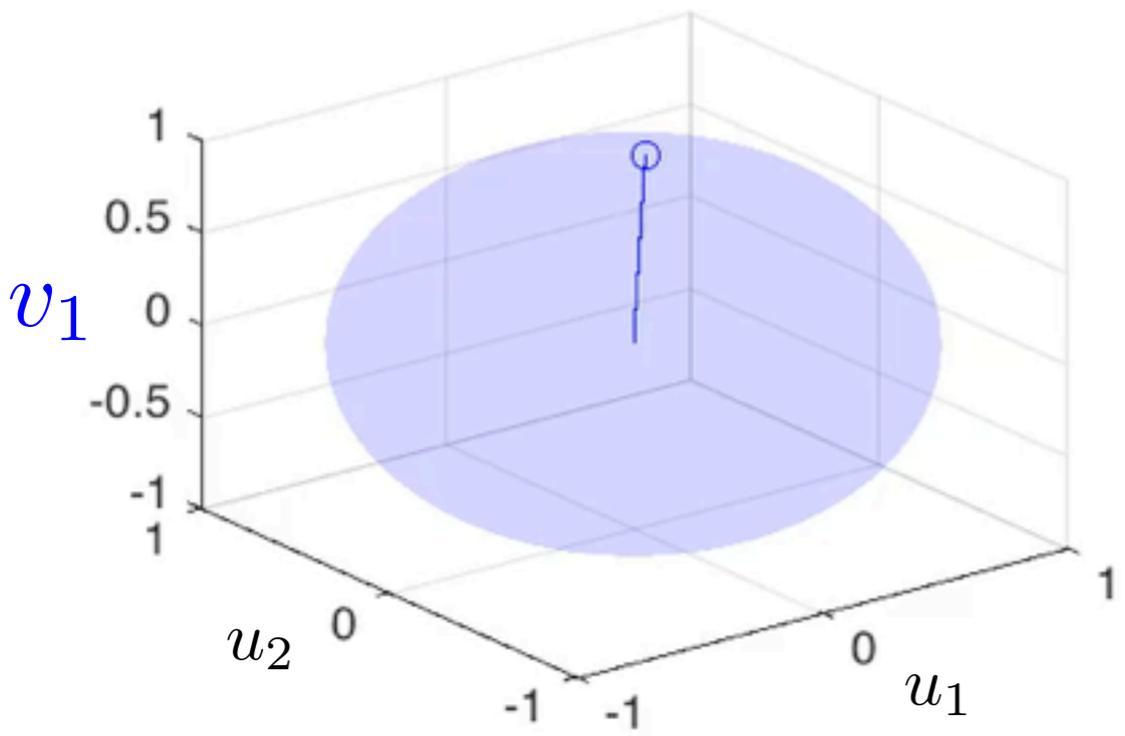
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# frequency response: MIMO case



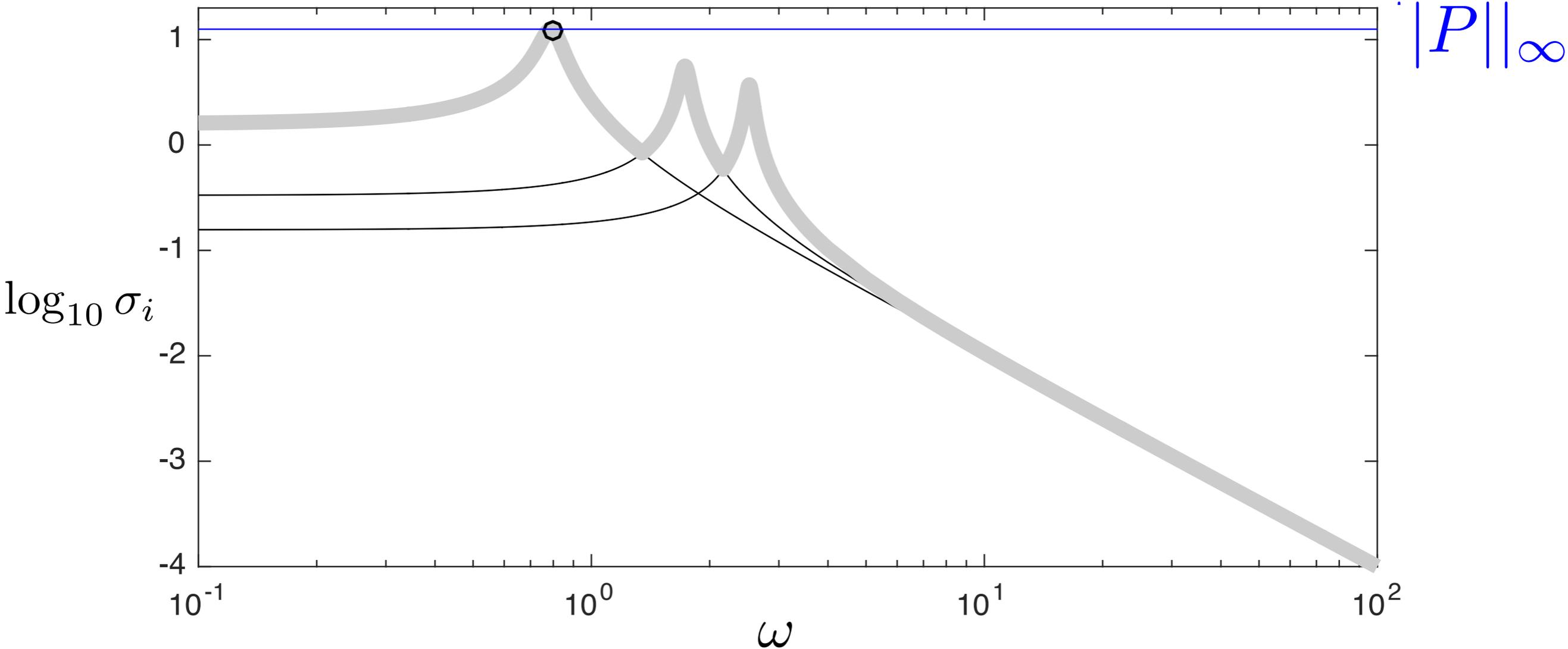
$u_2$



the  $\infty$ -norm: MIMO case

SISO:  $\|P\|_\infty = \max_{\omega} |P(j\omega)|$

MIMO:  $\|P\|_\infty = \max_{\omega} \sigma_1(P(j\omega))$

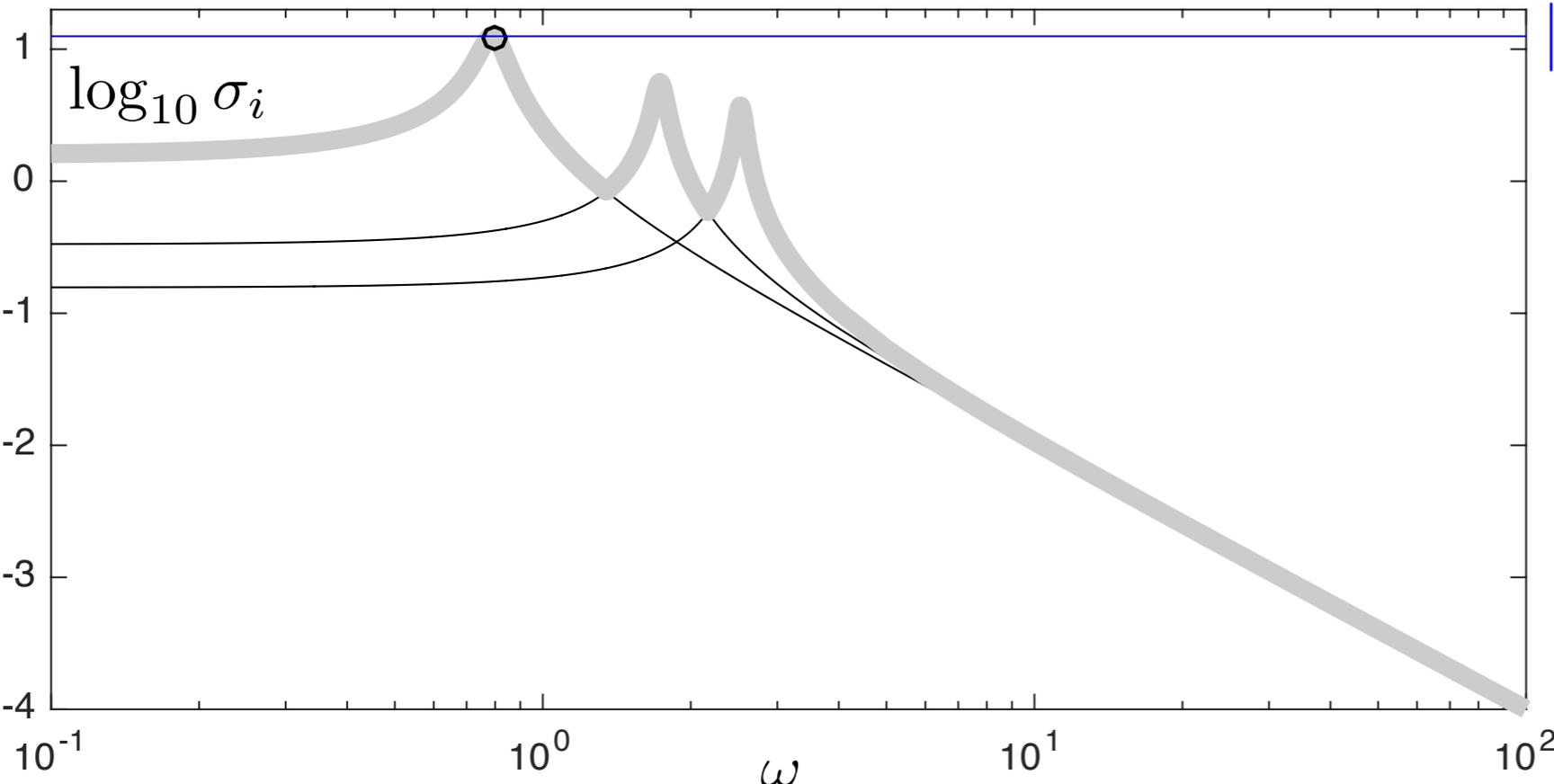


$\infty$ -norm: 'worst case' over all frequencies and all directions

compare with

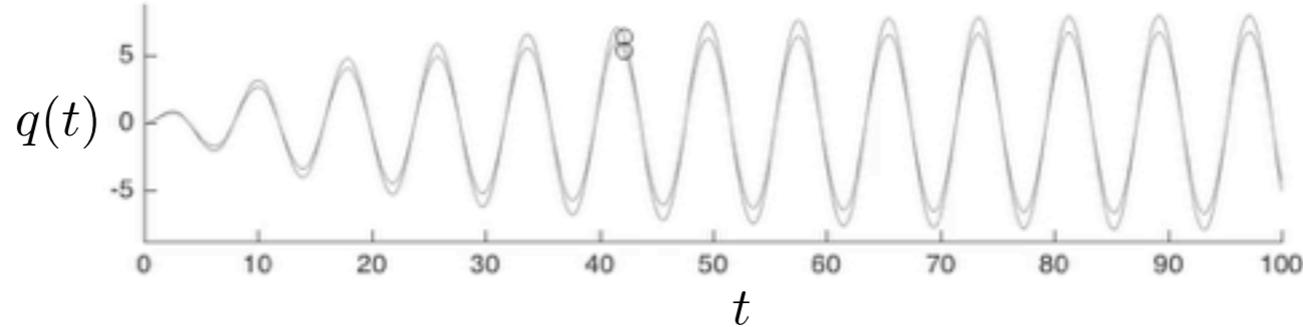
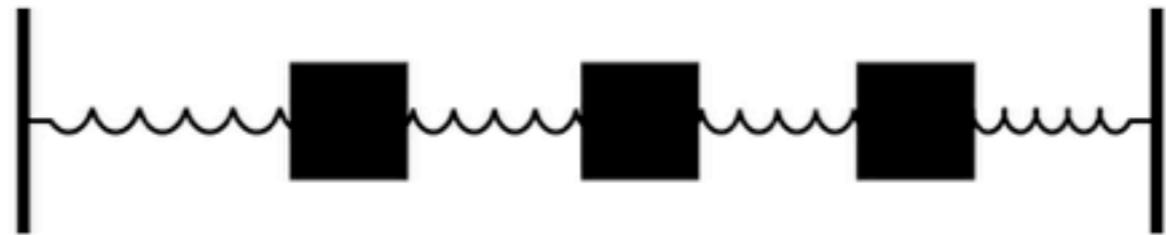
2-norm: average over all frequencies and directions

$\infty$ -norm: 'worst case' over all frequencies and all directions



$\|P\|_\infty$

$$f(t) = \begin{bmatrix} 0.542 \\ 0.643 \\ 0.542 \end{bmatrix} \cos(0.792 t)$$



# the $\infty$ -norm and model reduction

- suppose we have a plant of order  $n$

$$P_n$$

- and we want to approximate it by a reduced-order plant of order  $r < n$

$$P_r$$

- a good measure of the 'distance' between them is

$$\|P_n - P_r\|_\infty$$

# balanced truncation

$$\dot{x}(t) = A x(t) + B u(t)$$

$$y(t) = C x(t) + D u(t)$$

- suppose we decompose the state into two parts,  $x = [x_1 \quad x_2]^T$

$$\begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u(t)$$

$$y(t) = \begin{bmatrix} C_1 & C_2 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + D u(t)$$

- we will keep  $x_1$  and throw away  $x_2$ , leaving us with

$$\dot{x}_1(t) = A_1 x_1(t) + B_1 u(t)$$

$$y(t) = C_1 x_1(t) + D u(t)$$

# balanced truncation

how many states can we throw away, and how should we go about it?

# balanced truncation

how many states can we throw away, and how should we go about it?



# balanced truncation

how many states can we throw away, and how should we go about it?



# enter the observability/controlability Gramians

- observability Gramian:

$$W_o = \Psi_o^* \Psi_o = \int_0^\infty e^{A^* \tau} C^* C e^{A \tau} d\tau$$

- interpretation: for zero input,  $u(t) = 0$  and initial state  $x_0$ , the resulting output has energy

$$\|y\|_2^2 = \int_0^\infty y^*(\tau) y(\tau) d\tau = \int_0^\infty (C e^{A \tau} x_0)^* C e^{A \tau} x_0 d\tau = x_0^* W_o x_0$$

- if we consider initial states with  $|x_0| = 1$ , some will give higher output norms than others
- states giving larger output norms are considered more observable

# enter the observability/controlability Gramians

- controllability Gramian:

$$W_c = \Psi_c \Psi_c^* = \int_0^{\infty} e^{\tau A} B B^* e^{\tau A^*} d\tau$$

- interpretation: if we want to reach a state  $x_0$ , then the minimum input energy required to get there is

$$\|u\|^2 = x_0^* W_c^{-1} x_0$$

- if we consider initial states with  $|x_0| = 1$ , some are easier to “drive to” than others
- states that are easier to “drive to” are considered more observable

**balanced truncation:** change to coordinates in which the observability and controllability Gramians are *equal and diagonal*

- guaranteed error bound:

$$\sigma_{r+1} \leq \|P - P_r\|_\infty \leq 2(\sigma_{r+1} + \sigma_{r+2} + \cdots + \sigma_n)$$

- (“twice the sum of the tails”)
- note: balanced truncation is not optimal, but is (provably) not far off

balanced truncation: applied to Ginzburg-Landau system

$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t) \quad -\infty < x < \infty$$

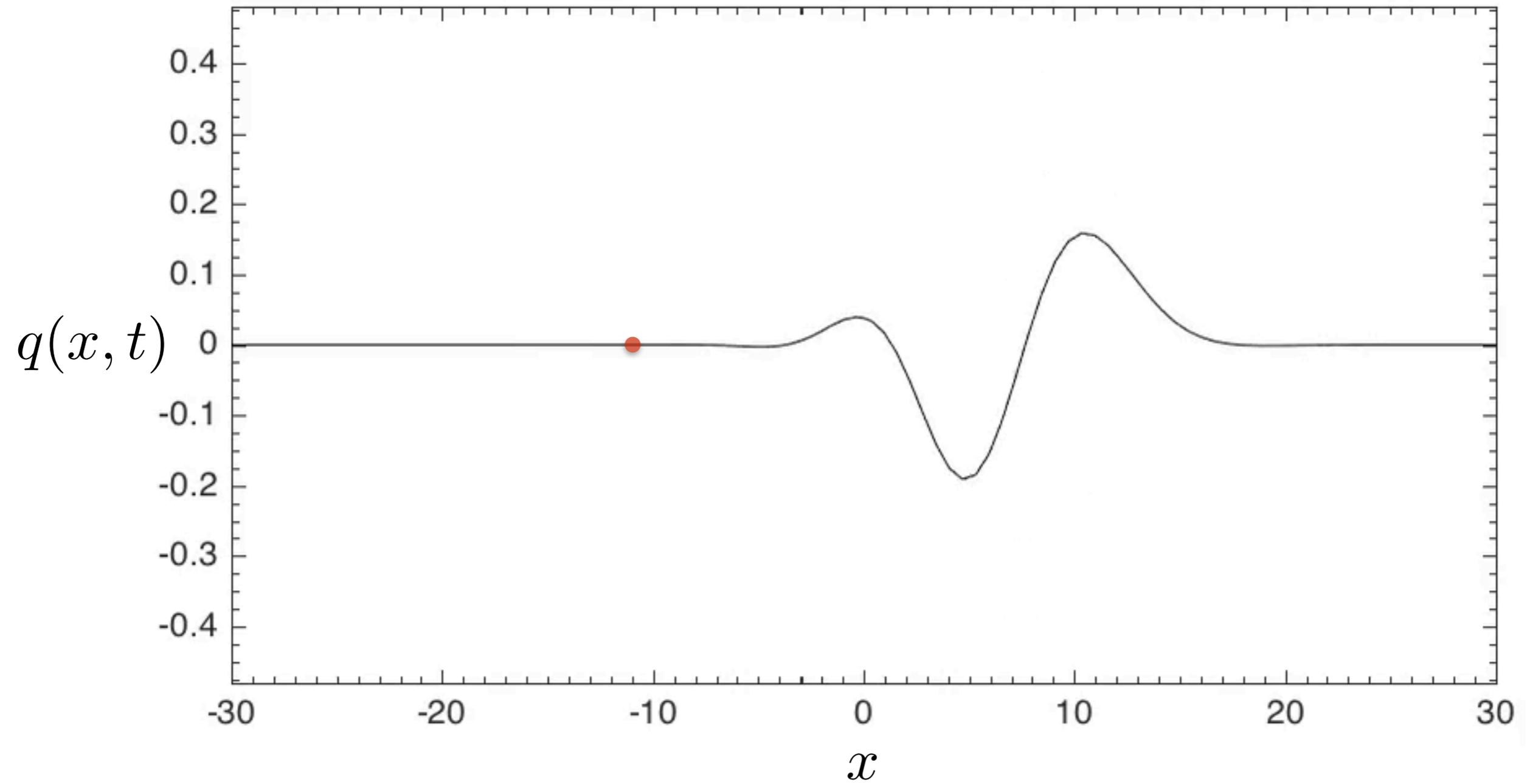
convection:  $\nu = U + i2c_u$

dissipation:  $\gamma = 1 + ic_d$

growth/decay:  $\mu(x) = \mu_0 - c_u^2 + \mu_2 x^2 / 2$

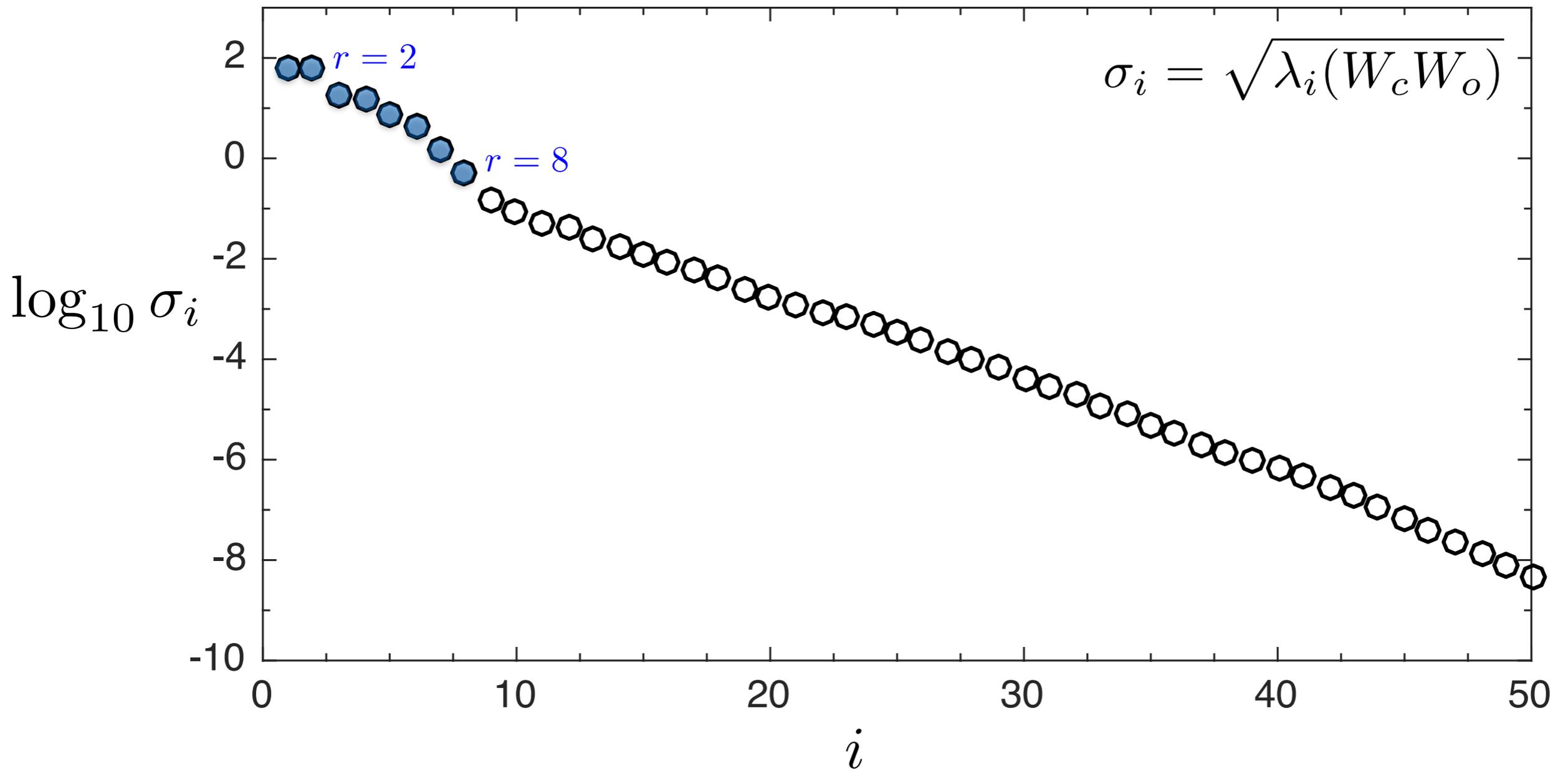
discretized using Hermite collocation method

# balanced truncation: applied to Ginzburg-Landau system

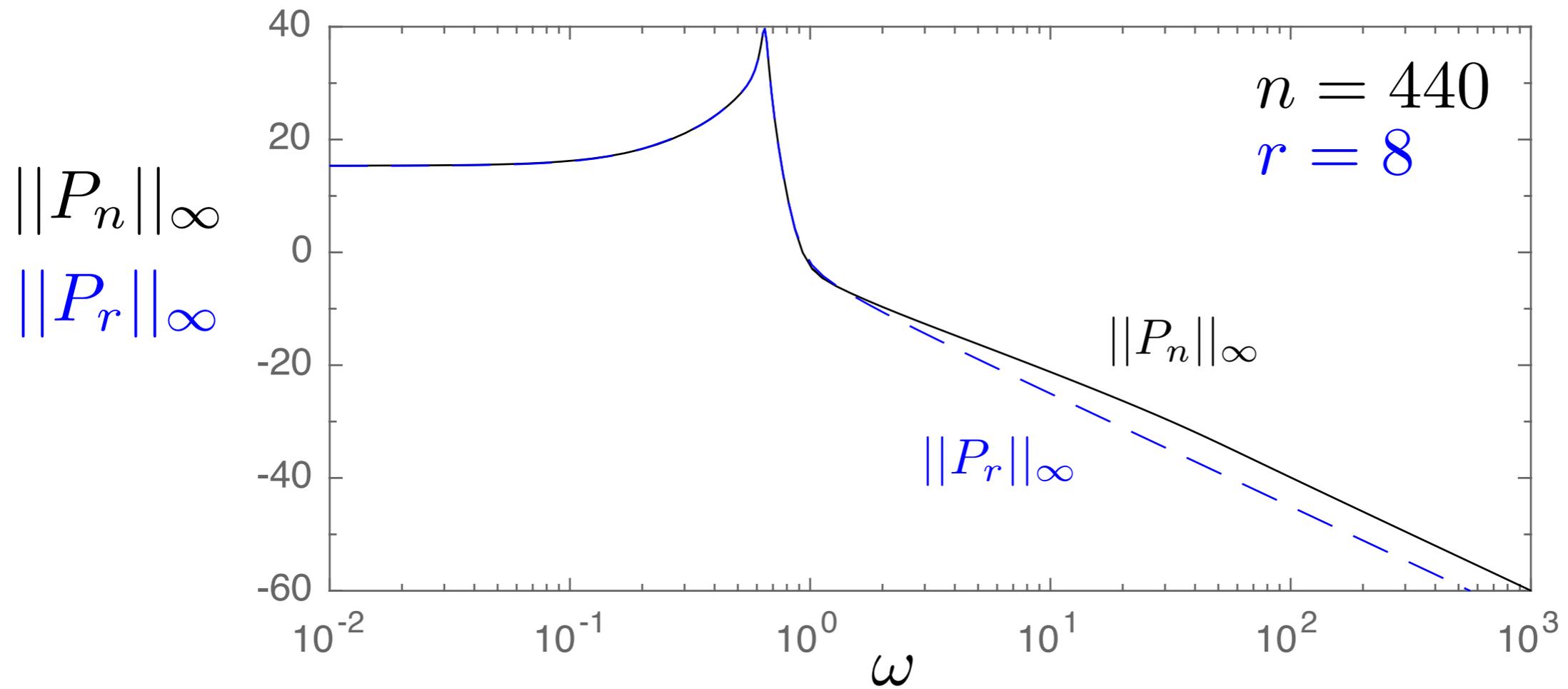


# balanced truncation: applied to Ginzburg-Landau system

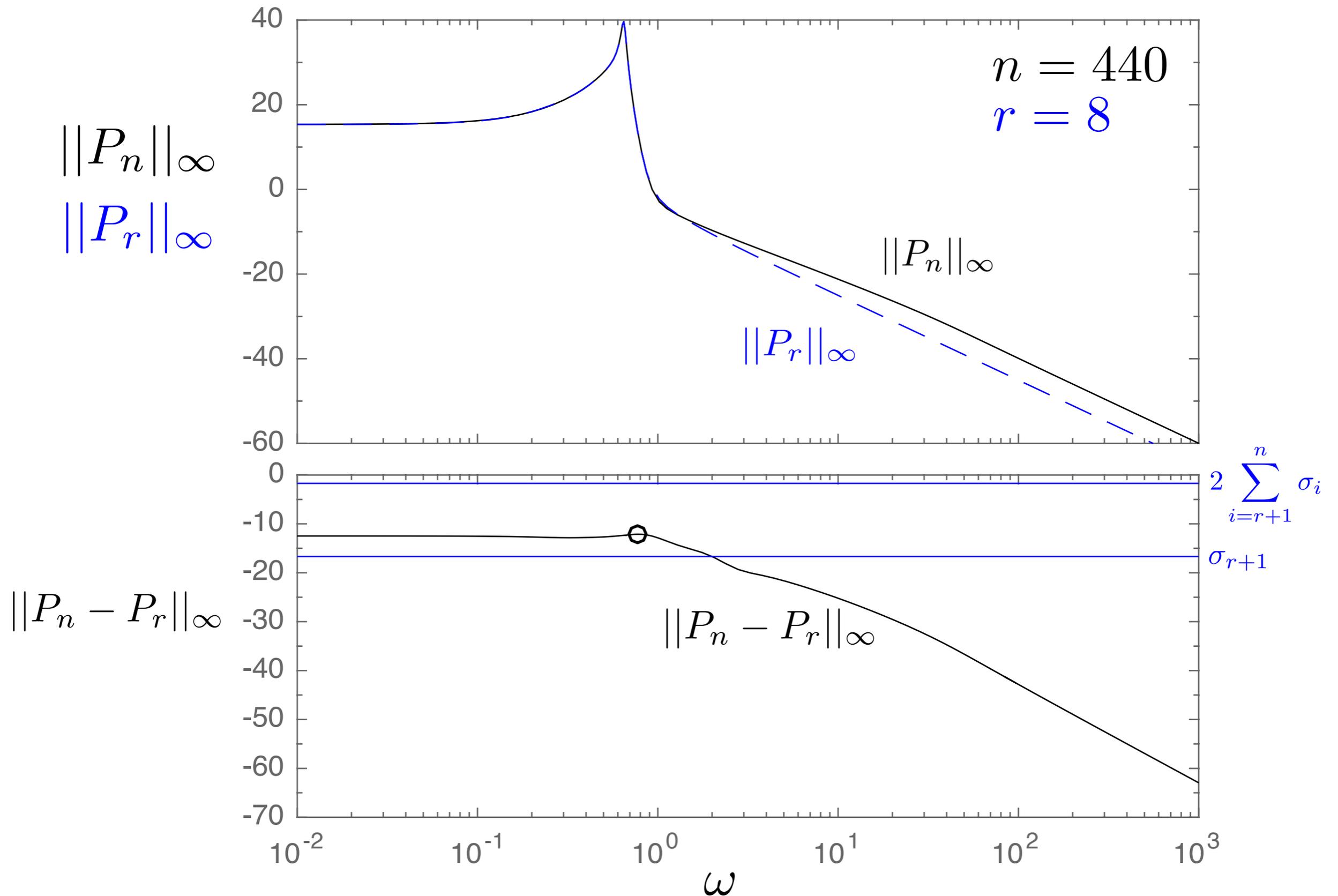
Hankel singular values,  $\sigma_i$



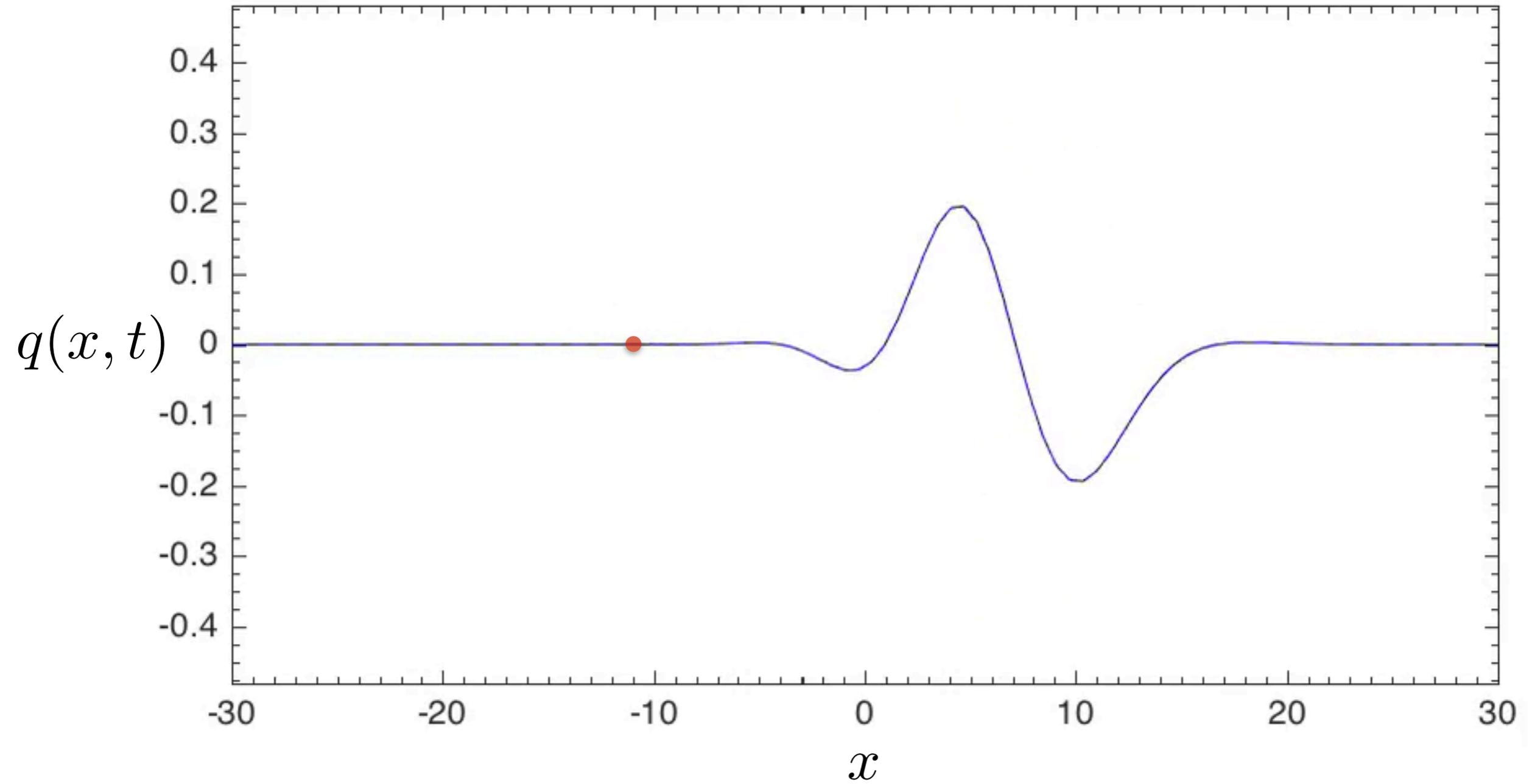
# balanced truncation: applied to Ginzburg-Landau system



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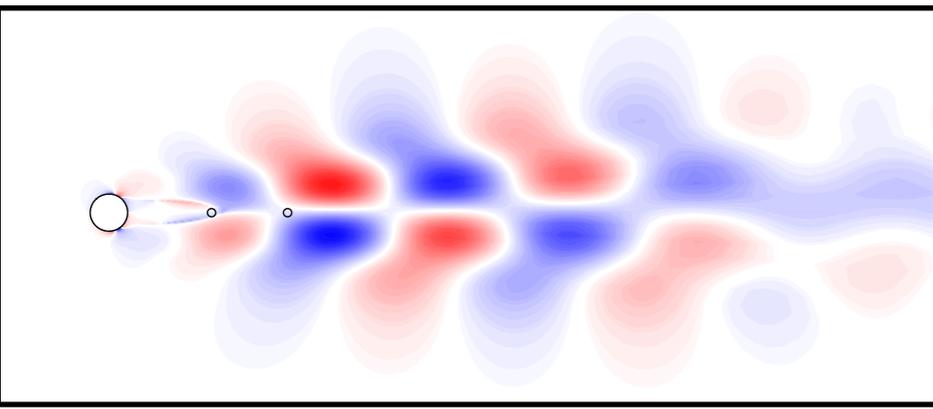
some applications

some applications

1. estimation
2. control

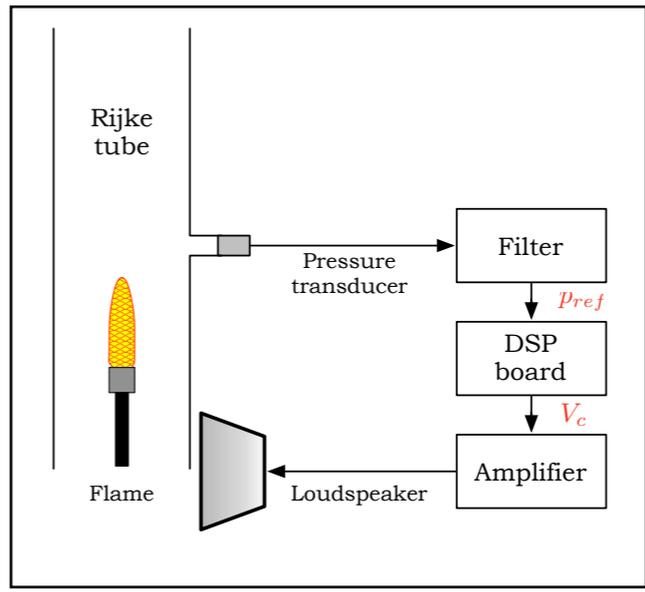
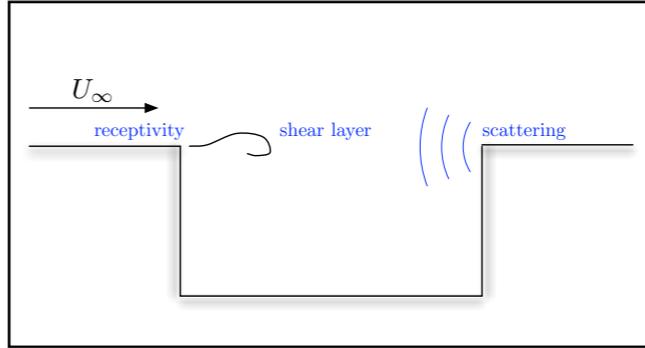
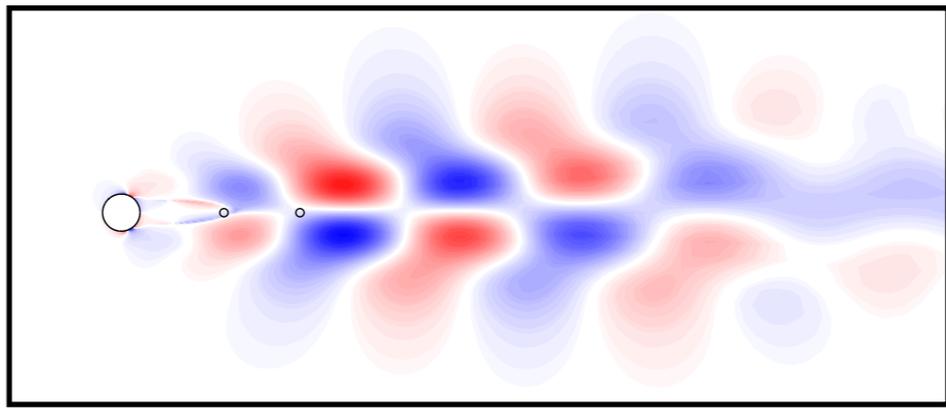
# some applications

## 1. estimation



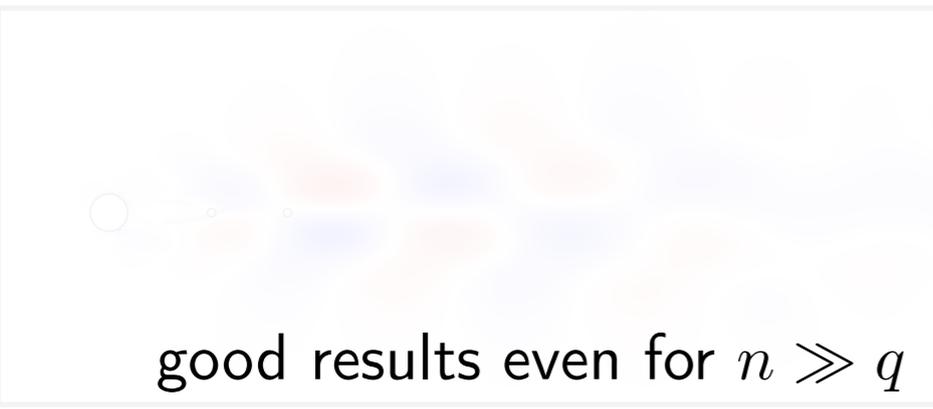
$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

## 2. control



# the take-away messages

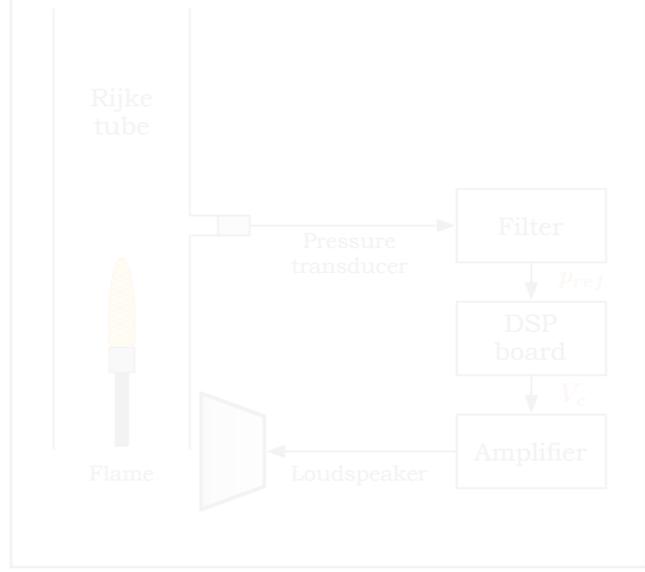
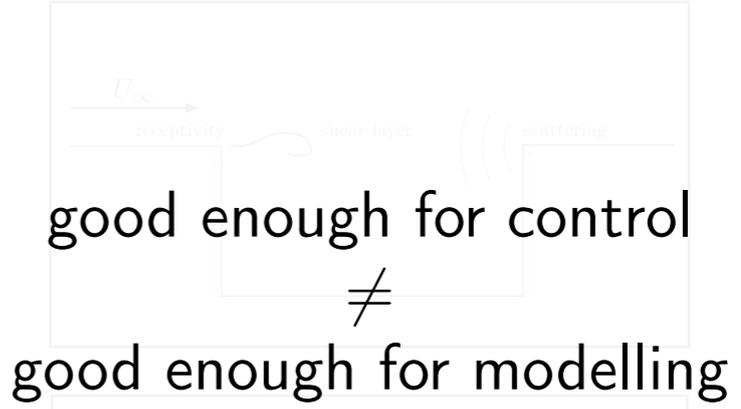
## 1. estimation



$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

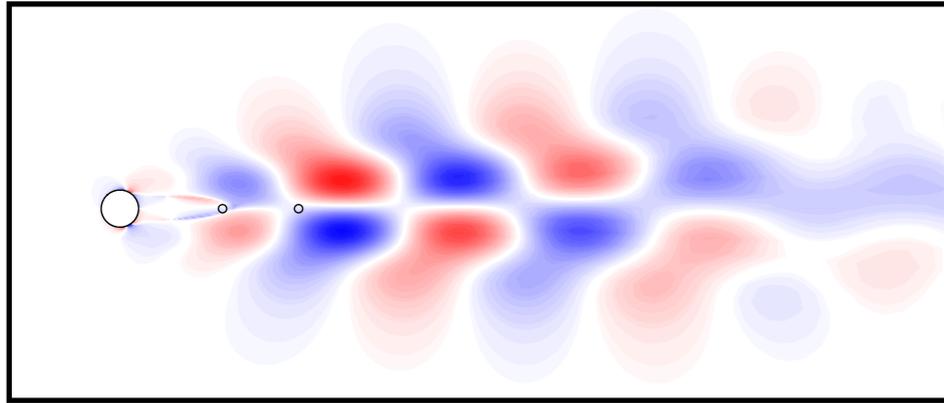
'local' or 'integral' sensing can work

## 2. control



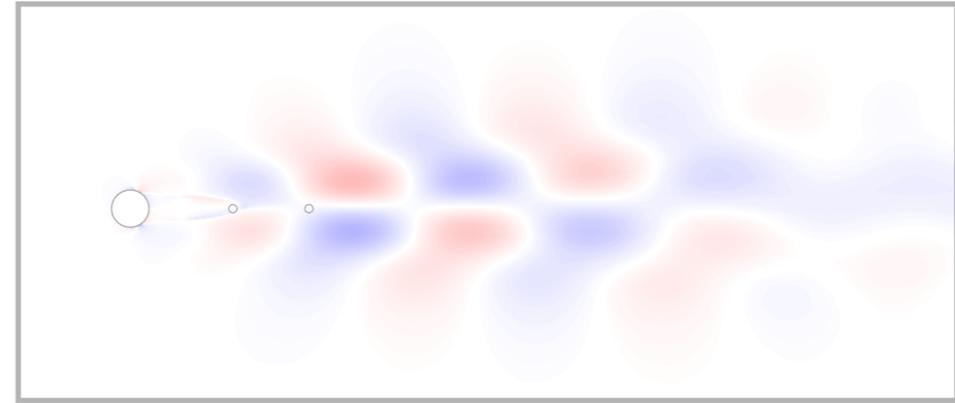
# some applications

## 1. estimation

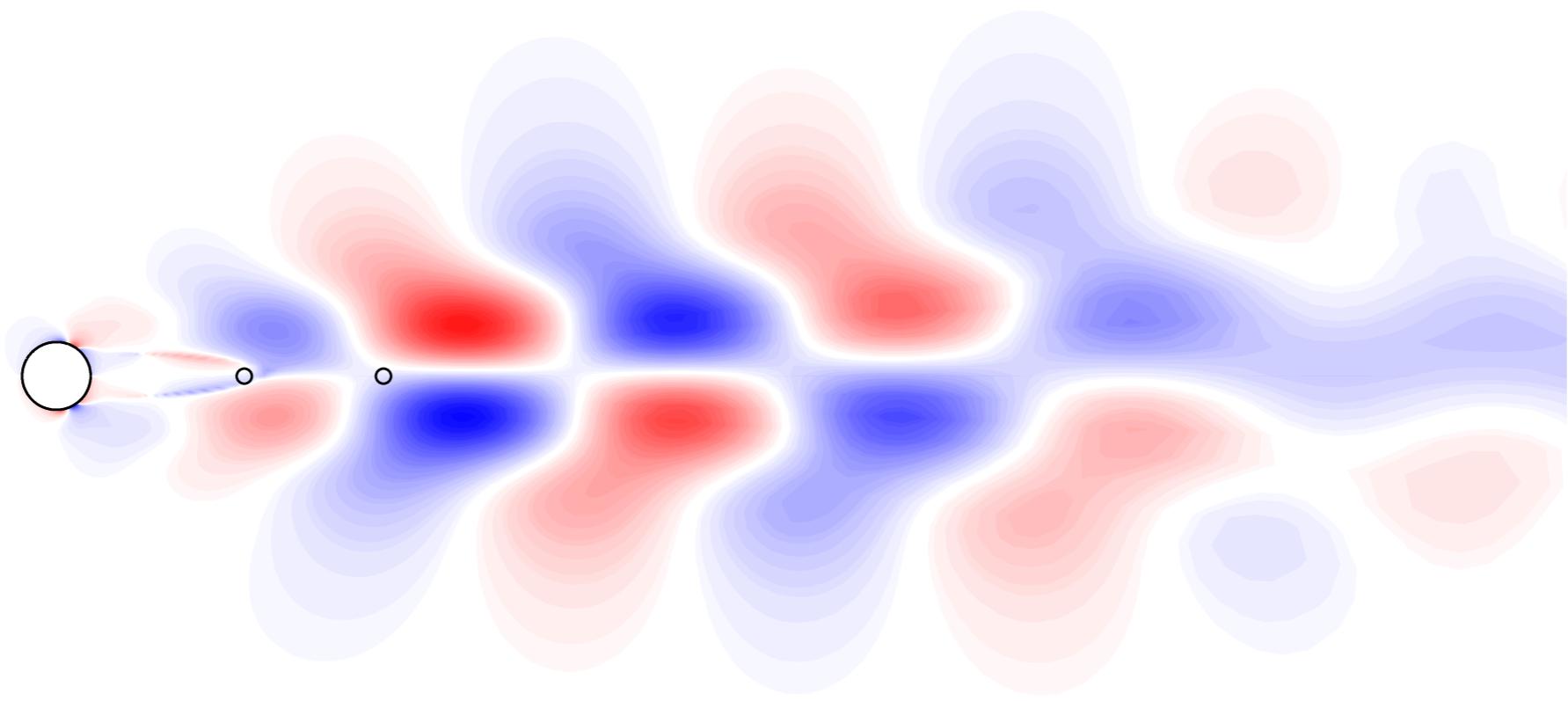


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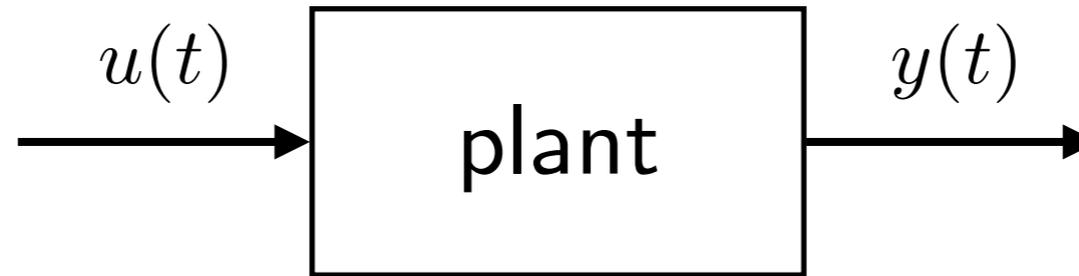
## 2. control



estimation



# dynamic estimation



$$\dot{x}(t) = Ax(t) + Bu(t)$$

$$y(t) = Cx(t) + Du(t)$$

*u(t) is the input*

*y(t) is the output*

*x(t) is the state,  $x(t) \in \mathbb{R}^n$*

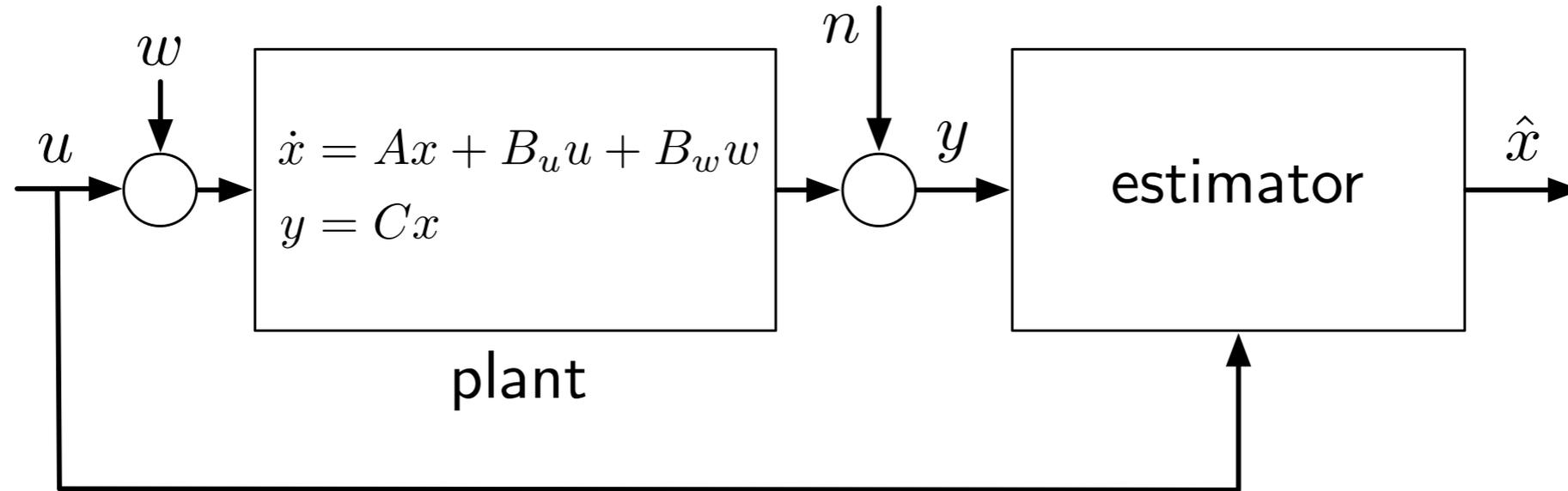
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$$\dot{\hat{x}}(t) = A\hat{x}(t) + Bu(t) + L(y(t) - \hat{y}(t))$$

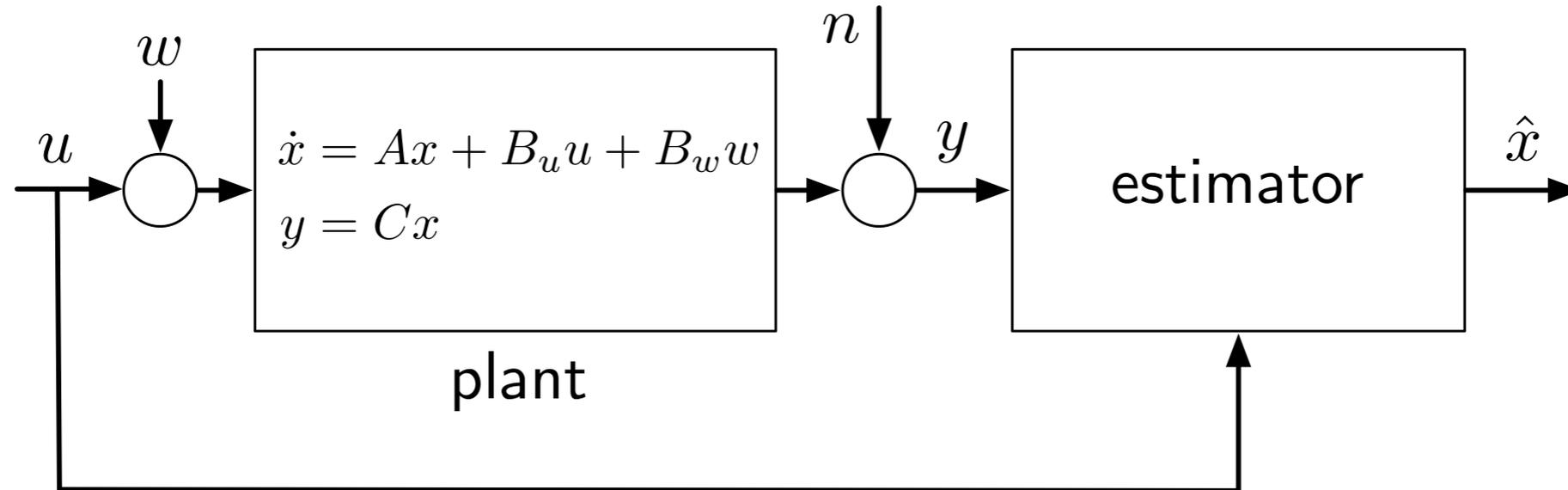
$$\hat{y}(t) = C\hat{x}(t) + Du(t)$$

# dynamic estimation



- define error: 
$$e(t) = \hat{x}(t) - x(t)$$
- then error satisfies: 
$$\dot{e}(t) = (A - LC) e(t)$$
- can specify error dynamics by suitable choice of the matrix  $L$
- Kalman filter amounts to a specific choice of  $L$

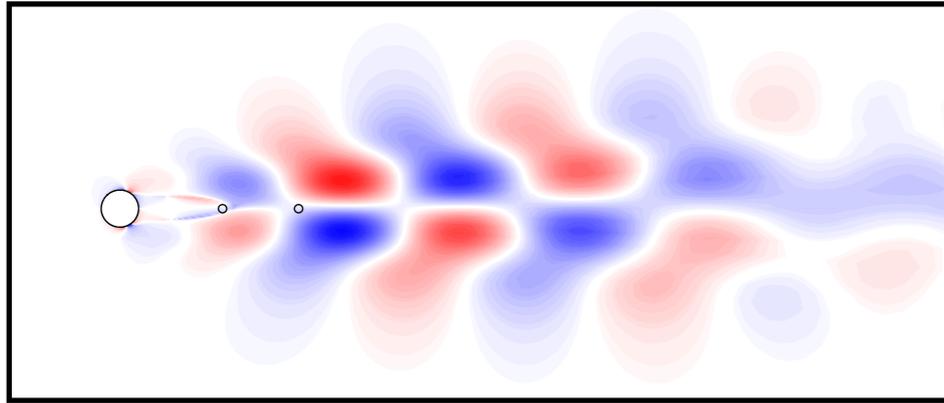
# dynamic estimation



- the Kalman filter:
  - is dynamic: i.e. it uses time-resolved data to form an estimate
  - accommodates unknown disturbances  $w$  and sensor noise  $n$  in its framework

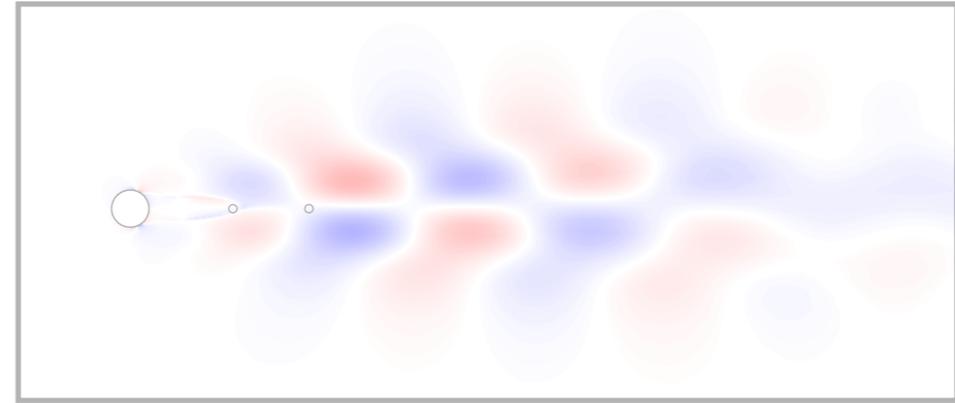
# some applications

## 1. estimation

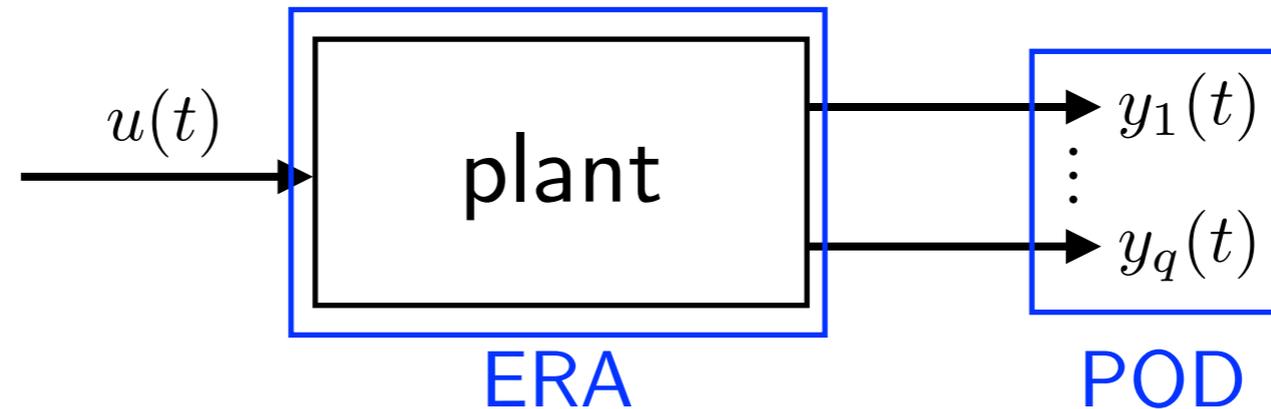


$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

## 2. control



POD is used to reduce the number of outputs



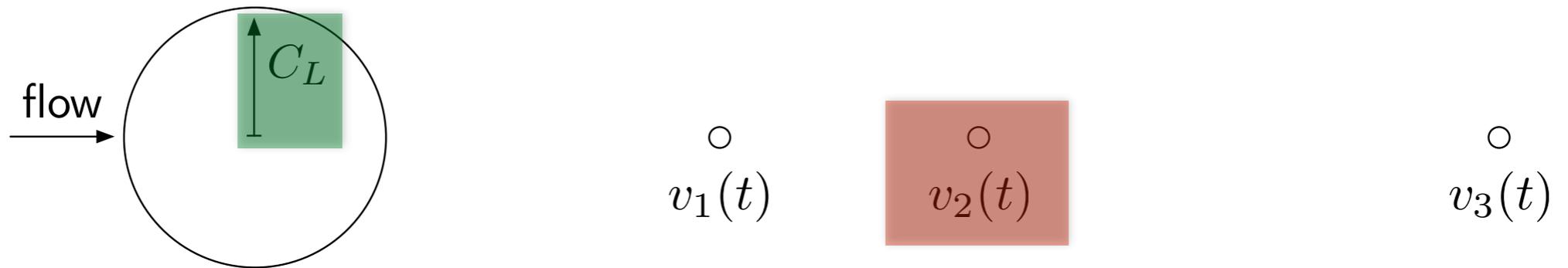
- DNS has  $256 \times 220 = 56320$  outputs
- solution: decompose output into leading POD modes
- 31 POD modes are used
- the ERA model order is 29 (order 4 performs almost as well)

# results

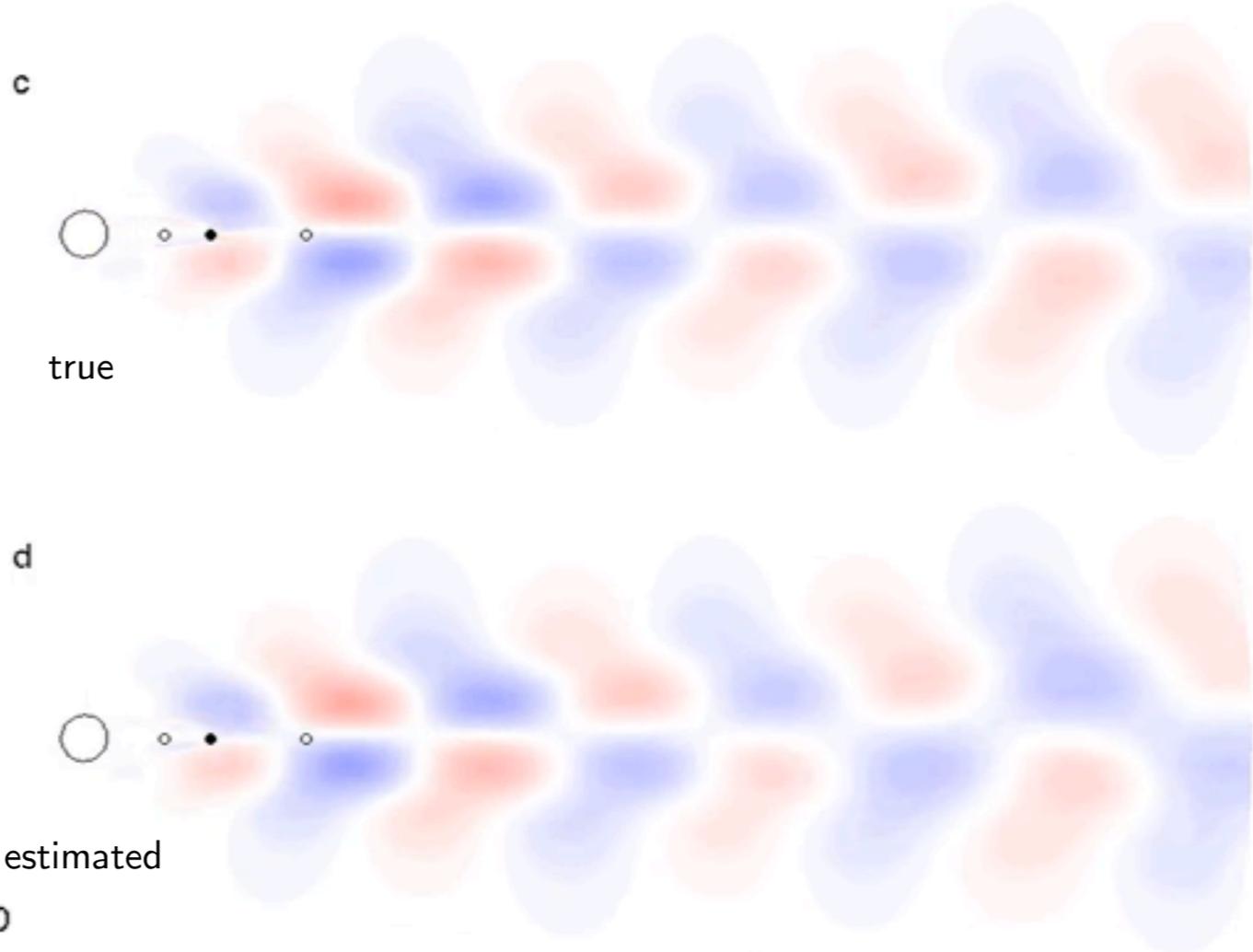
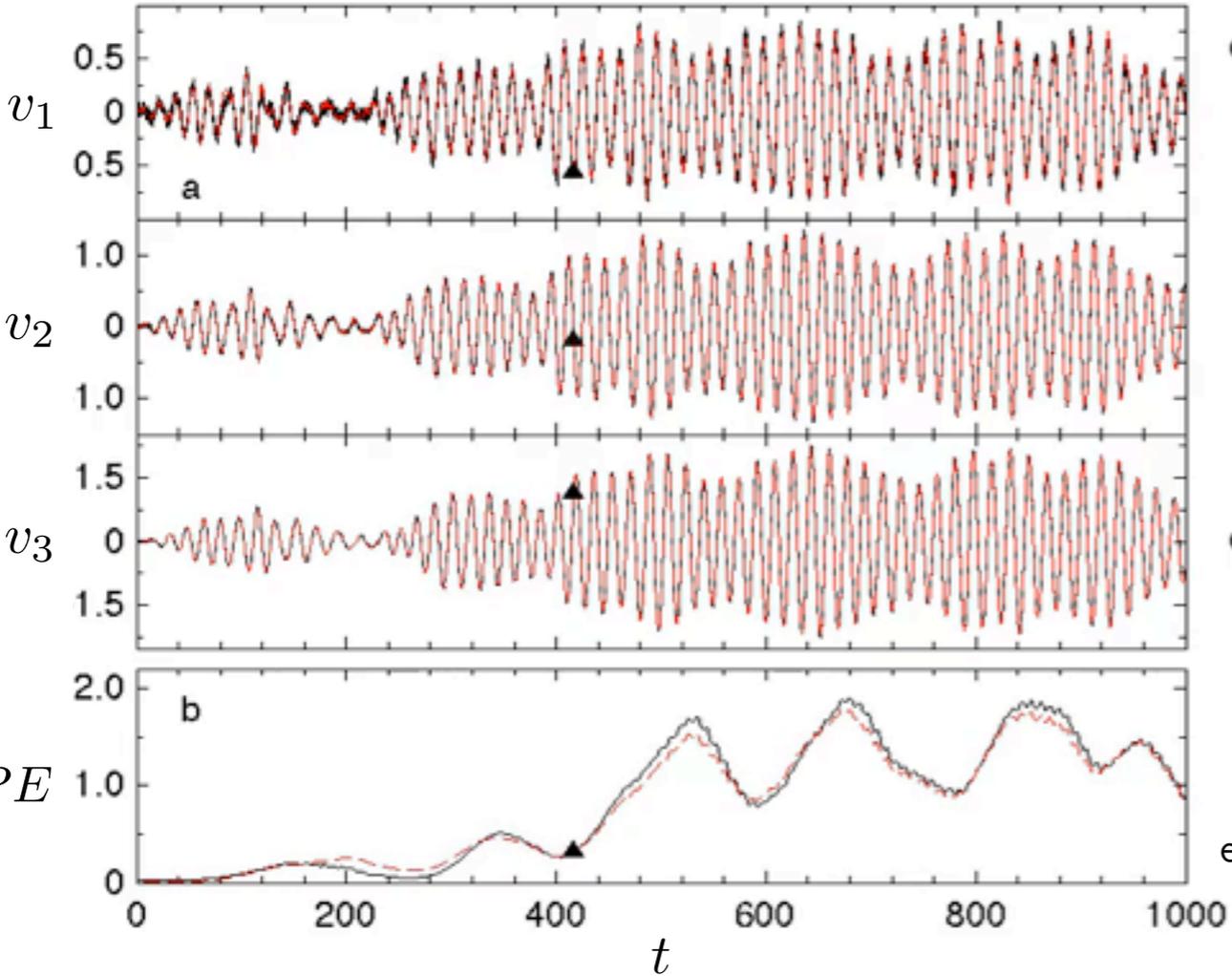
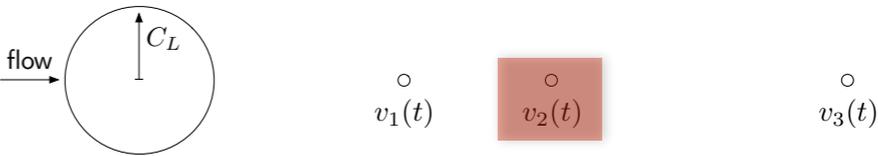
- we estimate the entire flow using

i. transverse velocity at sensor two only

ii. lift force only



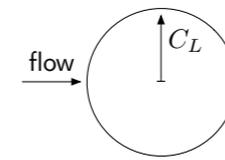
# transverse velocity at sensor two only



— true  
- - - estimated

good results even for  $n \gg q$

# performance in time



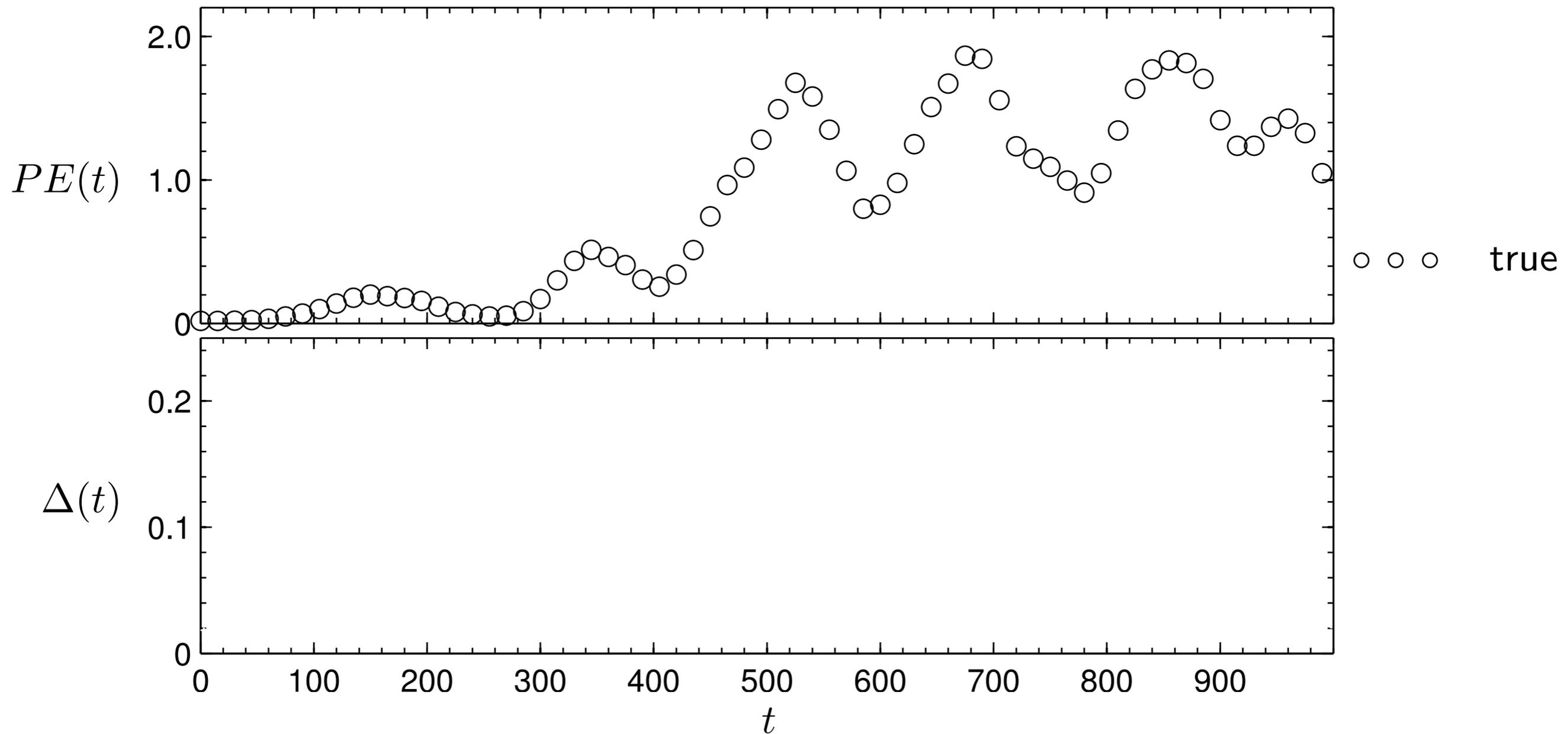
$v_1(t)$

$v_2(t)$

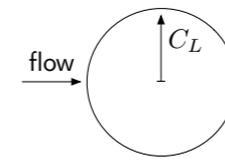
$v_3(t)$

$$PE(t) = \frac{1}{2} \iint u_T^2(x, y, t) dx dy$$

$$u_T = \sqrt{u^2 + v^2}$$



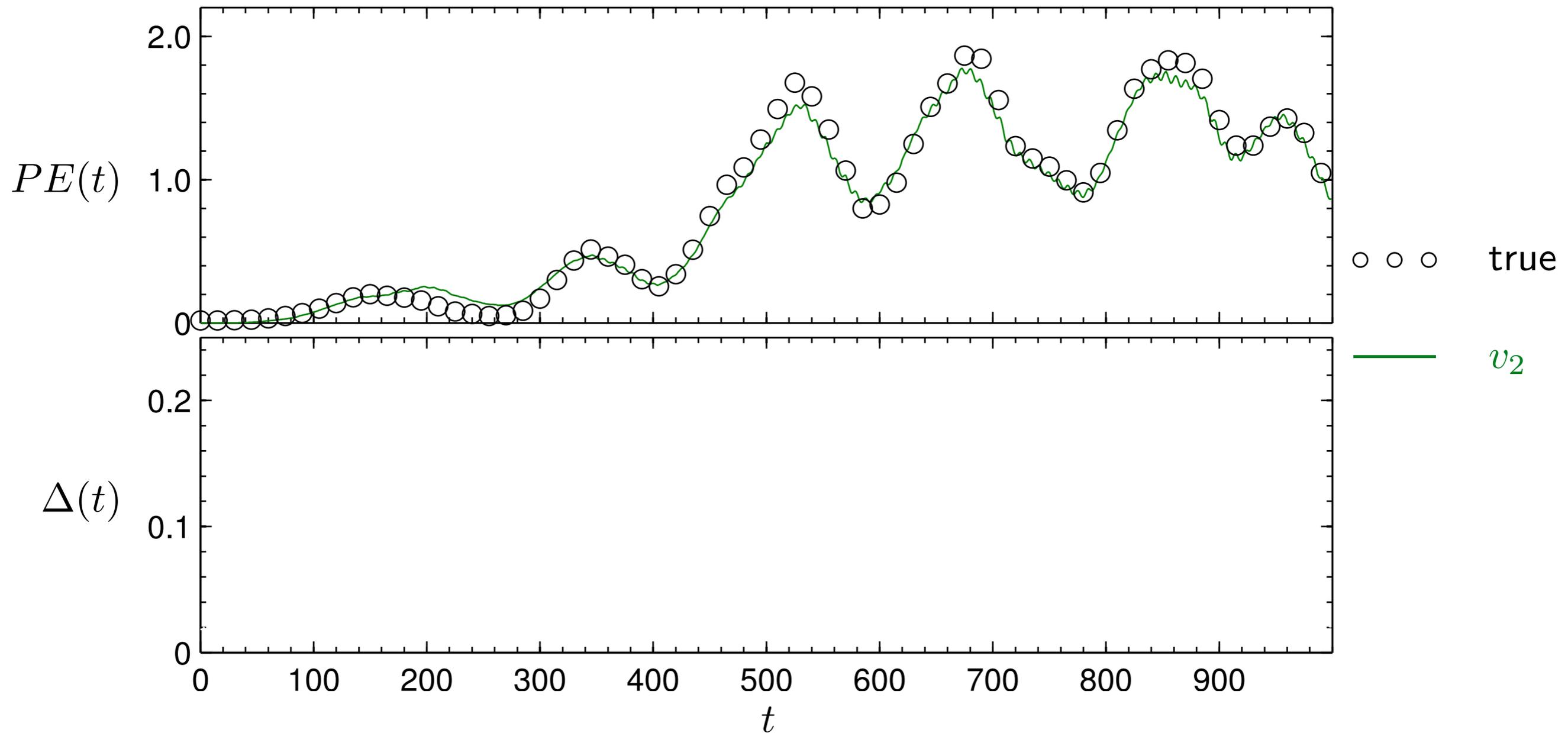
# performance in time



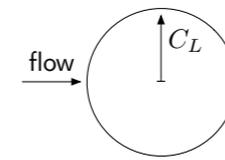
$v_1(t)$        $v_2(t)$        $v_3(t)$

$$PE(t) = \frac{1}{2} \iint u_T^2(x, y, t) dx dy$$

$$u_T = \sqrt{u^2 + v^2}$$



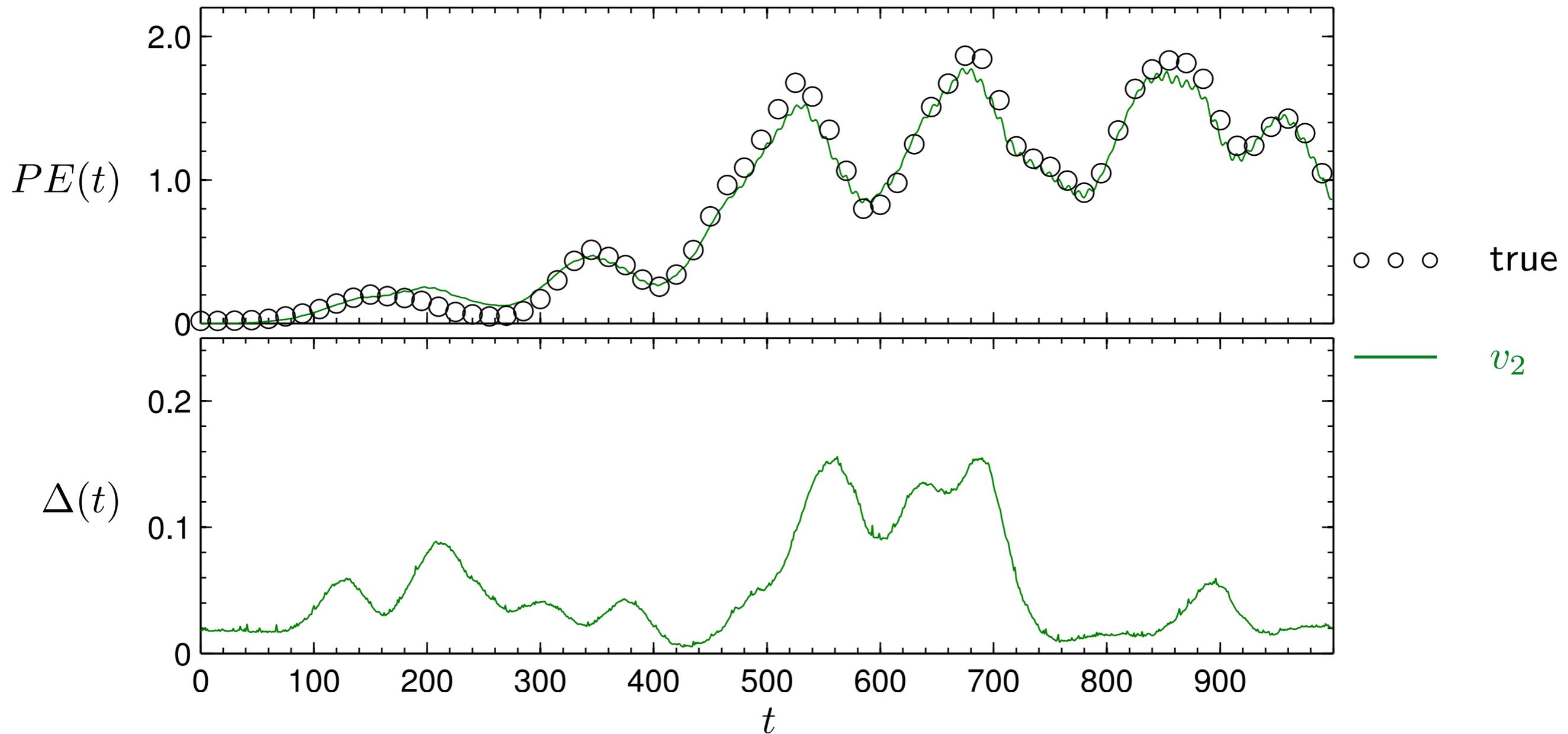
# performance in time

 $v_1(t)$  $v_2(t)$  $v_3(t)$ 

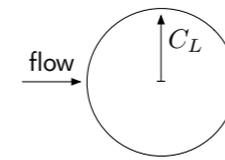
$$PE(t) = \frac{1}{2} \iint u_T^2(x, y, t) dx dy$$

$$u_T = \sqrt{u^2 + v^2}$$

$$\Delta(t) = \frac{1}{2} \iint [\hat{u}_T(x, y, t) - u_T(x, y, t)]^2 dx dy$$



# performance in time



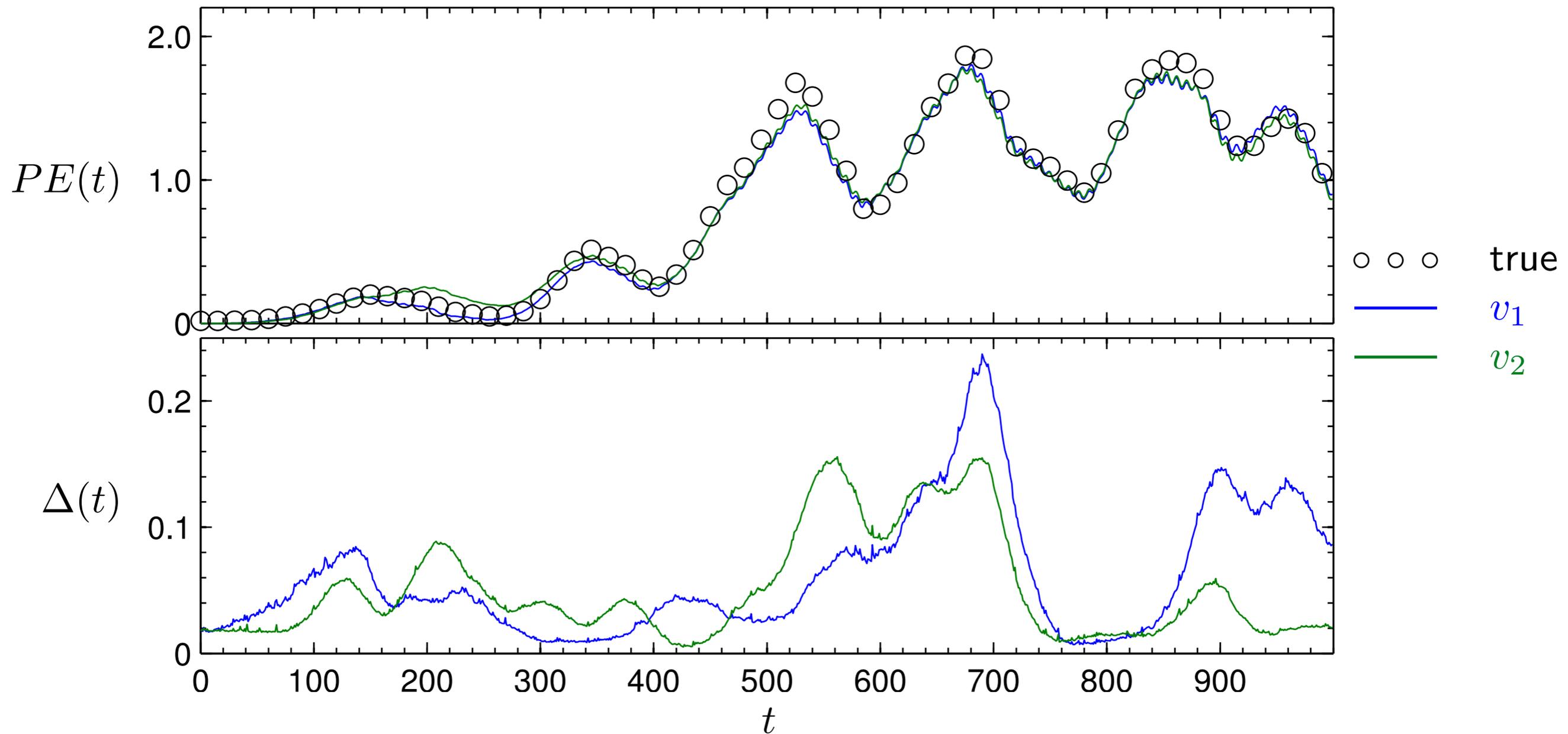
$v_1(t)$     $v_2(t)$

$v_3(t)$

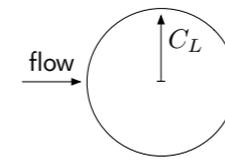
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# performance in time

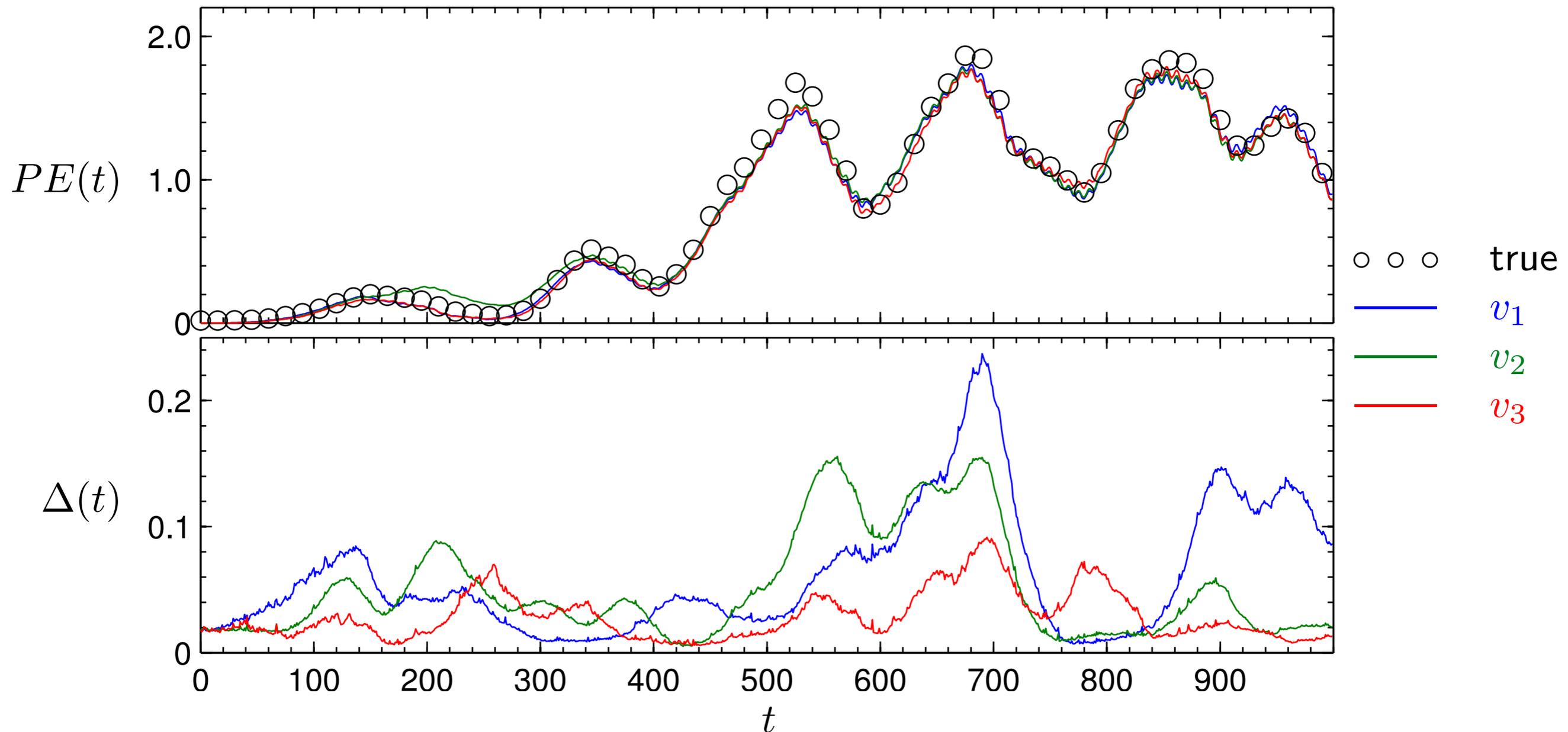


$v_1(t)$     $v_2(t)$     $v_3(t)$

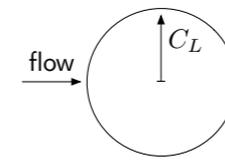
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# performance in time

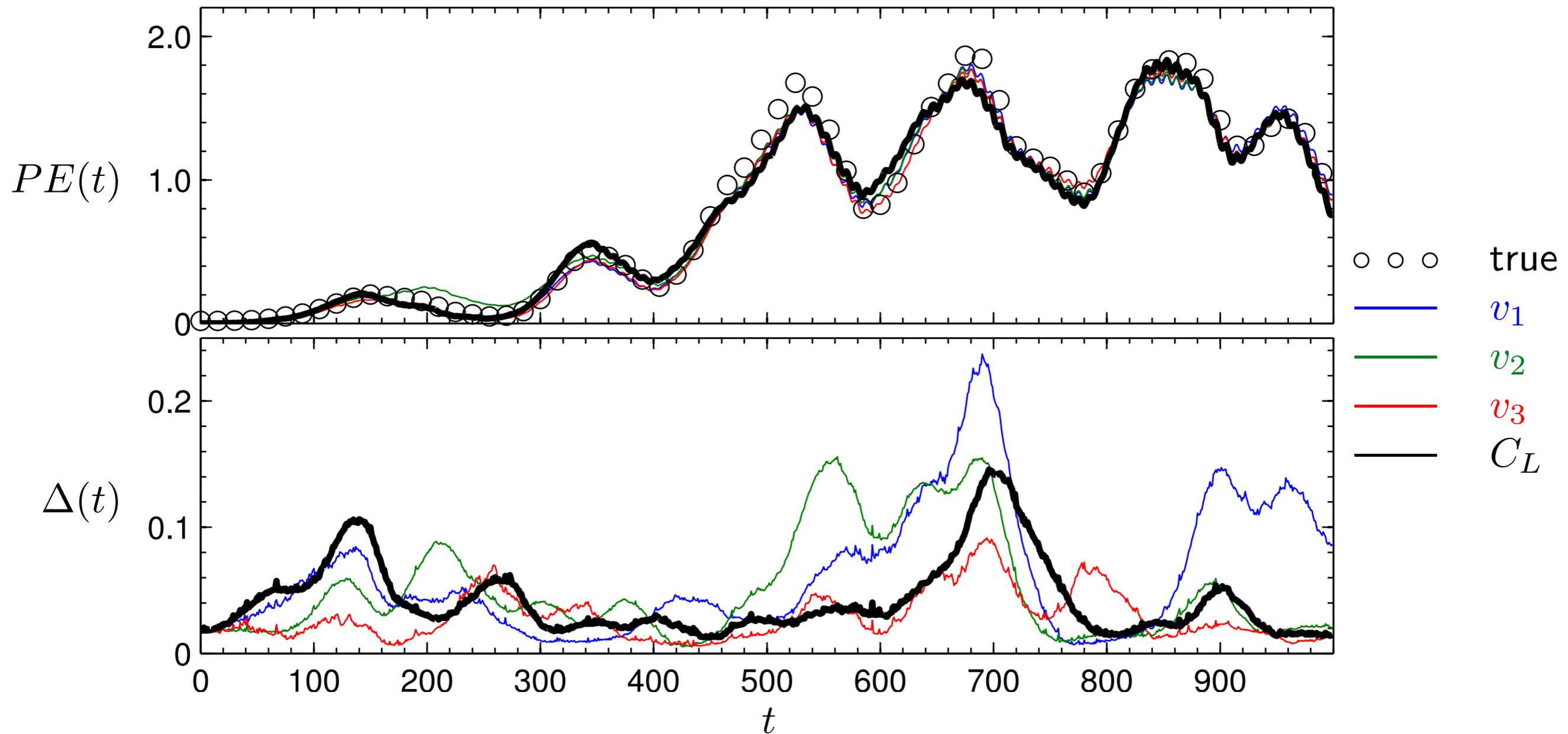


$v_1(t)$     $v_2(t)$     $v_3(t)$

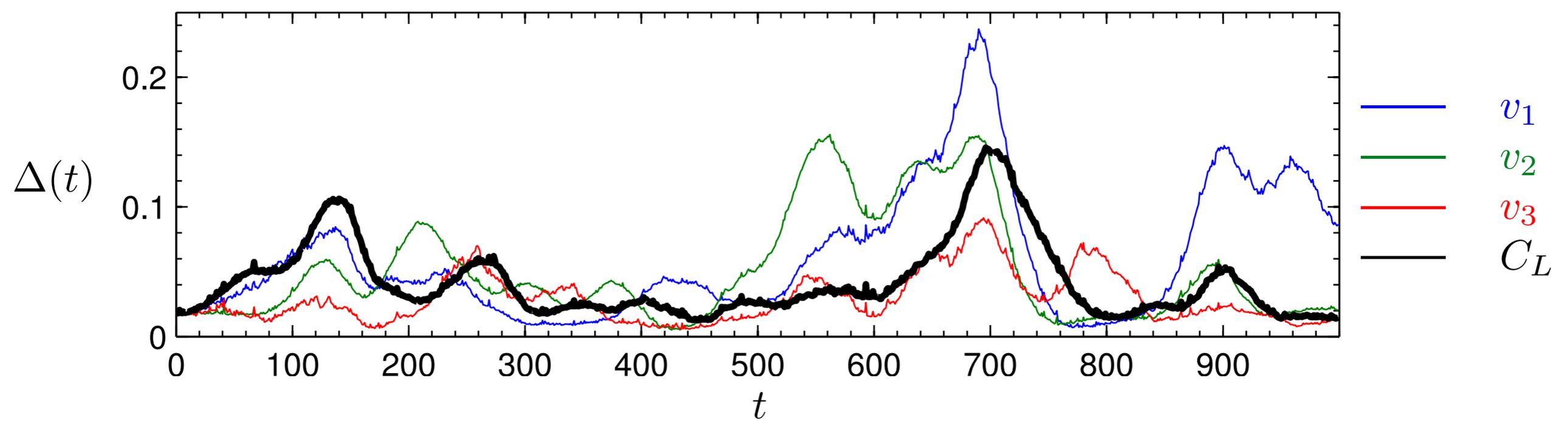
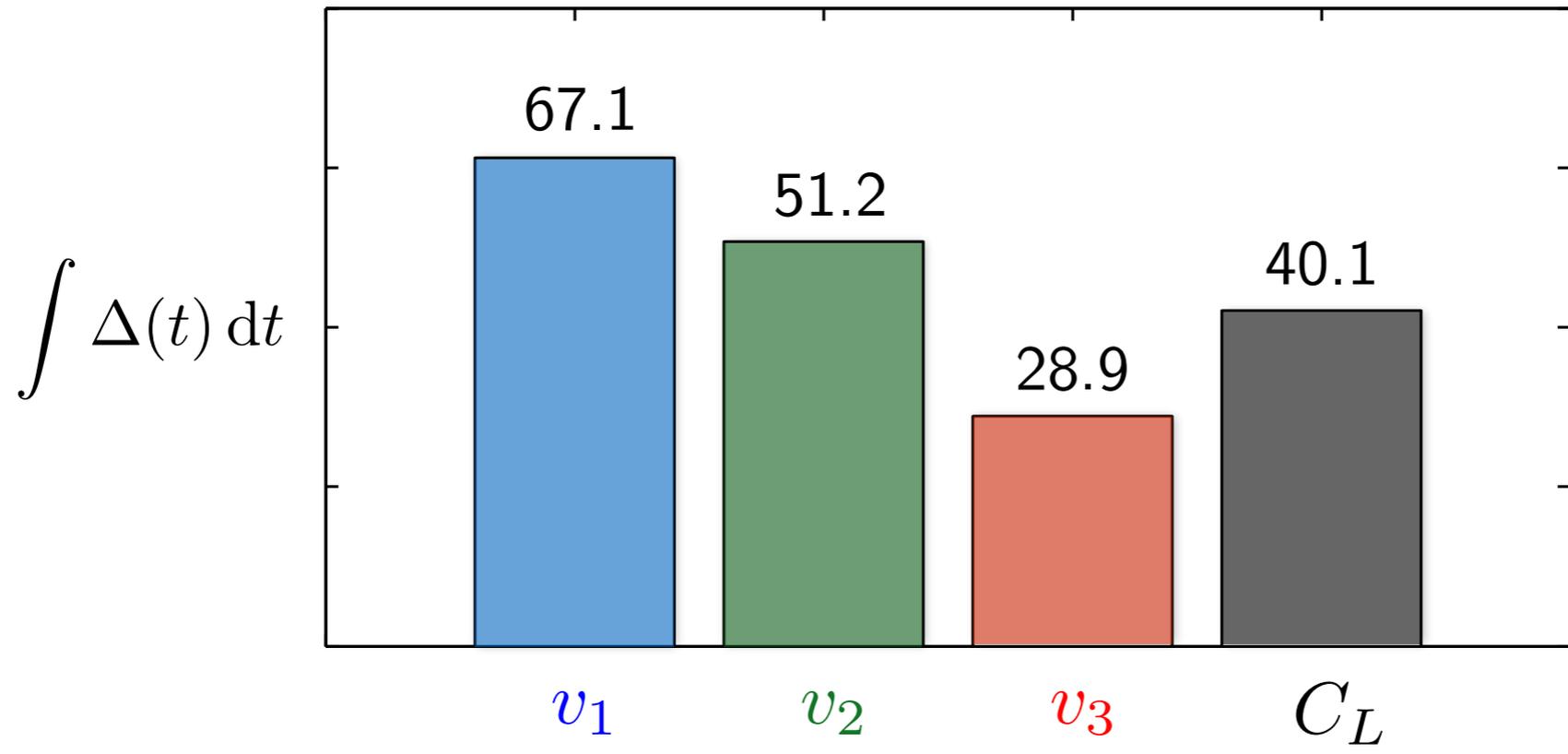
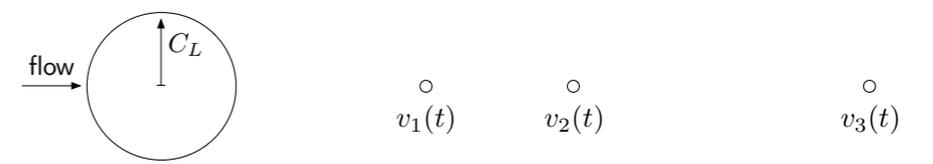
$$PE(t) = \frac{1}{2} \iint u_T^2(x, y, t) dx dy$$

$$u_T = \sqrt{u^2 + v^2}$$

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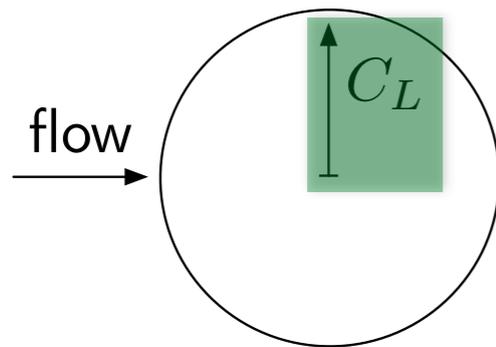
# performance in time



# results

- we estimate the entire flow using
  - i. transverse velocity at sensor two only

ii. lift force only

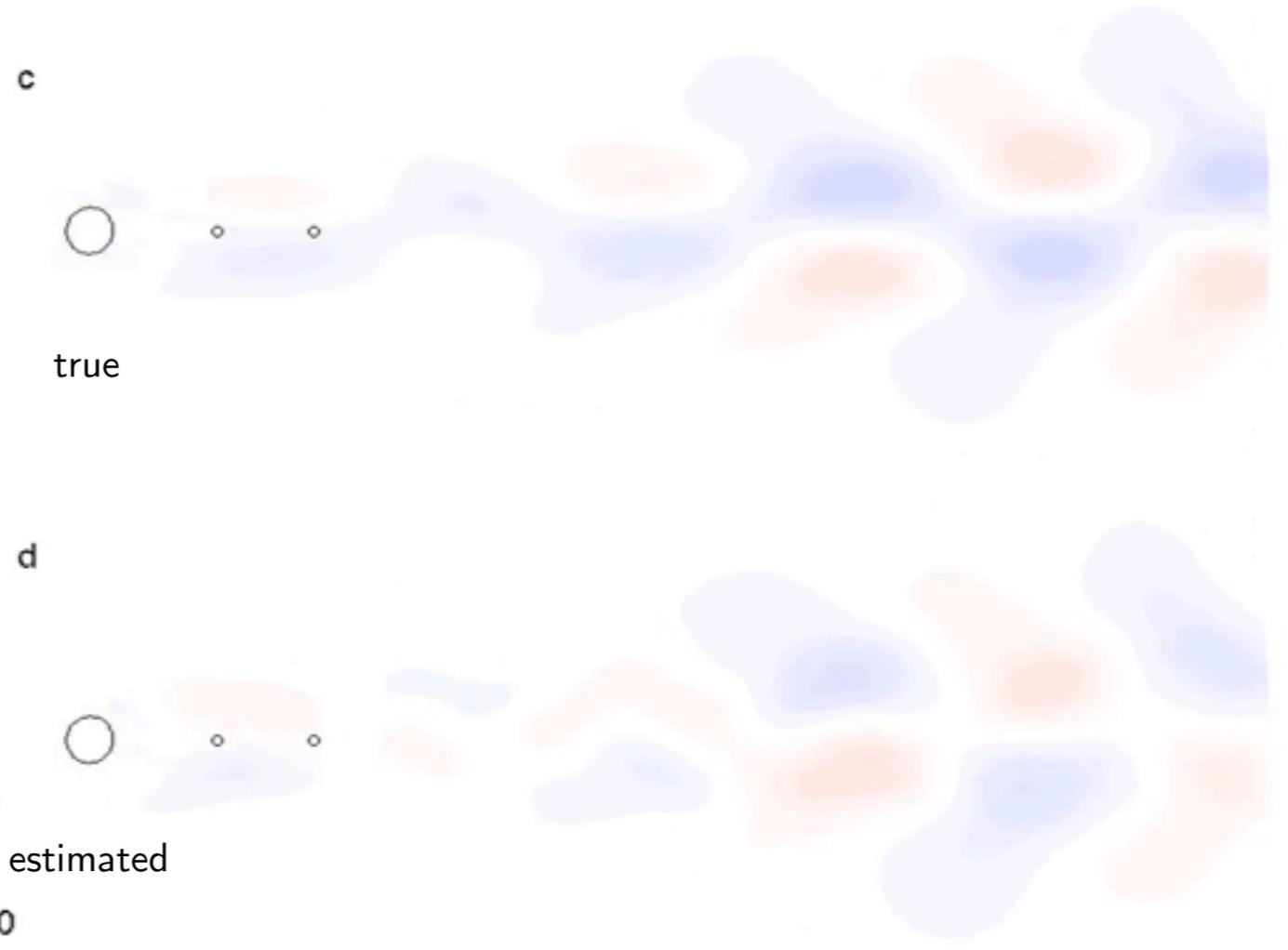
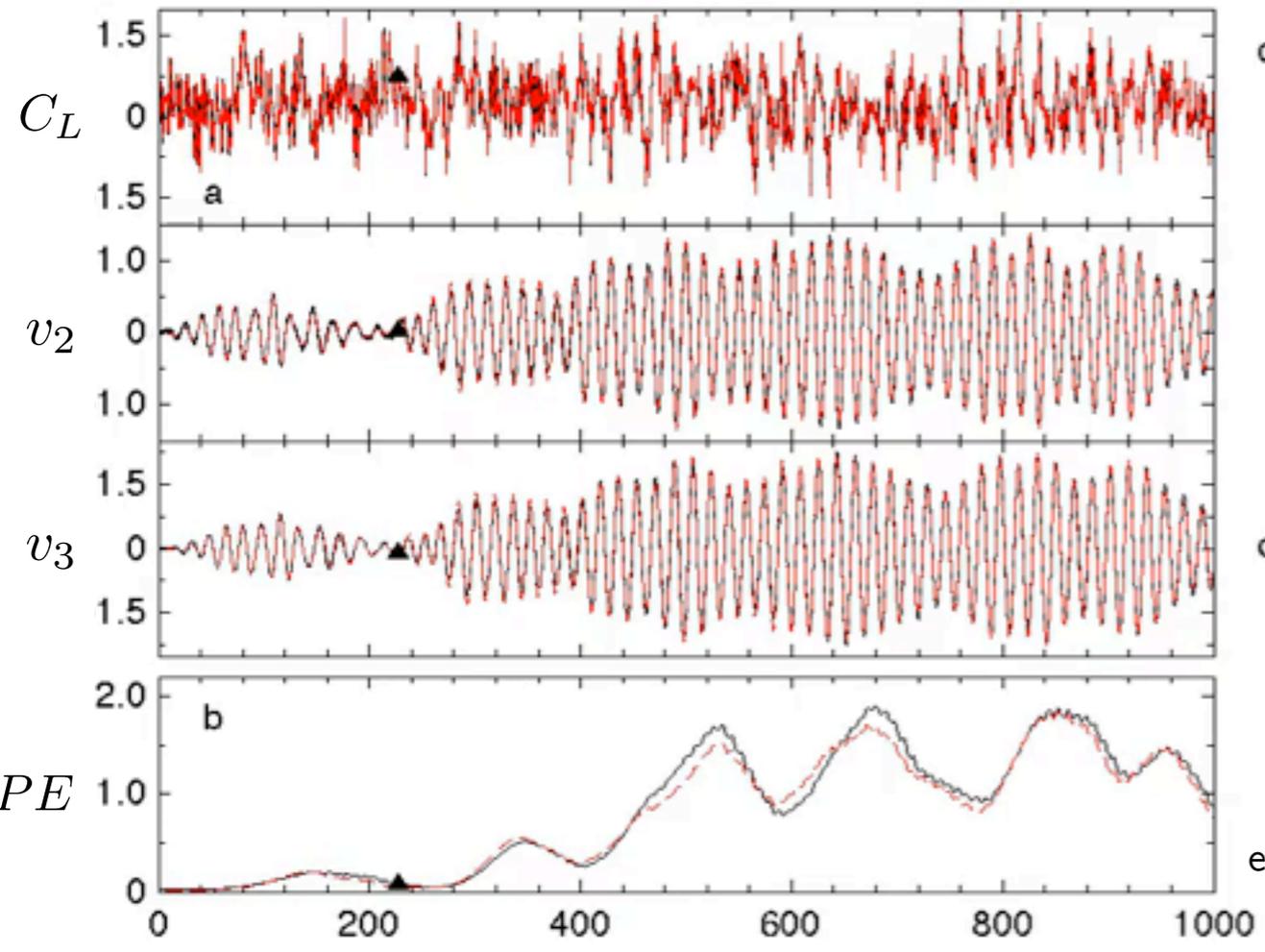
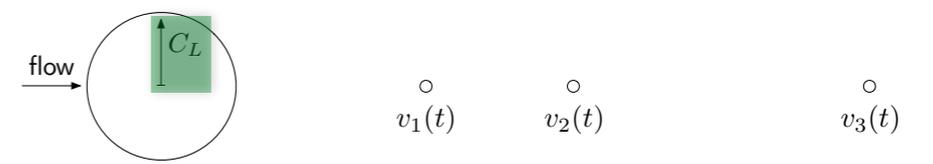


○  
 $v_1(t)$

○  
 $v_2(t)$

○  
 $v_3(t)$

# lift force only

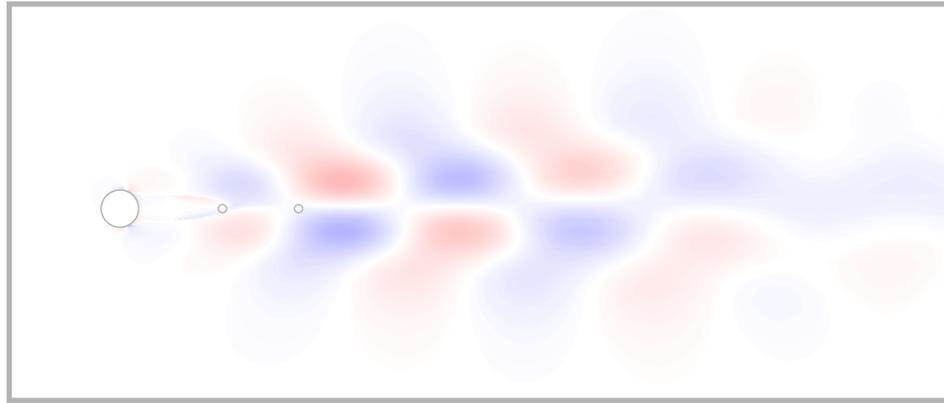


— true  
- - - estimated

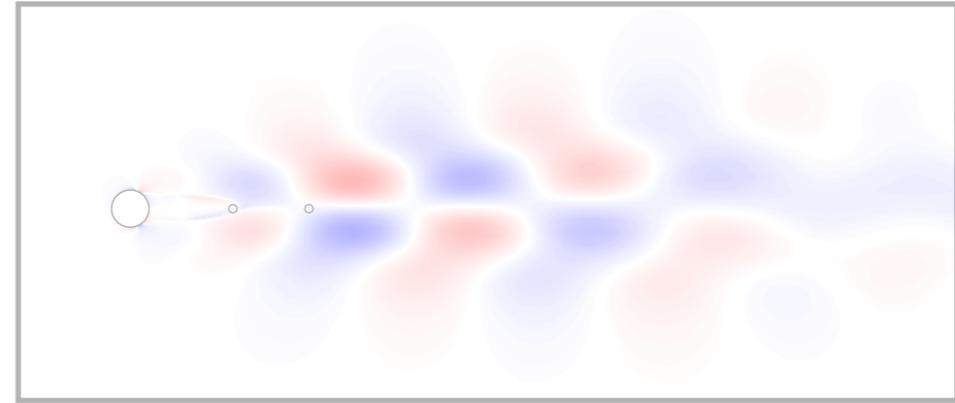
'local' or 'integral' sensing can work

# some applications

## 1. estimation

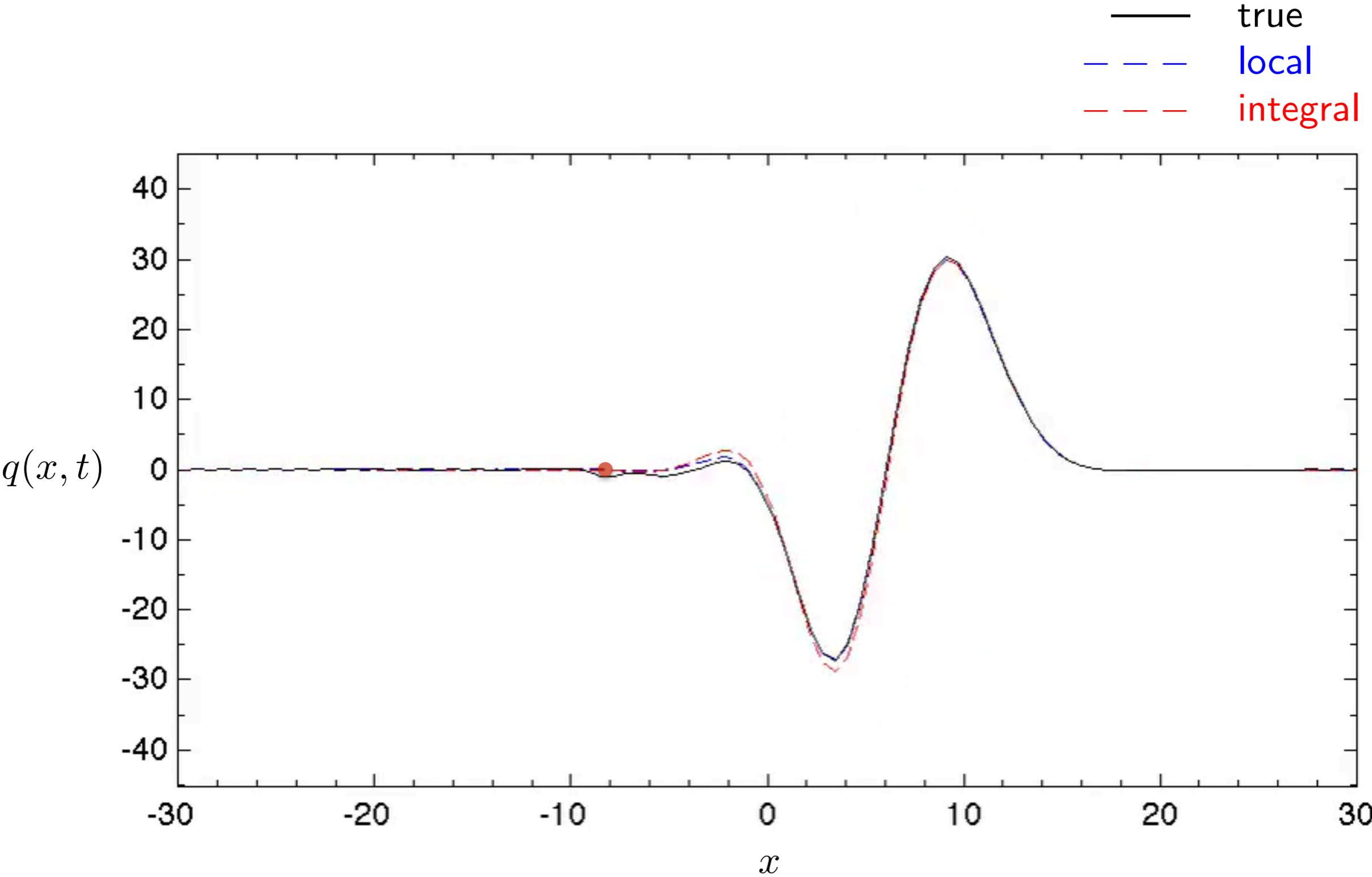


## 2. control



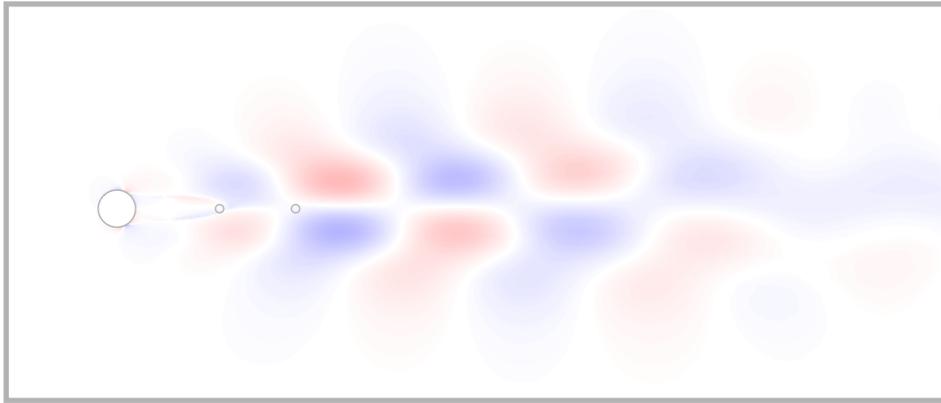
$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

# Ginzburg-Landau system



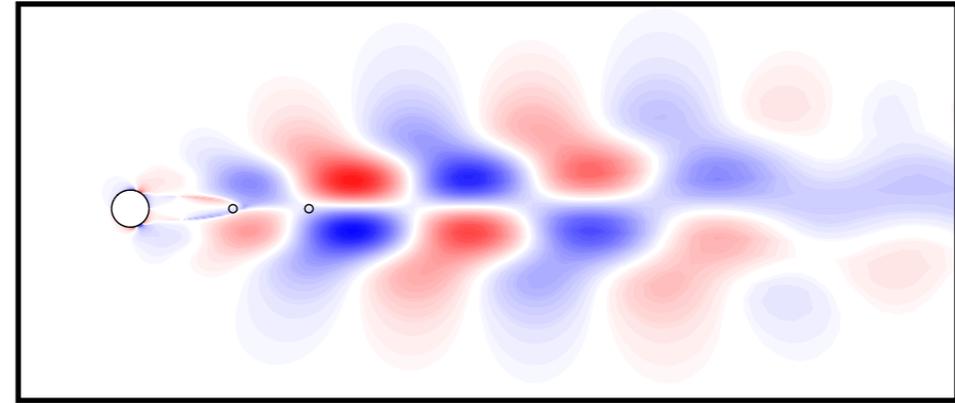
# some applications

## 1. estimation

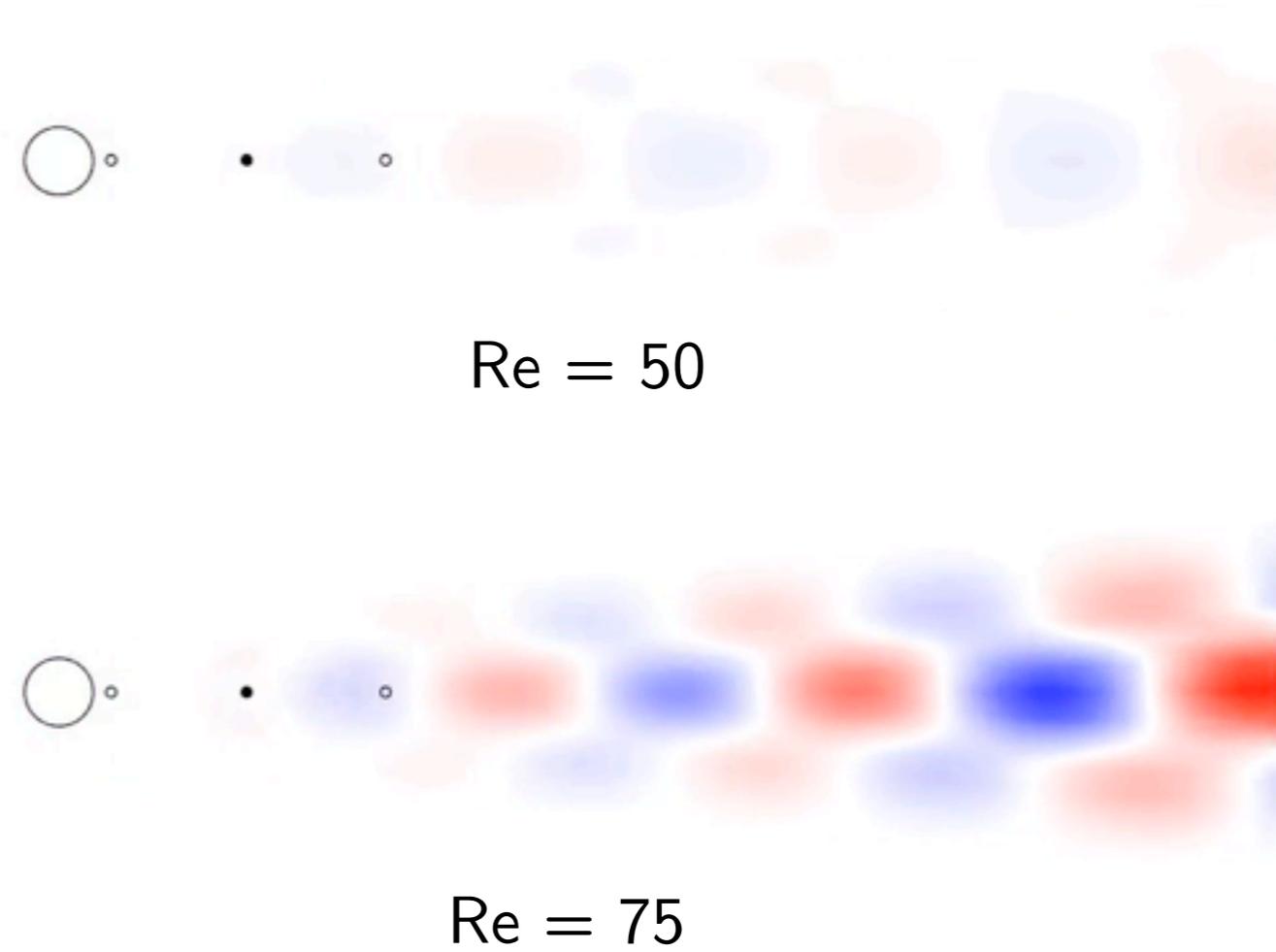
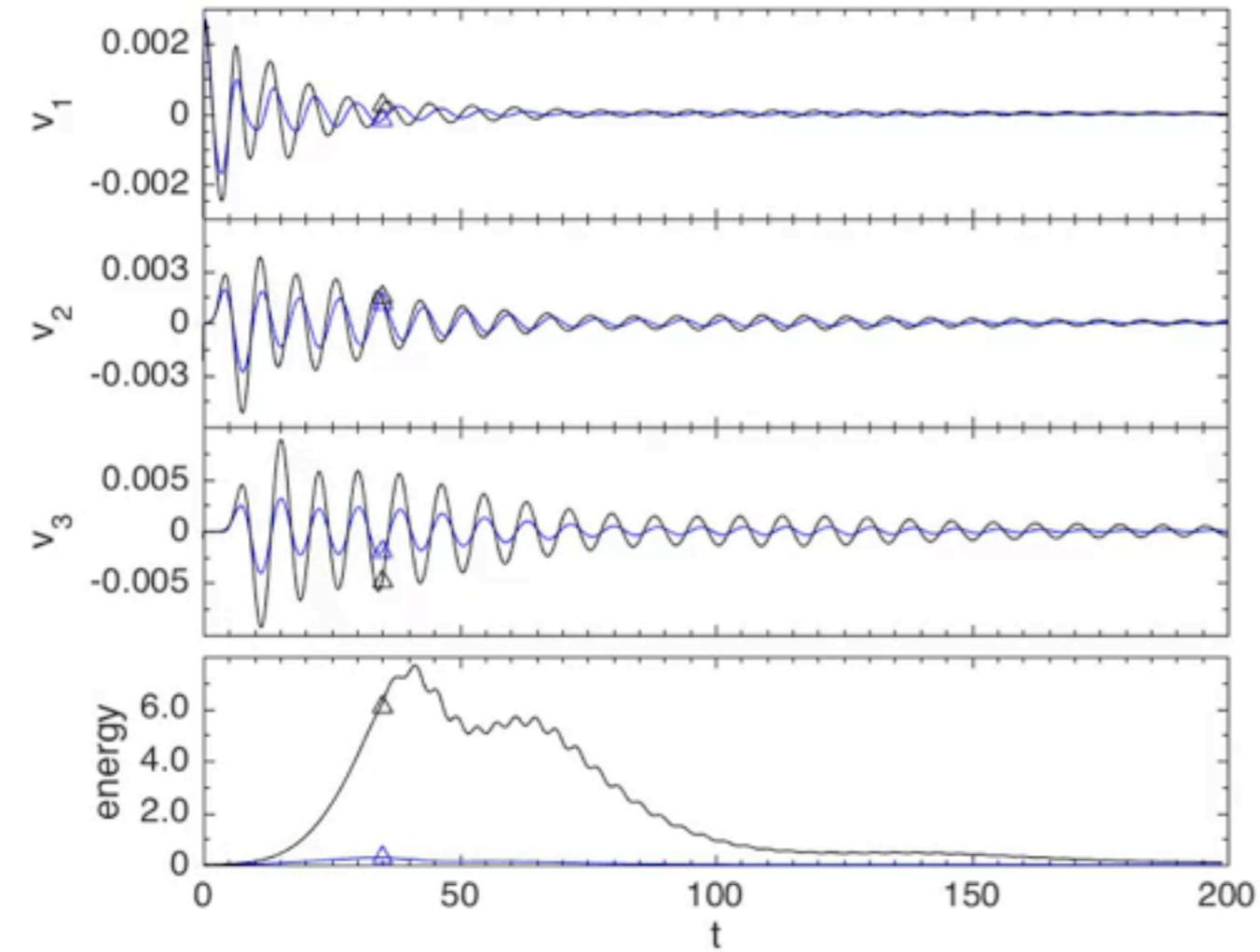


$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

## 2. control

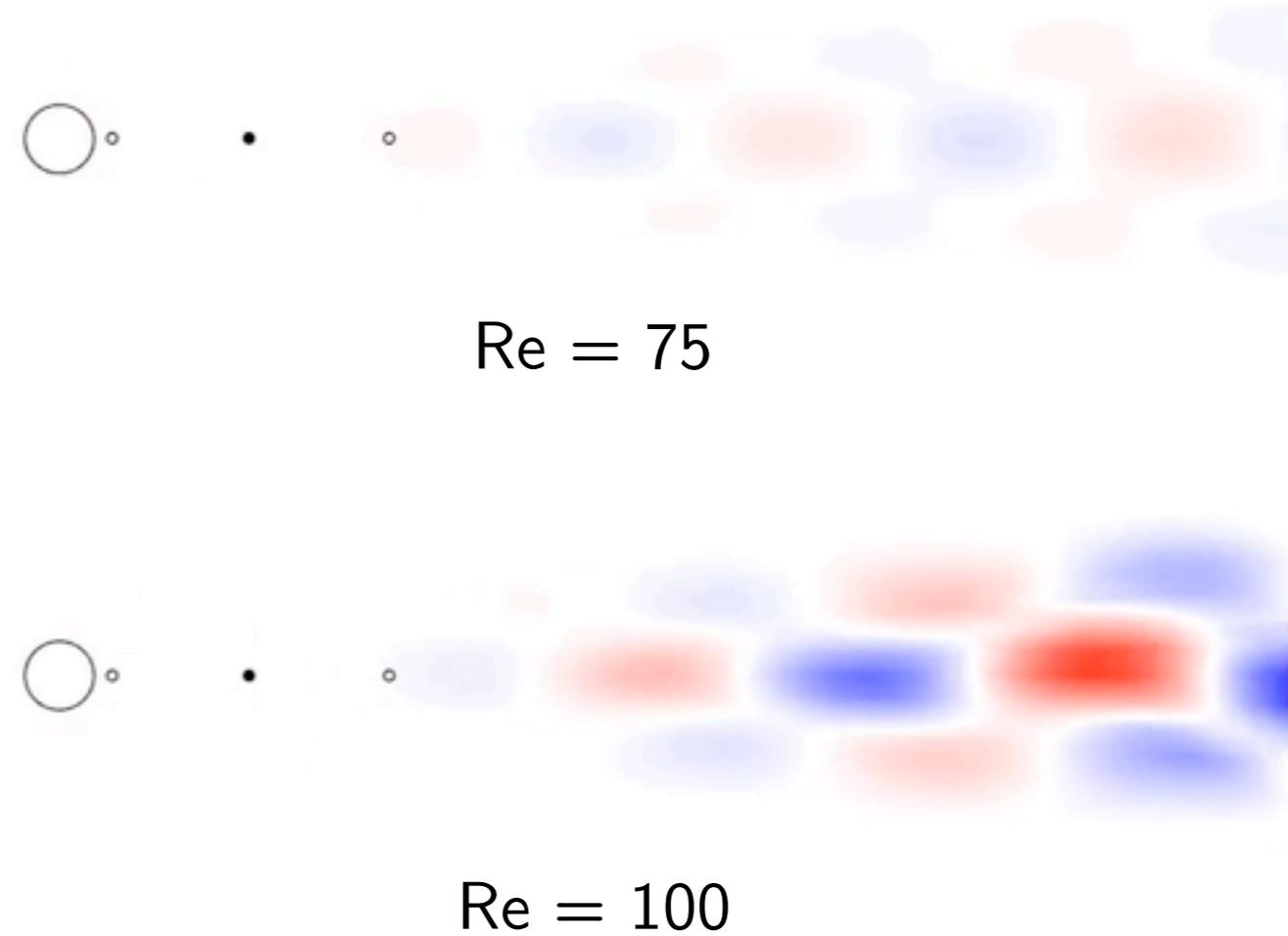
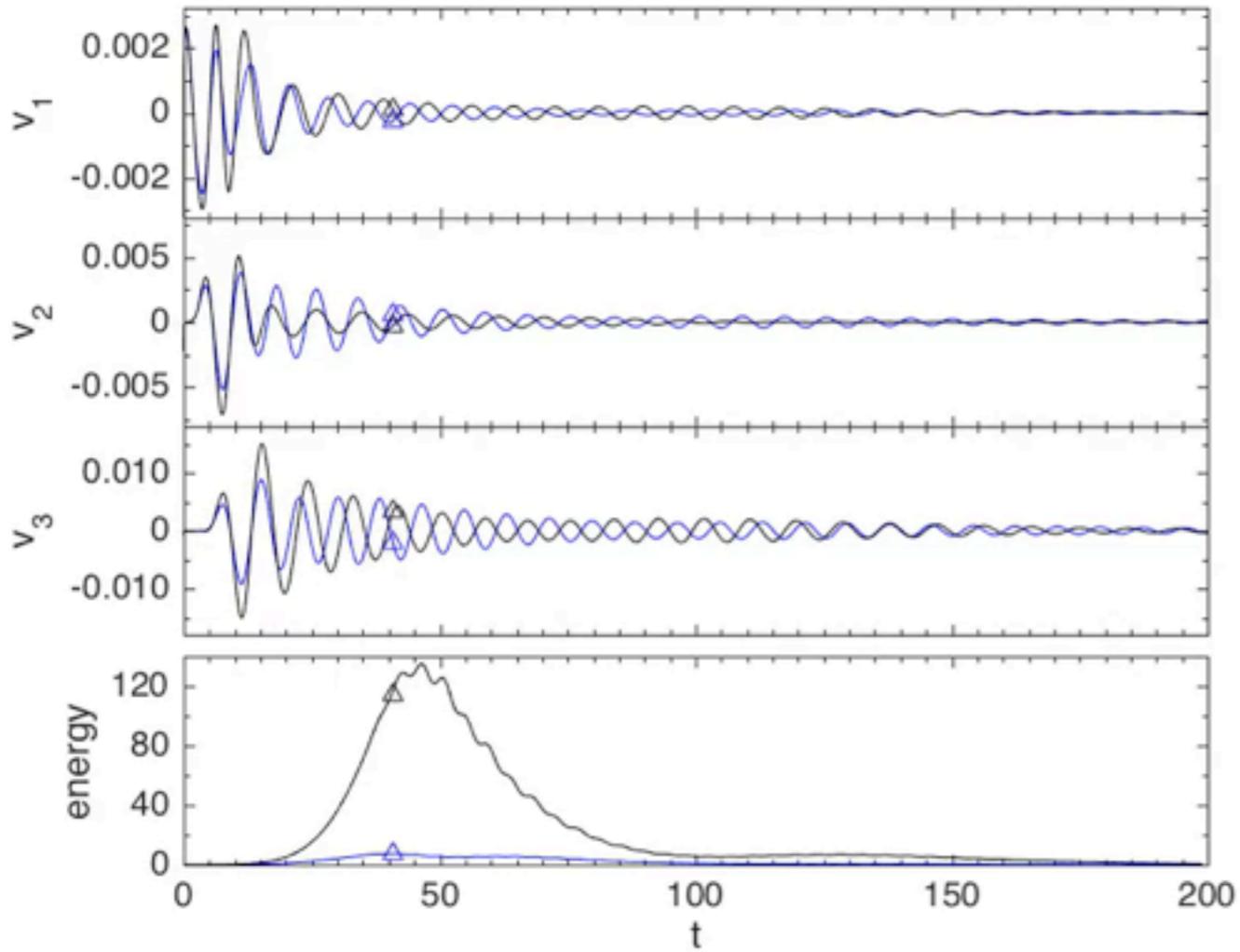


a single controller can be found that stabilizes for all  $50 \leq \text{Re} \leq 100$



robust controllers are (very) forgiving

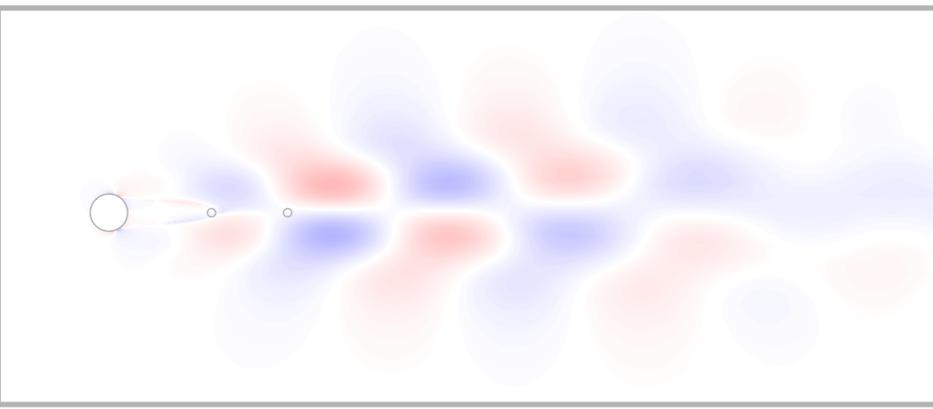
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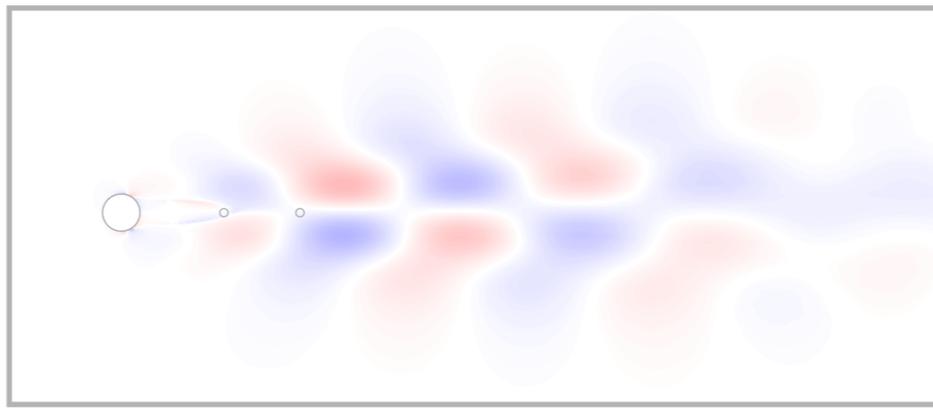
# some applications

## 1. estimation

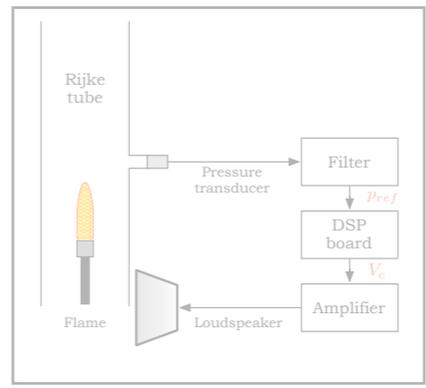
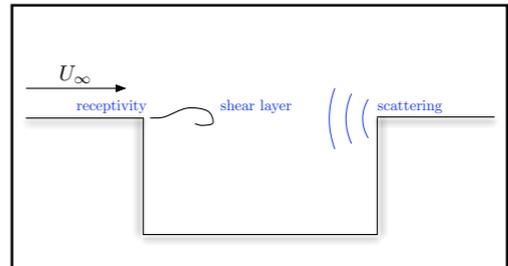


$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

## 2. control

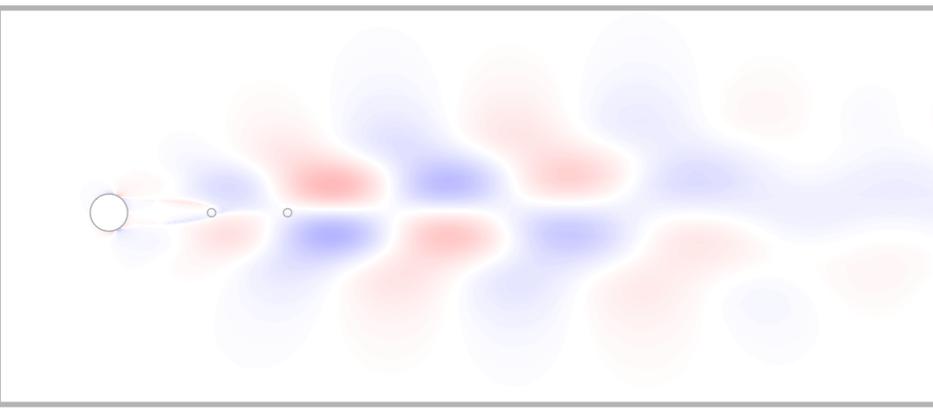


$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$



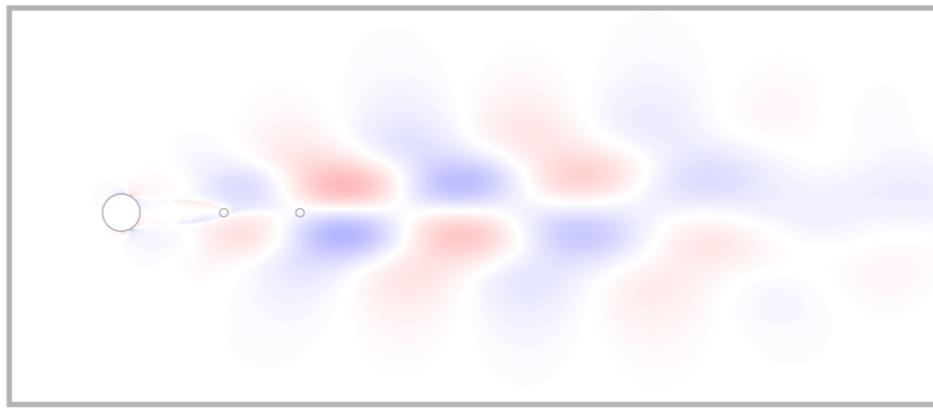
# some applications

## 1. estimation

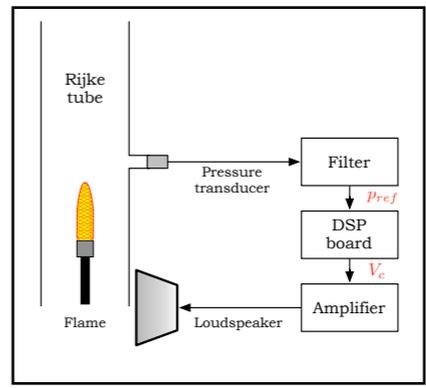
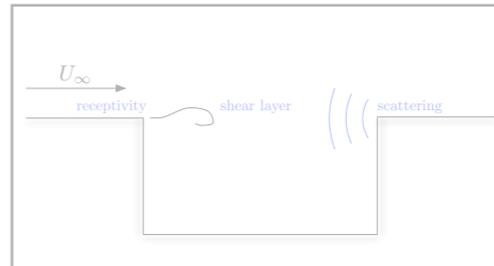


$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

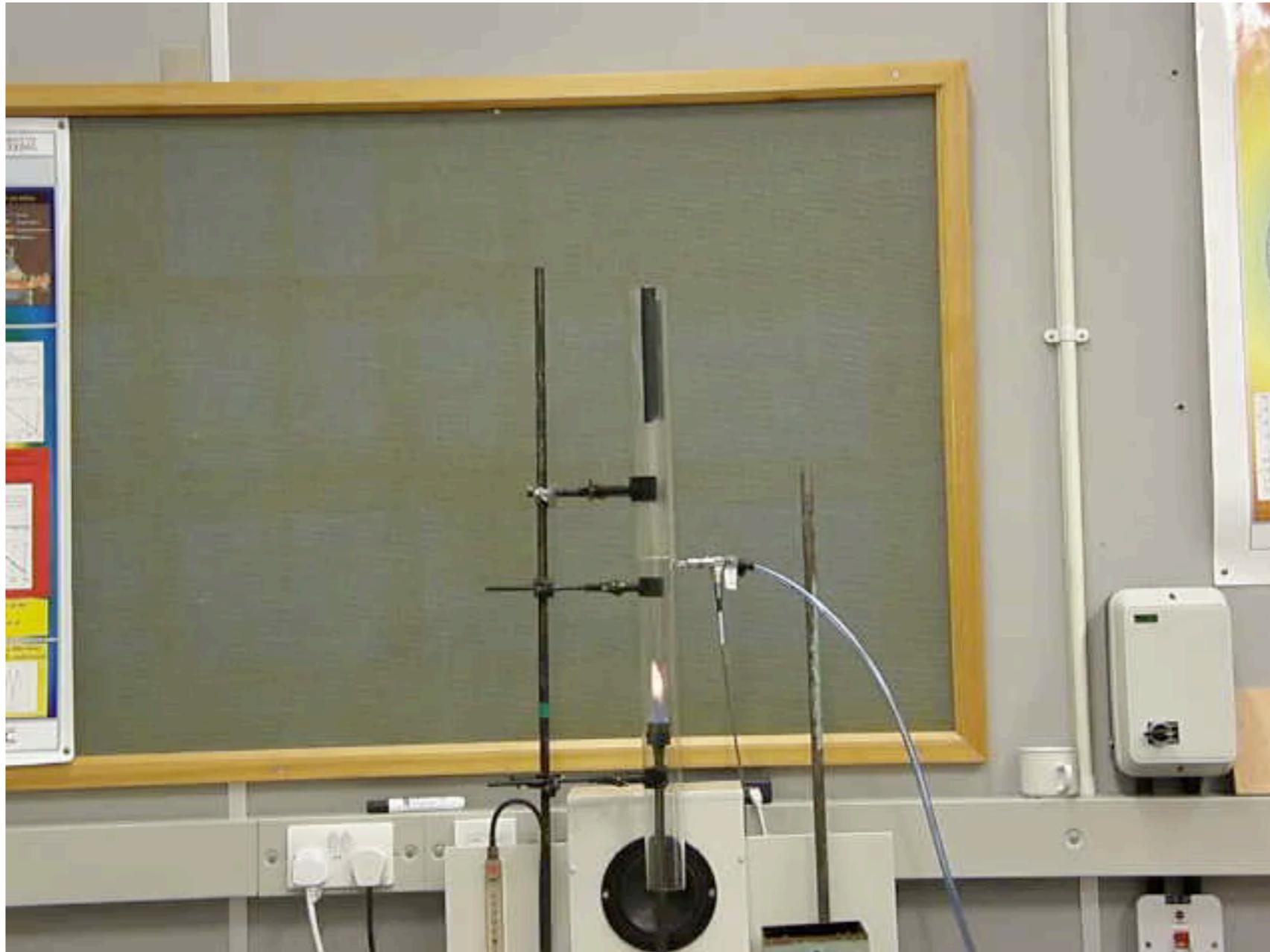
## 2. control



$$\frac{\partial q}{\partial t}(x, t) = \left( -\nu \frac{\partial}{\partial x} + \gamma \frac{\partial^2}{\partial x^2} + \mu(x) \right) q(x, t)$$

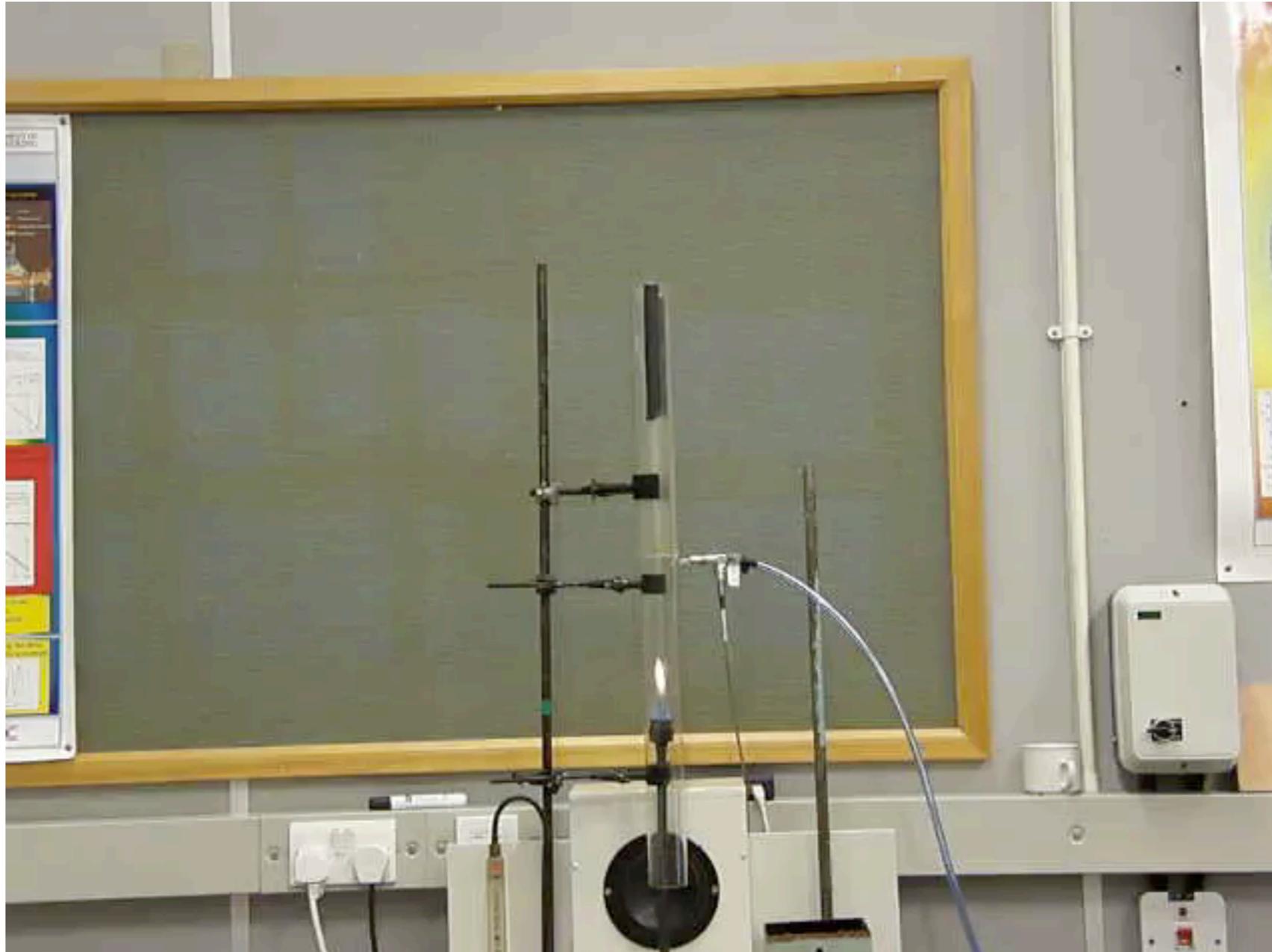


without control



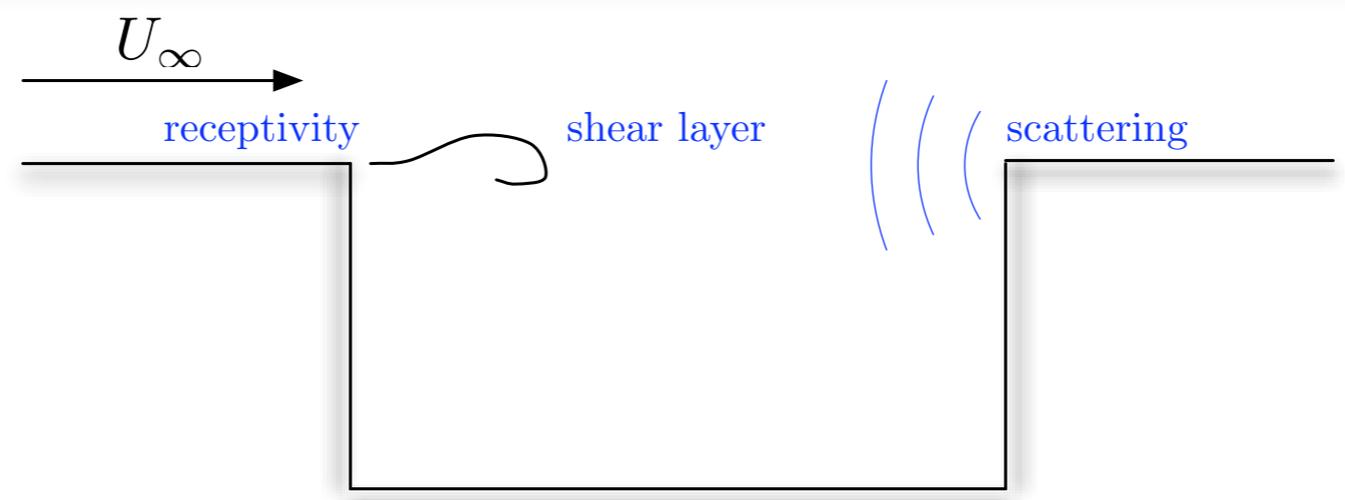
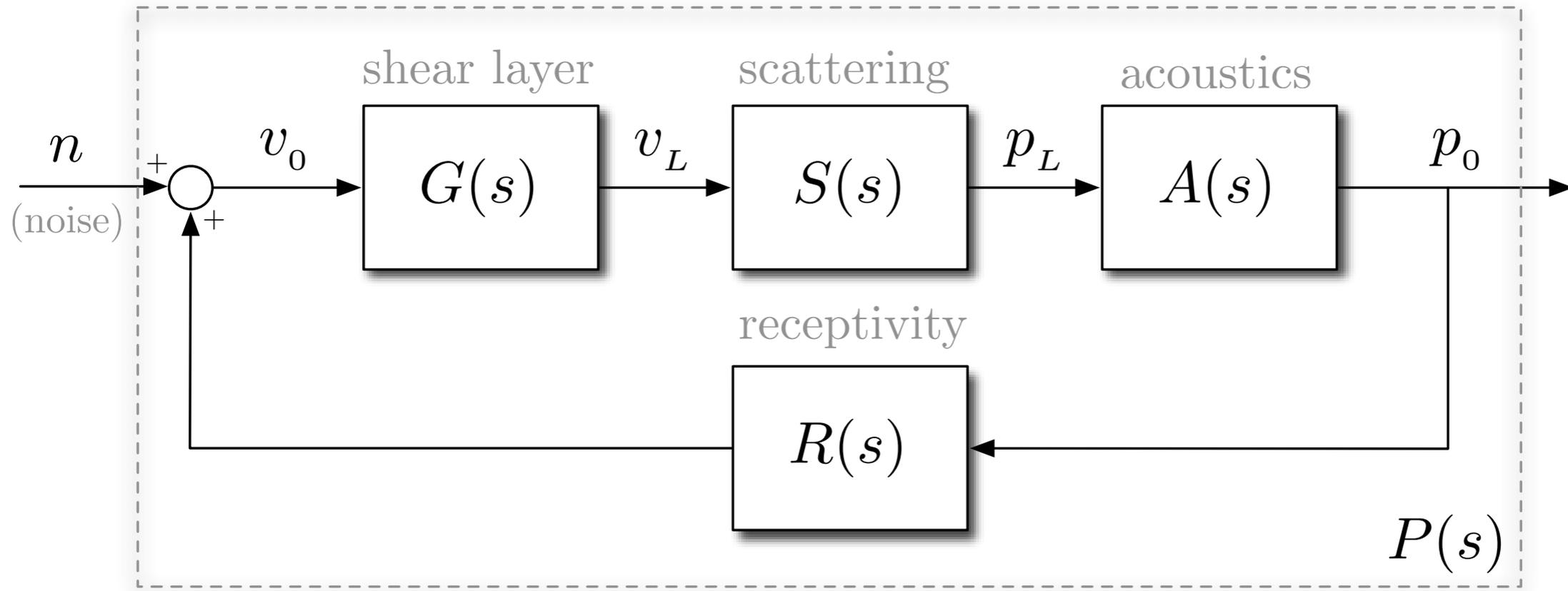
robust controllers are (very) forgiving

with control



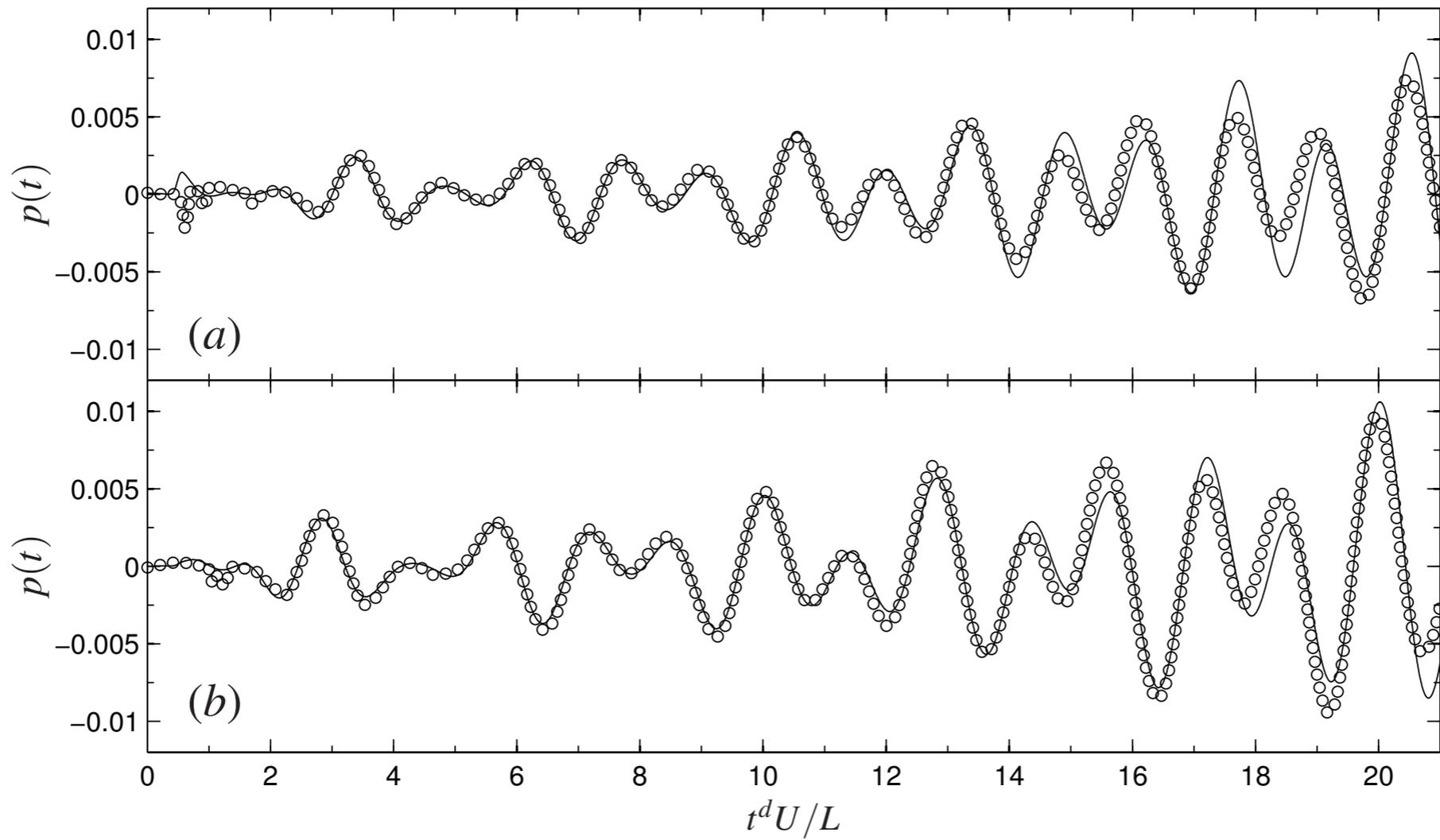
robust controllers are (very) forgiving

# compressible cavity oscillations



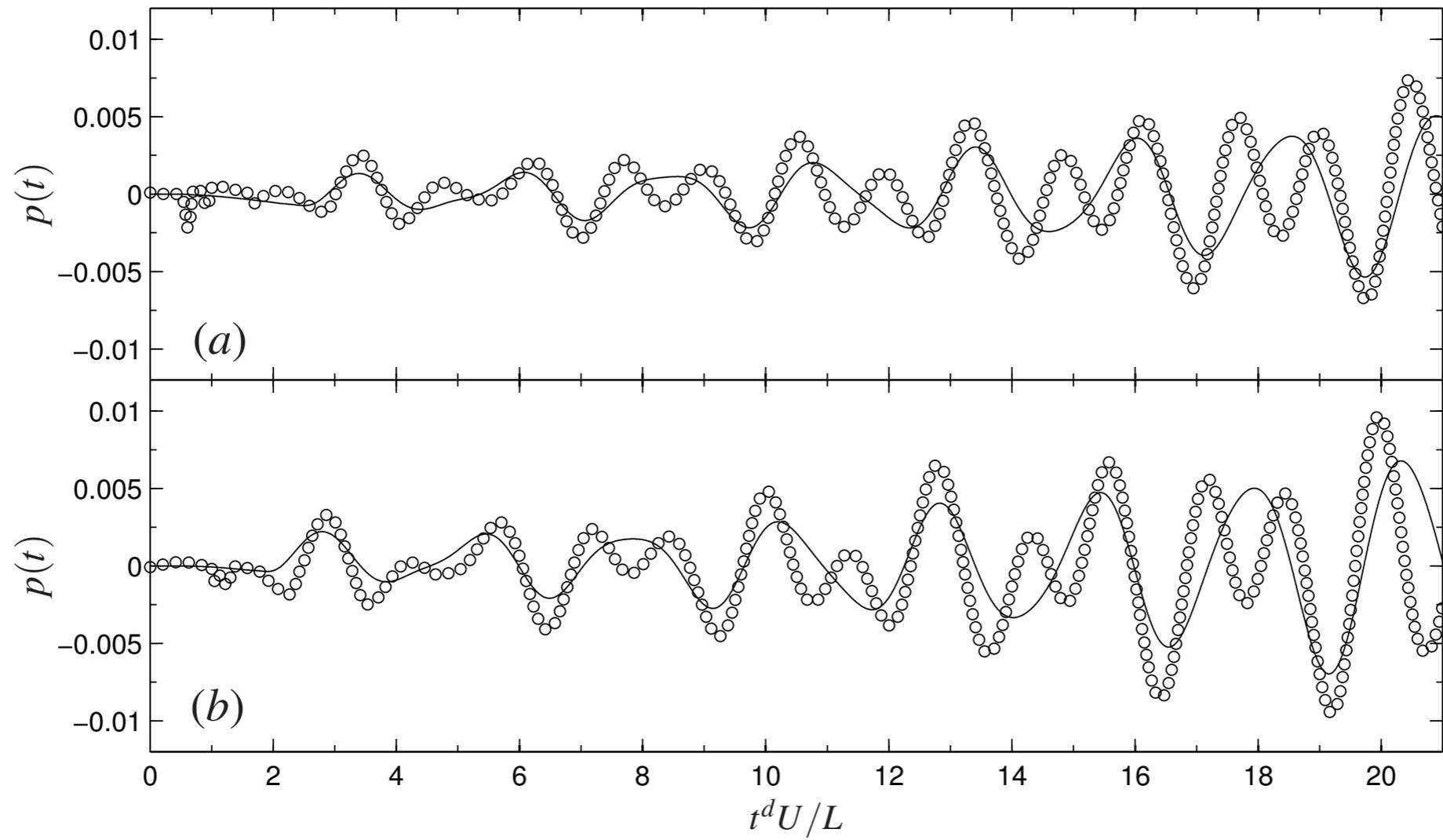
# linear model 1: measured directly

—impulse response

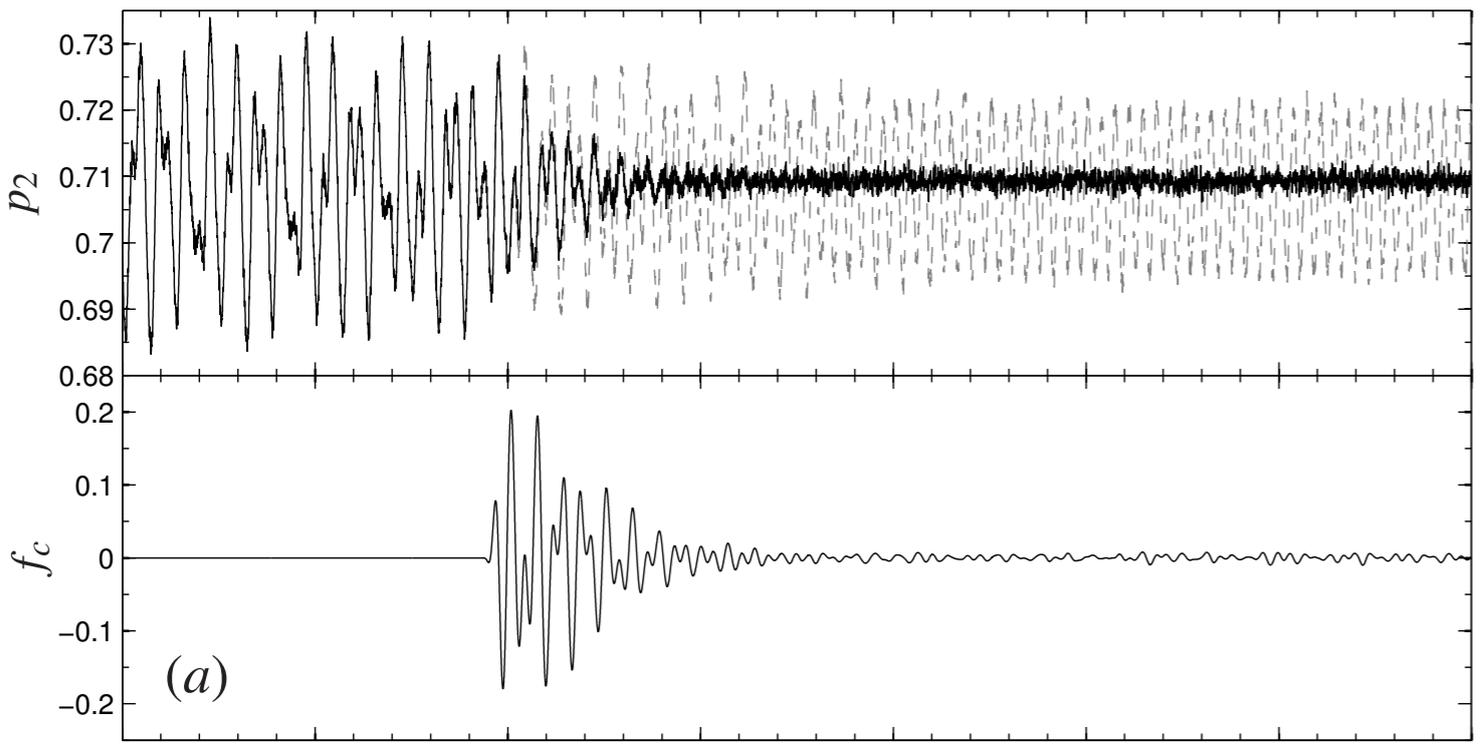


# linear model 2: simple constituent models

—impulse response

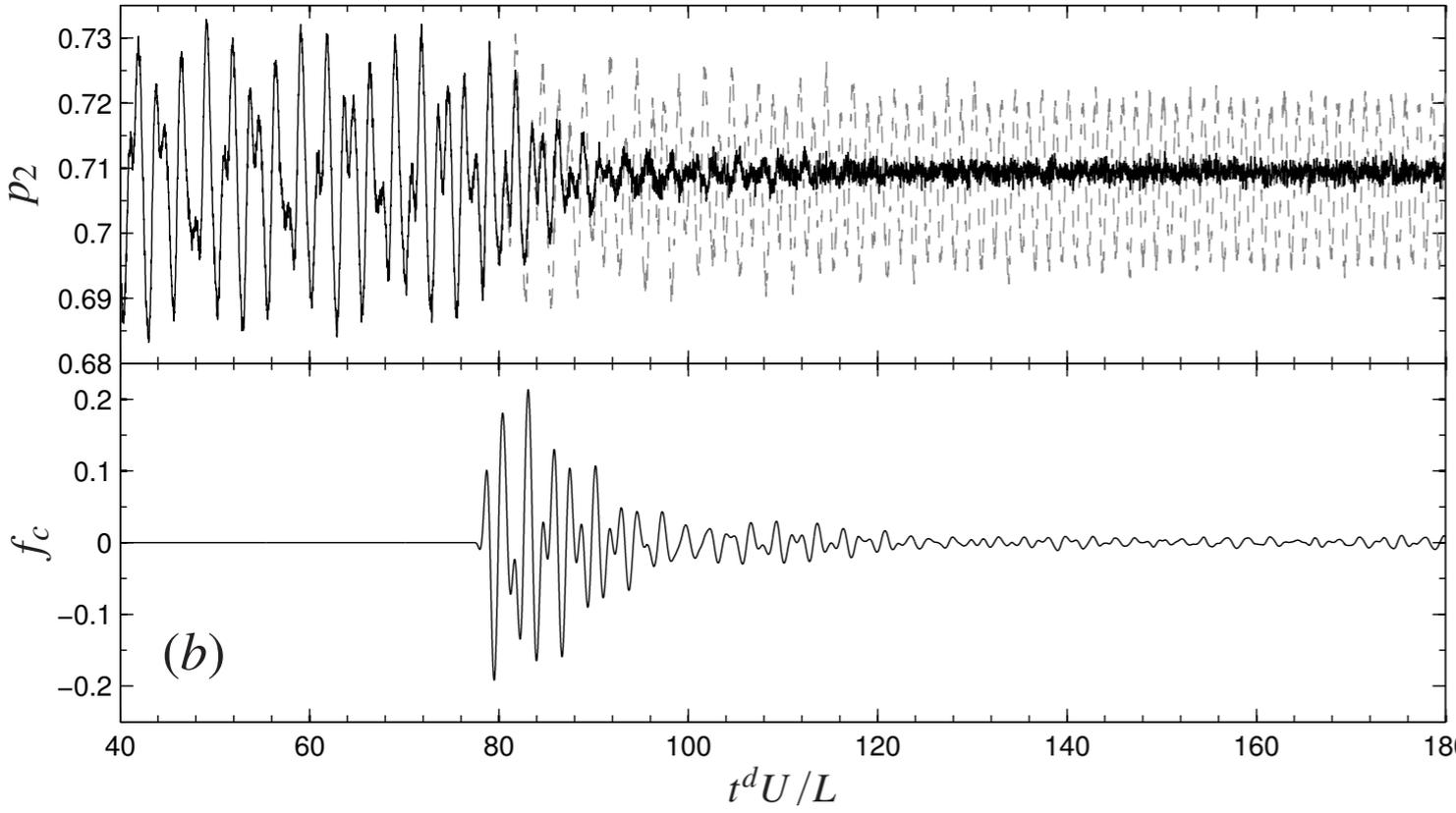


good enough for control  $\neq$  good enough for modelling

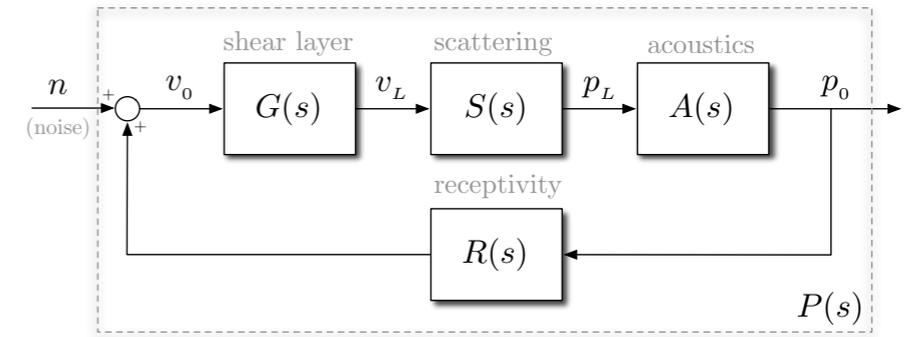


controller designed using linear model 1

$$\begin{array}{c}
 u(t) \rightarrow \boxed{\begin{array}{l} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{array}} \rightarrow y(t)
 \end{array}$$



controller designed using linear model 2



good enough for control  $\neq$  good enough for modelling